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**NASA TECHNICAL
MEMORANDUM**

NASA TM X- 73957-1

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(NASA-TM-X-73957-1) LaRC DESIGN ANALYSIS
REPORT FOR NATIONAL TRANSONIC FACILITY FOR
304 STAINLESS STEEL TUNNEL SHELL. VOLUME
1S: FINITE DIFFERENCE ANALYSIS OF
CONE/CYLINDER JUNCTION (NASA) 137 p HC

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LaRC DESIGN ANALYSIS REPORT
FOR
NATIONAL TRANSONIC FACILITY
FOR
304 STAINLESS STEEL TUNNEL SHELL
FINITE DIFFERENCE ANALYSIS OF CONE/CYLINDER JUNCTION
VOL. 1S

BY

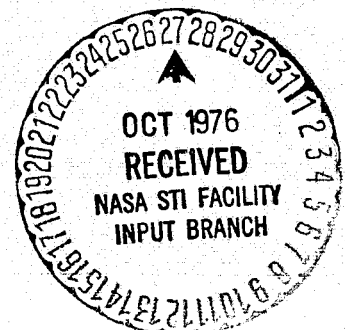
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AND JOHNNY W. ALLRED

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16. Abstract This report contains the results of extensive computer (finite element, finite difference and numerical integration), thermal, fatigue, and special analyses of critical portions of a large pressurized, cryogenic wind tunnel (National Transonic Facility). The computer models, loading and boundary conditions are described. Graphic capability was used to display model geometry, section properties, and stress results. A stress criteria is presented for evaluation of the results of the analyses. Thermal analyses were performed for major critical and typical areas. Fatigue analyses of the entire tunnel circuit is presented. The major computer codes utilized are: SPAR - developed by Engineering Information Systems, Inc. under NASA Contracts NAS8-30536 and NAS1-13977; SALORS - developed by Langley Research Center and described in NASA TN D-7179; and SRA - developed by Structures Research Associates under NASA Contract NAS1-10091; "A General Transient Heat-Transfer Computer Program for Thermally Thick Walls" developed by Langley Research Center and described in NASA TM X-2058.					
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NATIONAL TRANSONIC FACILITY

TUNNEL SHELL

NASA - LARC

FINITE DIFFERENCE ANALYSIS

OF

CONE/CYLINDER JUNCTION

304 STAINLESS STEEL

SEPTEMBER 1976

VOLUME 1S

LaRC CALCULATIONS
FOR THE
NATIONAL TRANSONIC FACILITY
TUNNEL SHELL

DATE: SEPTEMBER, 1976

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This report is one volume of a Design Analysis Report prepared by LaRC on portions of the pressure shell for the National Transonic Facility. This report is to be used in conjunction with reports prepared under NASA Contract NAS1-13535(c) by the Ralph M. Parsons Company (Job Number 5409-3 dated September 1976) and Fluidyne Engineering Corporation (Job Number 1060 dated September 1976). The volumes prepared by LaRC are listed below:

1. Finite Difference Analysis of Cone/Cylinder Junction (304 S.S.) Vol. 1, NASA TM X-73957-1.
2. Finite Element Analysis of Corners #3 and #4 (304 S.S.), Vol. 2S, NASA TM X-73957-2.
3. Finite Element Analysis of Plenum Region Including Side Access Reinforcement, Side Access Door and Angle of Attack Penetration (304 S.S.), Vol. 3S, NASA TM X73957-3.
4. Thermal Analysis (304 S.S.) Vol. 4S, NASA TM X73957-4.
5. Finite Element and Numerical Integration Analyses of the Bulkhead Region (304 S.S.), Vol. 5S, NASA TM X73957-5.
6. Fatigue Analysis (304 S.S.), Vol. 6S, NASA TM X73957-6.
7. Special Studies (304 S.S.), Vol. 7S, NASA TM X73957-7.

NTF DESIGN CRITERIA FOR 304 STAINLESS STEEL

GENERAL

THE DESIGN OF THE PRESSURE SHELL REFLECTED IN THIS REPORT SATISFIES THE DESIGN REQUIREMENTS OF THE ASME BOILER AND PRESSURE VESSEL CODE, SECTION VIII, DIVISION 1. SINCE DIVISION 1 DOES NOT CONTAIN RULES TO COVER ALL DETAILS OF DESIGN, ADDITIONAL ANALYSES WERE PERFORMED IN AREAS HAVING COMPLEX CONFIGURATIONS SUCH AS THE CONE CYLINDER JUNCTIONS, THE GATE VALVE BULKHEADS, THE BULKHEAD-SHELL ATTACHMENTS, THE PLENUM ACCESS DOORS AND REINFORCEMENT AREAS, THE ELLIPTICAL CORNER SECTIONS, AND THE FIXED REGION (RING S8) OF THE TUNNEL. THE DIVISION 1 DESIGN CALCULATIONS, THE ADDITIONAL ANALYSES AND THE CRITERIA FOR EVALUATION OF THE RESULTS OF THE ADDITIONAL ANALYSES TO ENSURE COMPLIANCE WITH THE INTENT OF DIVISION 1 REQUIREMENTS ARE CONTAINED IN THE TEXT OF THIS REPORT. THE DESIGN ANALYSES AND ASSOCIATED CRITERIA CONSIDERED BOTH THE OPERATING AND HYDROSTATIC TEST CONDITIONS.

IN CONJUNCTION WITH THE DESIGN, A DETAILED FATIGUE ANALYSIS OF THE PRESSURE SHELL WAS ALSO PERFORMED UTILIZING THE METHODS OF THE ASME CODE, SECTION VIII, DIVISION 2.

MATERIAL

THE PRESSURE SHELL MATERIAL SHALL BE ASME, SA-240, GRADE 304 FOR PLATE AND SA-182, GRADE F304 FOR FORGINGS. THE MATERIAL PROPERTIES AT TEMPERATURES EQUAL TO OR BELOW 150°F ARE AS FOLLOWS:

(A) PLATE

YIELD = 30.0 KSI
ULTIMATE = 75.0 KSI

(B) WELDS (AUTOMATIC, SEMIAUTOMATIC, OR "STICK")

YIELD = 30.0 KSI
ULTIMATE = 75.0 KSI

OPERATING, DESIGN AND TEST CONDITIONS

THE OPERATING, DESIGN AND TEST CONDITIONS FOR THE TUNNEL PRESSURE SHELL AND ASSOCIATED SYSTEMS AND ELEMENTS ARE SUMMARIZED BELOW:

1. OPERATING MEDIUM

ANY MIXTURE OF AIR AND NITROGEN

2. DESIGN TEMPERATURE RANGE

MINUS 320 DEGREES FAHRENHEIT TO PLUS 150 DEGREES FAHRENHEIT, EXCEPT IN THE REGION OF THE PLENUM BULKHEADS AND GATE VALVES INSIDE A 23-FOOT, 4-INCH DIAMETER, FOR WHICH THE TEMPERATURE RANGE IS MINUS 320 DEGREES FAHRENHEIT TO PLUS 200 DEGREES FAHRENHEIT.

3. PRESSURE RANGE

TUNNEL CONFIGURATION	OPERATING PRESSURE RANGE, PSIA	DESIGN PRESSURES PSID
A. CONDITION I - PLENUM ISOLATION GATES OPEN AND TUNNEL OPERATING:		
TUNNEL CIRCUIT EXCEPT PLENUM	8.3 to 130	A. 8 EXTERNAL B. 119 INTERNAL
PLENUM (PLENUM PRESS- URE IS LIMITED TO .4 TO 1 TIMES THE REMAINDER OF THE TUNNEL CIRCUIT	3.3 to 130	A. 15 EXTERNAL B. 119 INTERNAL
BULKHEAD		56 (EXTERNAL TO PLENUM)
B. CONDITION II - PLENUM ISOLATION GATES OPEN AND TUNNEL SHUTDOWN:		
ENTIRE TUNNEL CIRCUIT	8.3 to 130	A. 8 EXTERNAL B. 119 INTERNAL
BULKHEAD		0
C. CONDITION III - PLENUM ISOLATION GATES AND ACCESS DOORS CLOSED:		
TUNNEL CIRCUIT EXCEPT PLENUM	8.3 to 130	A. 8 EXTERNAL B. 119 INTERNAL

PLENUM (PLENUM OPER-
ATING PRESSURE CAN
EXCEED THE PRESSURE
IN THE REMAINDER OF
THE TUNNEL CIRCUIT BY
24 PSI, BUT DOES NOT
EXCEED THE 130 PSIA
MAXIMUM OPERATING
PRESSURE)

0 to 130

- A. 15 EXTERNAL
- B. 119 INTERNAL

BULKHEAD

- A. 25 (INTERNAL TO
PLENUM)
- B. 119 (EXTERNAL TO
PLENUM) FOR MINUS
320 DEGREES
FAHRENHEIT TO
PLUS 150 DEGREES
FAHRENHEIT

- *C. 115.7 (EXTERNAL TO
PLENUM) FOR PLUS
151 DEGREES
FAHRENHEIT TO PLUS
200 DEGREES
FAHRENHEIT

*OPERATING PROCEDURES LIMIT PRESSURES TO THAT SHOWN.

D. CONDITION IV - PLENUM
ISOLATION GATES CLOSED
AND ACCESS DOORS OPEN:

TUNNEL CIRCUIT EXCEPT
PLENUM

8.3 to 130

- A. 8 EXTERNAL
- B. 119 INTERNAL

PLENUM

14.7

0

BULKHEAD

- A. 119 (EXTERNAL TO
PLENUM) FOR MINUS
320 DEGREES FAHRENHEIT
TO PLUS 150 DEGREES
FAHRENHEIT
- *B. 115.7 (EXTERNAL TO
PLENUM) FOR PLUS 151
DEGREES FAHRENHEIT TO PLUS
200 DEGREES FAHRENHEIT

*OPERATING PROCEDURES LIMIT PRESSURES TO THAT SHOWN.

4. HYDROSTATIC TEST DESIGN CONDITIONS

THE PRESSURE SHELL WAS DESIGNED FOR HYDROSTATIC TEST IN ACCORDANCE WITH THE REQUIREMENTS OF THE ASME CODE, SECTION VIII, DIVISION 1. THE TEST PRESSURES SHALL BE AS FOLLOWS. PRESSURE SHELL TEMPERATURE SHALL BE EQUAL TO OR BELOW 100°F DURING HYDROSTATIC TESTS.

CONDITION (1) - MAXIMUM INTERNAL PRESSURE CONDITION FOR THE ENTIRE TUNNEL CIRCUIT

$$\begin{aligned} PH_1 &= 1.5 (119) \left(\frac{18.7}{18.2} \right) + \text{HYDROSTATIC HEAD} \\ &= 183.4 \text{ PSI} + \text{HYDROSTATIC HEAD} \end{aligned}$$

CONDITION (2) - MAXIMUM DIFFERENTIAL PRESSURE CONDITION ACROSS THE PLENUM BULKHEADS

$$\begin{aligned} PH_2 &= 1.5 \left(\frac{18.7}{18.2} \right) (119) + \text{HYDROSTATIC HEAD} \\ &= 183.4 + \text{HYDROSTATIC HEAD} \end{aligned}$$

$$\begin{aligned} PH_2^* &= 1.5 (115.7) \left(\frac{18.7}{17.7} \right) + \text{HYDROSTATIC HEAD} \\ &= 183.4 + \text{HYDROSTATIC HEAD} \end{aligned}$$

*TUNNEL OPERATION LIMITATIONS PRECLUDE PRESSURE DIFFERENTIALS ACROSS BULKHEADS IN EXCESS OF 115.7 PSI FOR BULKHEAD AND GATE TEMPERATURES IN EXCESS OF 150°F.

CONDITION (3) - MAXIMUM REVERSE DIFFERENTIAL PRESSURE CONDITION ACROSS THE PLENUM BULKHEADS

$$PH_3 = 1.5 \left(\frac{18.7}{18.2} \right) (25) = 38.5 \text{ PSI}$$

THE PRESSURE SHELL EXCEPT FOR THE PLENUM SHALL BE PRESSURIZED TO 144.9 PSIG. THE PLENUM SHALL BE PRESSURIZED TO 183.4 PSIG.

PRESSURE SHELL STRESS EVALUATION CRITERIA

THIS CRITERIA ESTABLISHES THE BASIS FOR ANALYSIS AND DESIGN OF THE PRESSURE SHELL SO IT WILL MEET OR EXCEED ALL OF THE REQUIREMENTS OF SECTION VIII, DIVISION 1 OF THE ASME BOILER AND PRESSURE VESSEL CODE AND CAN BE STAMPED WITH A DIVISION 1 "U" STAMP.

1. SECTION VIII, DIVISION 1, DIRECT APPLICATION

(A) THE MAXIMUM ALLOWABLE STRESS (S)

$$S = 18.2 \text{ KSI } (-320^{\circ}\text{F TO } +150^{\circ}\text{F})$$

$$S = 17.7 \text{ KSI } (-320^{\circ}\text{F TO } +200^{\circ}\text{F})$$

(B) PRIMARY BENDING PLUS PRIMARY MEMBRANE STRESSES

THE LOCAL MEMBRANE STRESSES ARE NOT GENERALLY CONSIDERED IN SECTION VIII, DIVISION 1 DESIGNS. HOWEVER, FOR THE PURPOSE OF DESIGNING LOCAL REINFORCEMENT AT BRACKETS, RINGS OR PENETRATIONS NOT COVERED BY DESIGN BASED ON STRESS ANALYSIS, THE LOCAL SHELL MEMBRANE STRESS SHALL BE:

$$P_b + P_m \leq 1.5 SE$$

NOTE: E IS JOINT EFFICIENCY

2. IN REGIONS OF THE PRESSURE SHELL WHERE DIVISION 1 DOES NOT CONTAIN RULES TO COVER ALL DETAILS OF DESIGN (REF. U-2(g)), ADDITIONAL ANALYSES WERE PERFORMED UTILIZING THE GUIDELINES OF THE ASME CODE, SECTION VIII, DIVISION 2, APPENDIX 4, "DESIGN BASED ON STRESS ANALYSIS." THE BASIC STRESS CRITERIA FOR DIVISION 2 IS REPRESENTED IN FIGURE 4-130.1 AND RESTATED BELOW INDICATING ANY MODIFICATIONS OR EXCESS REQUIREMENTS APPLIED TO IT TO REMAIN WITHIN THE INTENT OF DIVISION 1 AND TO OBTAIN A DIVISION 1 STAMP.

A. GENERAL PRINCIPAL MEMBRANE STRESS

MAXIMUM ALLOWABLE STRESS

$$S = 18.2 \text{ KSI } (-320^{\circ}\text{F TO } +150^{\circ}\text{F})$$

$$S = 17.7 \text{ KSI } (-320^{\circ}\text{F TO } +200^{\circ}\text{F})$$

MAXIMUM ALLOWABLE STRESS INTENSITY

$$S_m = 20.0 \text{ KSI } (-320^{\circ}\text{F TO } +300^{\circ}\text{F})$$

B. PRIMARY GENERAL MEMBRANE STRESS INTENSITY

$$P_m \leq S_m$$

AND IN ORDER TO COMPLY WITH DIVISION 1, THE MAXIMUM PRINCIPAL MEMBRANE STRESS MUST BE:

$$P_m^* \leq S$$

NOTE: THE * IS USED TO DENOTE THAT MAXIMUM PRINCIPAL STRESSES ARE TO BE COMPUTED FOR THE GIVEN LOADING CONDITION. THE INTENT IS TO DETERMINE THE STRESSES WHICH REPRESENT THE HOOP STRESSES AND MERIDIONAL STRESSES WHICH ARE THE STRESSES USED IN DIVISION 1 COMPUTATIONS.

C. DESIGN LOADS, PRIMARY LOCAL MEMBRANE STRESS INTENSITY

$$P_L \leq 1.5 S_m$$

NOTE: LOCAL MEMBRANE STRESS INTENSITY IS DEFINED IN ACCORDANCE WITH DIVISION 2, APPENDIX 4-112(i). THE TOTAL MERIDIONAL LENGTH IS CONSIDERED TO BE $1.0 \sqrt{RT}$.

D. DESIGN LOADS, PRIMARY LOCAL MEMBRANE PLUS PRIMARY BENDING STRESS INTENSITY

$$P_L + P_b \leq 1.5 S_m$$

E. OPERATING LOADS, PRIMARY PLUS SECONDARY STRESS INTENSITY

$$P_L + P_b + Q \leq 3 S_m$$

3. A FATIGUE ANALYSIS WAS CONDUCTED IN ACCORDANCE WITH SECTION VIII, DIVISION 2 WITHOUT MODIFICATION.

4. HYDROSTATIC TEST CONDITION DESIGN CONSIDERATIONS

A. PRESSURE SHELL

IN ACCORDANCE WITH DIVISION 1 OF THE ASME CODE, DESIGN ANALYSIS OF THE PRESSURE SHELL FOR THE HYDROSTATIC TEST CONDITION IS NOT REQUIRED. HOWEVER, IN ORDER TO PROVIDE A SATISFACTORY ENGINEERING DESIGN FOR THE PRESSURE SHELL SPECIAL EMPHASIS WAS GIVEN, AS PROMPTED BY NOTE (1) OF SECTION VIII, DIVISION 1 OF THE ASME CODE, TO FLANGES OF GASKETED JOINTS OR OTHER APPLICATIONS WHERE SLIGHT AMOUNTS OF DISTORTION CAN CAUSE LEAKAGE OR MALFUNCTION. EXAMPLES OF THESE AREAS ARE THE PLENUM, PLENUM ACCESS DOORS, PLENUM ACCESS DOOR REINFORCEMENT, THE BULKHEADS, AND BULKHEAD FLANGES.

B. SUPPORT RINGS

DESIGN OF THE PRESSURE SHELL SUPPORT RINGS, INCLUDING

THE CORNER RINGS, FOR THE HYDROSTATIC TEST CONDITION, COMPLIES WITH THE FOLLOWING:

- (A) THE COMBINED VALUE OF THE SHELL CIRCUMFERENTIAL PRESSURE STRESS, S_1 AND SHELL

BENDING STRESS S_2 , RESULTING FROM ACTION OF A

PORTION OF THE SHELL AS AN INNER FLANGE OF THE RING, SHALL NOT EXCEED 0.8 WELD YIELD STRESS:

$$S_1 + S_2 \leq 0.8 \text{ WELD YIELD STRESS,}$$

WHERE, FOR SUPPORT RINGS NOT ANALYZED BY FINITE ELEMENT TECHNIQUES,

$$S_1 = P_H \left(\frac{R}{T} \right) + .6 P_H; P_H \text{ INCLUDES HYDROSTATIC HEAD CORRECTION, AND}$$

S_2 = RING BENDING STRESS AT INNER FLANGE, BASED

ON AN EFFECTIVE WIDTH OF THE PRESSURE SHELL ACTING AS AN INNER FLANGE OF THE RING OF 1.1 MULTIPLIED BY THE SQUARE ROOT OF $D_0 T$.

- (B) THE BENDING STRESS, S_{2F} ON THE OUTSIDE FLANGE

SHALL NOT EXCEED .9 WELD YIELD STRESS. (IN THE COMPUTER ANALYSIS ALL LOADING CONDITIONS ARE LIMITED TO .9 S_Y ON THE OUTER FLANGE.)

- (C) BRACKETS AND SUPPORT PAD WELDMENTS

THE DESIGN FOR ALL LOADING CONDITIONS INCLUDING THE HYDROSTATIC TEST CONDITION OF THOSE PORTIONS OF BRACKETS AND SUPPORT PAD WELDMENTS WHICH ARE ATTACHED TO THE PRESSURE SHELL BUT NOT ON THE SURFACE OF THE SHELL SHALL COMPLY WITH THE REQUIREMENTS OF THE AISC CODE, I.E. MAXIMUM STRESS IN TENSION EQUALS .6 S_Y , ETC.

Vol 15

Finite Difference Analyses of Cone / Cylinder Junctions

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SUBJECT NTF
Cone / Cylinder Junctions

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304 S.S.

Part I

Reference Drawing No 944383 S & 944390 S

The cone / cylinder junctions were analyzed utilizing a shell of revolution computer code.

Computer Code

SALORS - Structural Analysis of
Layered Orthotropic Ring-Stiffened
Shell-of-Revolution - is a
finite-difference code

Reference NASA TN D-7179

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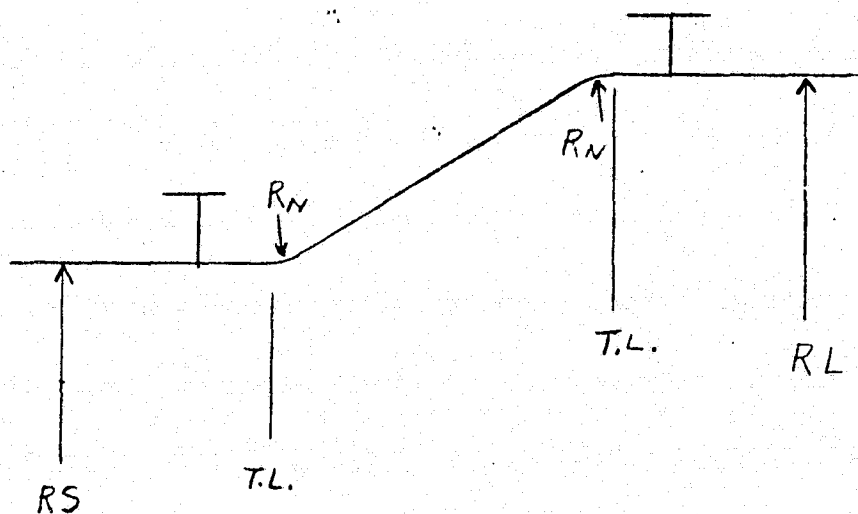
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A typical cone/cylinder is shown below.



⊕

Loading

Internal pressure = 119 psig for
Design condition

Internal pressure = $1.5(119) + \text{water head}$
for Hydro test condition

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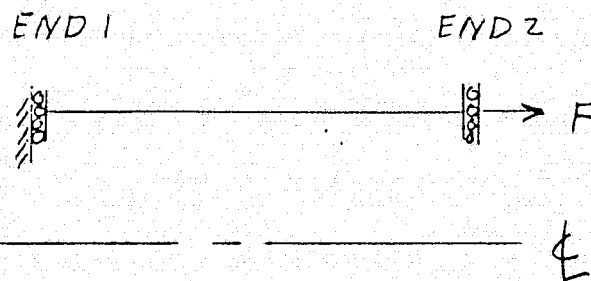
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All pressure loadings remain normal to the deformed surface

Boundary Conditions

Symmetric B.C. were applied to each end of the model

END 1 was fixed in the axial direction. A boundary force of $\frac{1}{2}PR$ (lb/in of circ.) was applied to end 2



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R1 to S2

R1 to S2

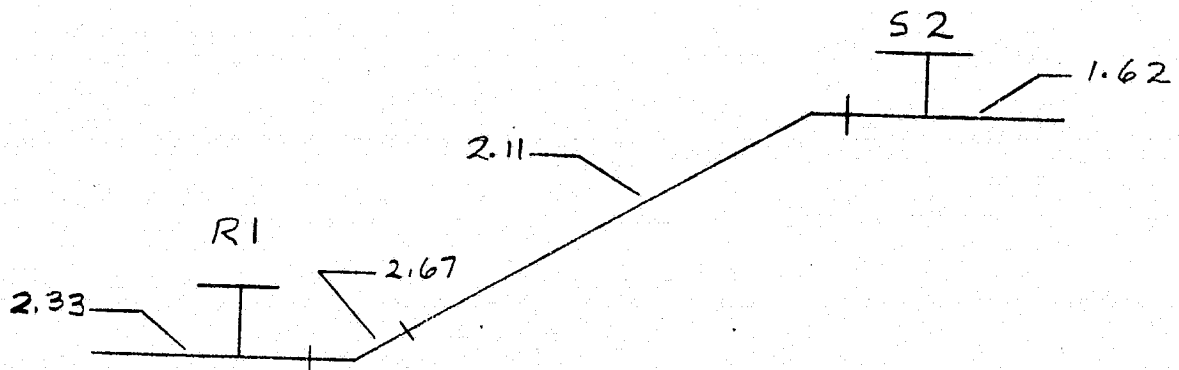


Fig 1 computer plot of geometry

Fig 2 Average net-section hoop stress
 $P = 119$ psi

Fig 3 Inside surface stress
longitudinal & hoop
 $P = 119$ psi

Fig 4 Outside surface stress
longitudinal & hoop
 $P = 119$ psi

Fig 5 Radial displacement
 $P = 119$ psi

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_____ R1 to S2 _____

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Knuckle region at small dia cylinder

This model did not include the influence from corner #4 (elliptical ring R1). This region was considered in detail in the analyses of corner #4. See corner #4 (VOL 45) analyses of this region.

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F. 15 S2.

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Knuckle region at large dia cylinder

Membrane stress (intensity)

Primary local membrane stress intensity
(Fig 2, 3 & 4) $P = 119 \text{ psi}$

$$\sigma_1 = -14.5 \text{ KSI}$$

$$\sigma_2 = \frac{28 + (-14.8)}{2} = 6.6 \text{ KSI}$$

$$\sigma_3 = -\frac{119}{2} = -.06 \text{ KSI}$$

$$S_{12} = \sigma_1 - \sigma_2 = -14.5 - 6.6 = -21.1 \text{ KSI}$$

$$S_{23} = 6.6 - (-.06) = 6.66 \text{ KSI}$$

$$S_{31} = -.06 - (-14.5) = 14.44 \text{ KSI}$$

$$P_L = |-21.1| = 21.1 \text{ KSI}$$

$$P_L \leq 1.5 S_m$$

$$21.1 < 1.5(20) = 30$$

O.K.

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P1 to S2

The meridional length at a stress intensity of $1.1 S_m$ ($1.1 \times 20 = 22 \text{ KSI}$) is 0. The peak stress intensity is less than $1.1 S_m$

$$0 < \sqrt{RT}$$

\therefore This stress intensity in this region is a local membrane stress intensity

General Membrane Stress Intensity

$$\sigma_1 = 18.0 \text{ KSI}$$

$$\sigma_2 = \frac{9+9}{2} = 9 \text{ KSI}$$

$$\sigma_3 = -\frac{.119}{2} = -.06 \text{ KSI}$$

REPRODUCIBILITY OF THE
ORIGINAL PAGE IS POOR

$$S_{12} = 18.0 - 9 = 9.0 \text{ KSI}$$

$$S_{23} = 9 - (-.06) = 9.06 \text{ KSI}$$

$$S_{31} = -.06 - 18 = -18.06 \text{ KSI}$$

$$P_m = |-18.06| = 18.06 \text{ KSI}$$

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_____ PL to 52 _____

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$$P_m \leq S_m$$

$$18.06 < 20 \text{ KSI} \quad \text{O.K.}$$

General principal membrane stress

$$S = 18.0 \text{ KSI}$$

$$S < 23.7 \text{ KSI}$$

$$18.0 < 23.7 \text{ KSI} \quad \text{O.K.}$$

The membrane stress (intensity)
for the region meets the stress
evaluation criteria.

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R1 to S2Primary Plus Secondary Stress

Inside Surface

$$\sigma_1 = 28 \text{ KSI}$$

$$\sigma_2 = -14.2 \text{ KSI}$$

$$\sigma_3 = -\frac{119}{2} = -0.06 \text{ KSI}$$

$$S_{12} = 28.0 - (-14.2) = 42.2 \text{ KSI}$$

$$S_{21} = -14.2 - (-0.06) = -14.14 \text{ KSI}$$

$$S_{31} = -0.06 - 28.0 = 28.06 \text{ KSI}$$

$$S = |42.2| = 42.2 \text{ KSI}$$

$$P_L + P_b + \phi < 3S_m$$

$$42.2 < 3(20) = 60 \quad \text{O.K.}$$

Outside Surface

$$\sigma_1 = -20.5 \text{ KSI}$$

$$\sigma_2 = -13.8 \text{ KSI}$$

$$\sigma_3 = 0$$

$$S_{12} = -20.5 - (-13.8) = -6.7 \text{ KSI}$$

$$S_{23} = -13.8 - 0 = -13.8 \text{ KSI}$$

$$S_{31} = 0 - 20.5 = -20.5 \text{ KSI}$$

$$S = |-20.5| = 20.5 \text{ KSI}$$

$$P_L + P_b + Q \leq 3 S_m$$

$$20.5 < 3(20) = 60 \text{ KSI} \quad \text{O.K.}$$

The primary plus secondary stress intensity meets the stress evaluation criteria

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S3 to R3

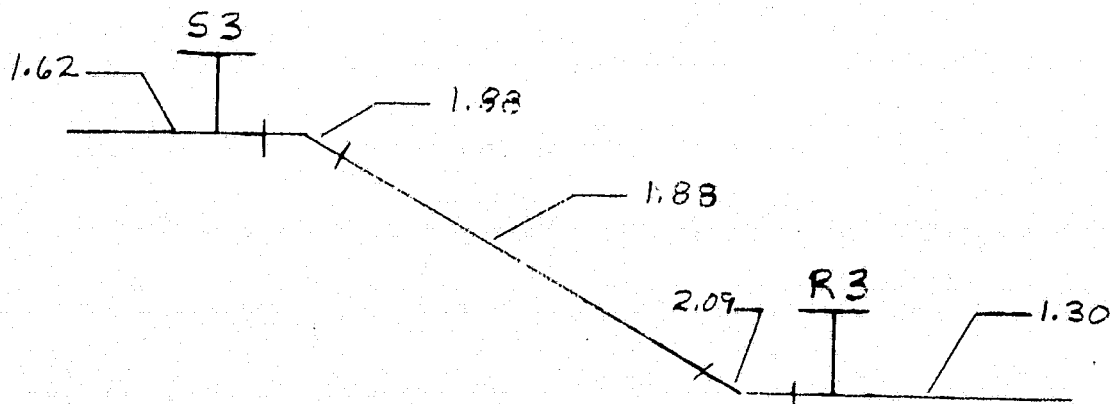


Fig 6 Computer Plot of Geometry

Fig 7 Average net section hoop stress
 $P = 119 \text{ psi}$

Fig 8 Inside surface stress
longitudinal & hoop
 $P = 119 \text{ psi}$

Fig 9 outside surface stress
longitudinal & hoop
 $P = 119 \text{ psi}$

Fig 10 Radial displacement
 $P = 119 \text{ psi}$

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BY _____ DATE _____

SUBJECT _____

SHEET NO. 11 OF _____

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23.10.83

Knuckle region at the small dia cylinder

Membrane stress (intensity)

Primary local membrane stress intensity
see Fig

$$\sigma_1 = 25 \text{ KSI}$$

$$\sigma_2 = \frac{20 + (-10)}{2} = 5 \text{ KSI}$$

$$\sigma_3 = -\frac{.119}{1} = -.06 \text{ KSI}$$

$$S_{12} = 25 - 5 = 20 \text{ KSI}$$

$$S_{23} = 5 - (-.06) = 5.06 \text{ KSI}$$

$$S_{31} = -.06 - 25 = -25.06 \text{ KSI}$$

$$S = |-25.06| = 25.06 \text{ KSI}$$

$$P_L \leq 1.5 S_m$$

$$25.06 < 1.5(20) = 30 \text{ KSI}$$

O.K.

Since the stress intensity (25.06 KSI) is equal to (within reading accuracy of the stress plots) the stress (25.0 KSI), the meridional distance vs. stress intensity is taken from fig 7

The meridional distance at a stress intensity of 1.1 S_m (1.1 x 20 = 22 KSI) is 18.5."

$$18.5 < \sqrt{RT} = \sqrt{(1168.75)(2.09)} = 18.78" \quad \text{O.K.}$$

∴ The stress intensity in the region is a local membrane stress intensity

General Membrane Stress Intensity

$$T_1 = 15.5 \text{ KSI}$$

$$T_2 = \frac{20.0 + (-10)}{2} = 5 \text{ KSI}$$

$$T_3 = -\frac{.119}{2} = -.06 \text{ KSI}$$

BY _____ DATE _____
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SUBJECT _____
S3 to R3

SHEET NO. 13 OF _____
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$$S_{12} = 15.5 - 5 = 10.5 \text{ KSI}$$

$$S_{23} = 5 - .06 = 4.94 \text{ KSI}$$

$$S_{31} = -.06 - 15.5 = -15.56 \text{ KSI}$$

$$S = |-15.56| = 15.56 \text{ KSI}$$

$$P_m \leq S_m$$

$$15.56 < 20.0 \text{ KSI} \quad \text{O.K.}$$

General principle membrane stress

$$\sigma = 15.5 \text{ KSI}$$

$$\sigma \leq S$$

$$15.5 < 23.7 \text{ KSI}$$

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The membrane stress (intensity)
meets the stress evaluation criteria.

BY _____ DATE _____

SUBJECT _____

SHEET NO. 14 OF _____

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JOB NO. _____

S3 to R3Primary Plus Secondary Stress Intensity

Inside Surface (Fig

$$\sigma_1 = 20.3 \text{ KSI}$$

$$\sigma_2 = -10.0 \text{ KSI}$$

$$\sigma_3 = -.119$$

$$S_{12} = 20.3 - (-10.0) = 30.3 \text{ KSI}$$

$$S_{23} = -10.0 - (-.119) = -9.881 \text{ KSI}$$

$$S_{31} = -.119 - 20.3 = -20.419 \text{ KSI}$$

$$S = |30.3| = 30.3 \text{ KSI}$$

$$P_L + P_b + Q \leq 35m$$

$$30.3 < 3(20) = 60 \text{ KSI} \quad \text{O.K.}$$

BY _____ DATE _____
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SUBJECT _____
S3 to R3

SHEET NO. 15 OF _____
JOB NO. _____

Outside Surface

$$\sigma_1 = 20.0 \text{ KSI}$$

$$\sigma_2 = 29.0 \text{ KSI}$$

$$\sigma_3 = 0$$

$$S_{12} = 20.0 - 29.0 = -9.0 \text{ KSI}$$

$$S_{23} = 29.0 - 0 = 29.0 \text{ KSI}$$

$$S_{31} = 0 - 20.0 = -20.0 \text{ KSI}$$

$$S = 29.0 \text{ KSI}$$

$$P_L + P_b + Q \leq 3S_m$$

$$29.0 < 3(20) = 60 \text{ KSI} \quad \text{O.K.}$$

\therefore The primary plus secondary stress intensity meets the stress evaluation criteria.

BY _____ DATE _____

SUBJECT _____

SHEET NO. 16 OF _____

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S3 to R3Knuckle region at the large dia cylinder

Membrane stress (intensity)

Primary local membrane stress intensity

see Fig

$$\sigma_1 = -10.2 \text{ KSI}$$

$$\sigma_2 = \frac{-15 + 31.0}{2} = 8 \text{ KSI}$$

$$\sigma_3 = -\frac{.119}{2} = -.06 \text{ KSI}$$

$$S_{12} = -10.2 - 8.0 = -18.2 \text{ KSI}$$

$$S_{23} = 8 - (-.06) = 8.06 \text{ KSI}$$

$$S_{31} = -.06 - (-10.2) = 10.14 \text{ KSI}$$

$$S = |-18.2| = 18.2 \text{ KSI}$$

$$P_L \leq 1.5 S_m$$

$$18.2 < 1.5(20) = 30 \text{ KSI.} \quad \text{O.K.}$$

BY _____ DATE _____

SUBJECT _____

SHEET NO. 17 OF _____

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JOB NO. _____

S3 to R3

Since the stress intensity is also $< S_m$ (20 KSI), the stress intensity meets the stress evaluation criteria.

General Membrane Stress

$S = -10.2$ KSI (largest negative stress)

or

$S = 17.0$ KSI positive stress on cone

$S \leq 18.2$ KSI

$17 < 18.2$ KSI O.K.

\therefore The membrane stress (intensity) meets the stress evaluation criteria.

BY _____ DATE _____

SUBJECT _____

SHEET NO. 18 OF _____

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S3 to R3Primary Plus Secondary Stress Intensity

Inside Surface

$$\sigma_1 = 31.0 \text{ KSI}$$

$$\sigma_2 = -3.5 \text{ KSI}$$

$$\sigma_3 = -.119 \text{ KSI}$$

$$S_{12} = 31.0 - (-3.5) = 34.5 \text{ KSI}$$

$$S_{23} = -3.5 - (-.119) = -3.381 \text{ KSI}$$

$$S_{31} = -.119 - 31.0 = -31.119 \text{ KSI}$$

$$S = |34.5| = 34.5 \text{ KSI}$$

$$P_L + P_b + Q \leq 3S_m$$

$$34.5 < 3(20) = 60 \text{ KSI} \quad \text{O.K.}$$

BY _____ DATE _____
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SUBJECT _____
S3 to R3

SHEET NO. 19 OF _____
JOB NO. _____

Outside Surface

$$\sigma_1 = -17.0 \text{ KSI}$$

$$\sigma_2 = -15.0 \text{ KSI}$$

$$\sigma_3 = 0$$

$$S_{12} = -17.0 - (-15.0) = -2.0 \text{ KSI}$$

$$S_{23} = -15.0 - 0 = -15.0 \text{ KSI}$$

$$S_{31} = 0 - (-17.0) = 17.0 \text{ KSI}$$

$$S = |17.0| = 17.0 \text{ KSI}$$

$$P_L + P_b + Q \leq 1.5 S_m$$

$$17.0 < 1.5(20) = 30 \text{ KSI} \quad \text{O.K.}$$

\therefore The primary plus secondary stress intensity meets the stress evaluation criteria.

BY _____ DATE _____
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SUBJECT _____
_____ R6 to R9 _____

SHEET NO. 20 OF _____
JOB NO. _____

R6 to R9

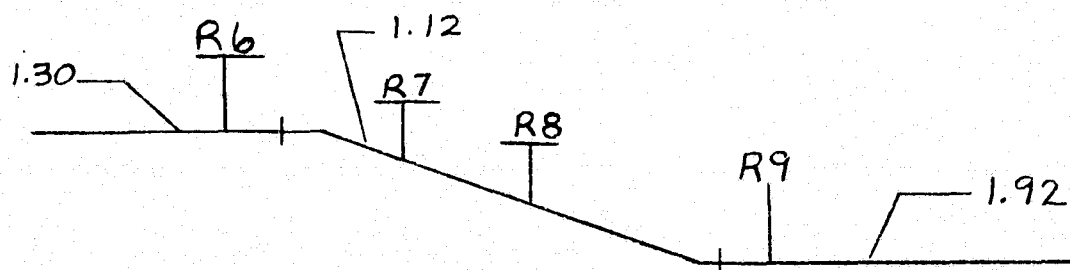


Fig 11 Computer plot of geometry

Fig 12 Average net-section hoop stress
 $P = 119$ psi

Fig 13 Inside surface stress
longitudinal & hoop
 $P = 119$ psi

Fig 14 Outside surface stress
longitudinal & hoop
 $P = 119$ psi

Fig 15 Radial displacement
 $P = 119$ psi

BY _____ DATE _____

SUBJECT _____

SHEET NO. 21 OF _____

CHKD. BY _____ DATE _____

JOB NO. _____

R6 to R9Junction region at the small dia cylinder

Membrane Stress (intensity)

Primary general membrane stress intensity

$$\sigma_1 = 14.0 \text{ KSI}$$

$$\sigma_2 = \frac{1 + 14}{2} = 7.5 \text{ KSI}$$

$$\sigma_3 = -\frac{11.9}{2} = -0.06 \text{ KSI}$$

$$S_{12} = 14.0 - 7.5 = 6.5 \text{ KSI}$$

$$S_{23} = 7.5 - (-0.06) = 7.56 \text{ KSI}$$

$$S_{31} = -0.06 - 14.0 = -14.06 \text{ KSI}$$

$$S = |-14.06| = 14.06 \text{ KSI}$$

This model did not consider the influence of corner #1.

The approximate influence for corner #1 can be determined by noting the influence of corner #4 on the cone / cylinder junction

From corner #4 analyses, the max. membrane stress was 24.10 KSI

From the salor analyses, the max membrane stress intensity was 20.8 KSI

% increase due to corner influence

$$\frac{24.10 - 20.8}{24.10} = 13.7 \%$$

BY _____ DATE _____
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SUBJECT _____
_____ R6 to R9 _____

SHEET NO. 23 OF _____
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From corner # 4 analyses, the
primary plus secondary stress
intensity was

$$S = 29.44 \text{ KSI} \quad \text{outside surface}$$

$$S = 29.46 \text{ KSI} \quad \text{inside surface}$$

For SALOR analyses

Outside surface

$$\sigma_1 = 24.0 \text{ KSI}$$

$$\sigma_2 = 14.4 \text{ KSI}$$

$$\sigma_3 = 0$$

$$S_{12} = 24.0 - 14.4 = 9.6 \text{ KSI}$$

$$S_{23} = 14.4 - 0 = 14.4 \text{ KSI}$$

$$S_{31} = 0 - 24.0 = -24.0 \text{ KSI}$$

$$S = |-24.0| = 24.0 \text{ KSI}$$

BY _____ DATE _____
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SUBJECT _____
_____ R6 to R9 _____

SHEET NO. 24 OF _____
JOB NO. _____

Inside surface

$$\sigma_1 = 17.5 \text{ KSI}$$

$$\sigma_2 = -7.0 \text{ KSI}$$

$$\sigma_3 = -\frac{.119}{2} = -.06 \text{ KSI}$$

$$S_{12} = 17.3 - (-7.0) = 24.3 \text{ KSI}$$

$$S_{23} = -7.0 - (-.06) = -6.94 \text{ KSI}$$

$$S_{31} = -.06 - 17.3 = 17.56 \text{ KSI}$$

$$S = |24.3| = 24.3 \text{ KSI}$$

% increase due to corner influence
primary plus secondary stress intensity

outside surface

$$\frac{29.44 - 24.0}{29.44} = 18.4\%$$

Inside Surface

$$\frac{29.46 - 24.3}{29.46} = 17.5\%$$

BY _____ DATE _____
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SUBJECT _____
R6 to R9

SHEET NO. 25 OF _____
JOB NO. _____

INCREASE ^{membrane} stress intensity at cone/
cylinder junction near R9 by
13.7%

$$S = 1.137 \times 14.06 = 15.99 \text{ KSI}$$

$$P_m \leq S_m$$

$$15.99 < 20.0 \text{ KSI} \quad \text{O.K.}$$

General Principal membrane stress

$$\sigma = 13.0$$

INCREASE by 13.7% due to corner influence

$$\sigma = 1.137 \times 13.0 = 14.78 \text{ KSI}$$

$$\sigma \leq 18.2 \text{ KSI}$$

$$14.78 < 18.2 \text{ KSI} \quad \text{O.K.}$$

∴ The membrane stress (intensity) for this region meets the stress evaluation criteria.

BY _____ DATE _____ SUBJECT _____ SHEET NO. 26 OF _____
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 _____ Pls to B9 _____

Primary Plus Secondary Stress Intensity

Inside Surface

$$\sigma_1 = 13.5 \text{ KSI}$$

$$\sigma_2 = 1 \text{ KSI}$$

$$\sigma_3 = -.118 \text{ KSI}$$

$$S_{12} = 13.5 - 1 = 12.5 \text{ KSI}$$

$$S_{23} = +1 - (-.12) = 1.12 \text{ KSI}$$

$$S_{31} = -.12 - 13.5 = -13.65 \text{ KSI}$$

$$S = |13.65| = 13.65 \text{ KSI}$$

INCREASE S by 17.5 % (see p.)

$$S = 1.175 (13.65) = 16.04 \text{ KSI}$$

$$P_L + P_H + Q \leq 3 S_m$$

$$16.04 < 3(20) = 60.0 \text{ KSI}$$

O.K.

BY _____ DATE _____
CHKD. BY _____ DATE _____

SUBJECT _____
R6 to R9

SHEET NO. 4 OF _____
JOB NO. _____

Outside Surface

$$\sigma_1 = 16.0 \text{ KSI}$$

$$\sigma_2 = 14.0 \text{ KSI}$$

$$\sigma_3 = 0$$

$$S_{12} = 16.0 - 14.0 = 2.0 \text{ KSI}$$

$$S_{23} = 14.0 - 0 = 14.0 \text{ KSI}$$

$$S_{31} = 0 - 16.0 = -16.0 \text{ KSI}$$

$$S = |-16.0| = 16.0 \text{ KSI}$$

INCREASE S by 18.4% (see p.

$$S = 1.184 \times 16.0 = 18.94 \text{ KSI}$$

$$P_L + P_b + Q \leq 3\sigma_m$$

$$18.94 < 3(20) = 60.0 \text{ KSI}$$

\therefore The primary plus secondary stress intensity for this region meets the stress evaluation criteria.

BY _____ DATE _____
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SUBJECT _____
R10 to R9

SHEET NO. 28 OF _____
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Junction region at the large dia. cylinder

Membrane stress (intensity)

General membrane stress

$$\sigma_1 = 8.0 \text{ KSI}$$

$$\sigma_2 = \frac{-9 + 27}{2} = 9 \text{ KSI}$$

$$\sigma_3 = -\frac{1.19}{2} = -.06 \text{ KSI}$$

$$S_{12} = 8.0 - 9 = -1 \text{ KSI}$$

$$S_{23} = 9 - (-.06) = 9.06 \text{ KSI}$$

$$S_{31} = -.06 - 8.0 = -8.06 \text{ KSI}$$

$$S = |9.06| = 9.06 \text{ KSI}$$

$$P_m \leq S_m$$

$$9.06 < 20.0 \text{ KSI} \quad \text{O.K.}$$

This region meets the criteria for
general membrane stress intensity

BY _____ DATE _____

SUBJECT _____

SHEET NO. 29 OF _____

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Rlp to P7

General Membrane stress

$\sigma = 8.0$ at junction

or

$\sigma = 16.0$ on large cylinder

$$\sigma \leq S$$

$$16 < 18.2 \text{ ksi}$$

The region meets the stress criteria
for the general membrane stress.

BY _____ DATE _____

SUBJECT _____

SHEET NO. 30 OF _____

CHKD. BY _____ DATE _____

JOB NO. _____

P16 to R9Primary Plus Secondary Stress Intensity

Inside Surface

$$\sigma_1 = 27.6 \text{ KSI}$$

$$\sigma_2 = 14.0 \text{ KSI}$$

$$\sigma_3 = -.119 = -.12 \text{ KSI}$$

$$S_{12} = 27.6 - 13.6 = 14.0 \text{ KSI}$$

$$S_{23} = 14.0 - (-.12) = 14.12 \text{ KSI}$$

$$S_{31} = -.12 - 27.6 = -27.72 \text{ KSI}$$

$$S = |-27.72| = 27.72 \text{ KSI}$$

$$P_L + P_b + Q \leq 3S_m$$

$$27.72 < 3(20) = 60.0 \text{ KSI} \quad \text{O.K.}$$

BY _____ DATE _____
CHKD. BY _____ DATE _____

SUBJECT _____
_____ R6 to R9 _____

SHEET NO. 31 OF _____
JOB NO. _____

Outside Surface

$$\sigma_1 = -9.0 \text{ KSI}$$

$$\sigma_2 = 3.5 \text{ KSI}$$

$$\sigma_3 = 0$$

$$S_{12} = -9.0 - 3.5 = 12.5 \text{ KSI}$$

$$S_{23} = 3.5 - 0 = 3.5 \text{ KSI}$$

$$S_{31} = 0 - (-9.0) = +9.0 \text{ KSI}$$

$$S = |12.5| = 12.5 \text{ KSI}$$

$$P_c + P_b + Q \leq 3 S_m$$

$$12.5 < 3(20) = 60 \text{ KSI}$$

The primary plus secondary stress intensity meets the stress evaluation criteria

BY _____ DATE _____

SUBJECT _____

SHEET NO. 32 OF _____

CHKD. BY _____ DATE _____

JOB NO. _____

R6 to R9

The stresses in the region of R7 to R8 are approximately the same as the junction regions. Since the stresses in the junction meet the criteria by a large margin, a detail summary of the stresses at R7 and R8 is not given in this stress evaluation.

BY _____ DATE _____
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SUBJECT _____
R10 to R12

SHEET NO. 33 OF _____
JOB NO. _____

R10 to R12

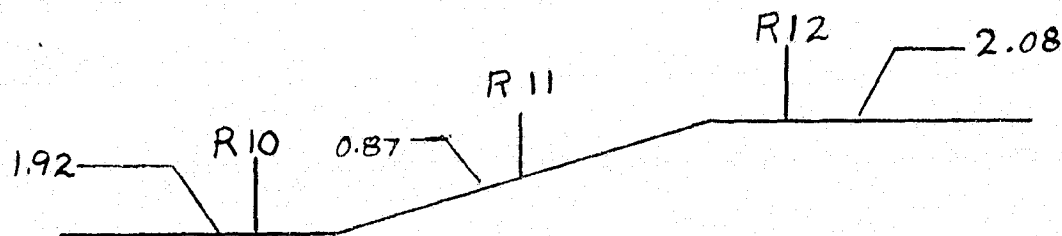


Fig 16 Computer plot of geometry

Fig 17 Average net-section hoop stress
 $P = 119 \text{ psi}$

Fig 18 Inside surface stress
longitudinal + hoop
 $P = 119 \text{ psi}$

Fig 19 Outside surface stress
longitudinal + hoop
 $P = 119 \text{ psi}$

Fig 20 Radial displacement
 $P = 119 \text{ psi}$

BY _____ DATE _____

SUBJECT _____

SHEET NO. 34 OF _____

CHKD. BY _____ DATE _____

JOB NO. _____

R10 to R12

This model did not consider
the influence from corner
1 and corner # 2.

% increase due to corner
influence for membrane
see p.

% increase 13.7 %

primary plus secondary stress intensity
(see p.

% increase

Outside Surface 18.4 %

Inside surface 17.5 %

BY _____ DATE _____
CHKD. BY _____ DATE _____

SUBJECT _____

SHEET NO. 35 OF _____

JOB NO. _____

R12 to R12

Junction region at the small dia cylinder

Membrane stress (intensity)

Primary local membrane stress intensity

$$\sigma_1 = 17.0 \text{ KSI}$$

$$\sigma_2 = \left[\frac{17.0 + 0}{2} \right] = 8.5 \text{ KSI}$$

$$\sigma_3 = -\frac{.119}{2} = -.06 \text{ KSI}$$

$$S_{12} = 17.0 - 8.5 = 8.5 \text{ KSI}$$

$$S_{23} = 8.5 - (-.06) = 8.56 \text{ KSI}$$

$$S_{31} = -.06 - 17.0 = -17.06 \text{ KSI}$$

$$S = |-17.06| = 17.06 \text{ KSI}$$

$$S = 1.137(17.06) = 19.39$$

$$P_m \leq S_m$$

$$19.39 < 20.0 \text{ KSI}$$

O.K.

The general membrane stress intensity
for this region meets the
stress evaluation criteria

BY _____ DATE _____ SUBJECT _____ SHEET NO. 36 OF _____
CHKD. BY _____ DATE _____ JOB NO. _____
_____ R10 to R12 _____

General principal stress

$$\sigma = 1.137(16) = 18.19 \text{ KSI}$$

$$\sigma \leq S$$

$$18.19 < 18.2 \text{ KSI} \quad \text{O.K.}$$

This region meets the criteria
for membrane stress (intensity).

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BY _____ DATE _____

SUBJECT _____

SHEET NO. 37 OF _____

CHKD. BY _____ DATE _____

R10 to R12

JOB NO. _____

Primary Plus Secondary Stress intensity

Inside Surface

$$\sigma_1 = 16.5 \text{ KSI}$$

$$\sigma_2 = 7.0 \text{ KSI}$$

$$\sigma_3 = -.119 \text{ KSI}$$

$$S_{12} = 16.5 - 7.0 = 9.5 \text{ KSI}$$

$$S_{23} = 7.0 - (-.12) = 7.12 \text{ KSI}$$

$$S_{31} = -.12 - 16.5 = -16.62 \text{ KSI}$$

$$S = |-16.62| = 16.62 \text{ KSI}$$

Increase due to corner 17.5%

$$S = 1.175(16.62) = 19.53 \text{ KSI}$$

$$P_L + P_b + Q \leq 3 S_m$$

$$19.53 < 3(20) = 60.0 \text{ KSI}$$

O.K.

BY _____ DATE _____

SUBJECT _____

SHEET NO. 38 OF _____

CHKD. BY _____ DATE _____

JOB NO. _____

R10 to R12

Outside Surface

$$\sigma_1 = 19.0 \text{ KSI}$$

$$\sigma_2 = 12.0 \text{ KSI}$$

$$\sigma_3 = 0$$

$$S_{12} = 19.0 - 12.0 = 7.0 \text{ KSI}$$

$$S_{23} = 12.0 - 0 = 12.0 \text{ KSI}$$

$$S_{31} = 0 - 19.0 = -19.0 \text{ KSI}$$

$$S = |-19.0| = 19.0 \text{ KSI}$$

increase due to corner 18.4 %

$$S = 1.184 (19.0) = 22.50 \text{ KSI}$$

$$P_L + P_b + Q \leq 3 S_m$$

$$22.50 < 3(20) = 60.0 \text{ KSI} \quad \text{O.K.}$$

The primary plus secondary stress intensity for this region meet the evaluation criteria.

REPRODUCIBILITY OF THE
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Junction at the large dia cylinder

Membrane Stress (intensity)

$$\sigma_1 = 18.2 \text{ KSI}$$

$$\sigma_2 = \frac{12.2 + 6}{2} = 9.1 \text{ KSI}$$

$$\sigma_3 = -\frac{.119}{2} = -.06 \text{ KSI}$$

$$S_{12} = 18.2 - 9.1 = 9.1 \text{ KSI}$$

$$S_{23} = 9.1 - (-.06) = 9.06 \text{ KSI}$$

$$S_{31} = -.06 - 18.2 = -18.26 \text{ KSI}$$

$$S = |-18.26| = 18.26 \text{ KSI}$$

Assume the same % increase for this region as the small dia region (13.7 %)

$$\therefore S = 1.137(18.26) = 20.76 \text{ KSI}$$

$$P_L \leq 1.5 S_m$$

$$20.76 < 1.5(20) = 30.0 \text{ KSI} \quad \text{O.K.}$$

The stress intensity (20.76 KSI) < 1.1 S_m (1.1 × 20 = 22.0 KSI)

∴ The meridional length over which the stress intensity exist is 0

∴ This stress is a local membrane stress intensity

General Membrane Stress intensity

$$\sigma_1 = 18.0 \text{ KSI}$$

$$\sigma_2 = \frac{9+9}{2} = 9.0 \text{ KSI}$$

$$\sigma_3 = -\frac{119}{2} = -59.5 \text{ KSI}$$

$$S_{12} = 18.0 - 9.0 = 9.0 \text{ KSI}$$

$$S_{23} = 9.0 - (-59.5) = 68.5 \text{ KSI}$$

$$S_{31} = -59.5 - 18.0 = -77.5 \text{ KSI}$$

$$S = |-77.5| = 77.5 \text{ KSI}$$

BY _____ DATE _____
CHKD. BY _____ DATE _____

SUBJECT _____
_____ R10 to R12 _____

SHEET NO. 41 OF _____
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$$P_m \leq S_m$$

$$18.06 < 20.0 \text{ KSI}$$

O.K

General membrane stress

$$\sigma = 18.0 \text{ KSI}$$

$$\sigma \leq S$$

$$18.0 < 18.2 \text{ KSI}$$

O.K.

\therefore The membrane stress (intensity) meets the stress evaluation criteria.

Primary Plus Secondary Stress Intensity

Inside Surface

$$\sigma_1 = 18.0 \text{ KSI}$$

$$\sigma_2 = 11.0 \text{ KSI}$$

$$\sigma_3 = -.119 \text{ KSI}$$

$$S_{12} = 18.0 - 11.0 = 7.0 \text{ KSI}$$

$$S_{23} = 11.0 - (-.12) = 11.12 \text{ KSI}$$

$$S_{31} = -.12 - 18.0 = -18.12 \text{ KSI}$$

$$S = |-18.12| = 18.12 \text{ KSI}$$

increase due to corner influence

$$S = 1.175 (18.12) = 21.29 \text{ KSI}$$

$$P_L + P_b + Q \leq 3S_m$$

$$21.29 < 3(20) = 60 \text{ KSI} \quad \text{O.K.}$$

BY _____ DATE _____

SUBJECT _____

SHEET NO. 43 OF _____

CHKD. BY _____ DATE _____

JOB NO. _____

R10 to R12

Outside Surface

$$\sigma_1 = 18.0 \text{ KSI}$$

$$\sigma_2 = 12.0 \text{ KSI}$$

$$\sigma_3 = 0$$

$$S_{12} = 18.0 - 12.0 = 6.0 \text{ KSI}$$

$$S_{23} = 12 - 0 = 12.0 \text{ KSI}$$

$$S_{31} = 0 - 18.0 = -18.0 \text{ KSI}$$

$$S = |-18.0| = 18.0 \text{ KSI}$$

INCREASE due to corner

$$S = 1.184 (18.0) = 21.31 \text{ KSI}$$

$$P_L + P_b + q \leq 3 S_m$$

$$21.31 < 3(20) = 60 \text{ KSI} \quad \text{O.K.}$$

The primary plus secondary stress intensity meets the criteria.

BY _____ DATE _____

SUBJECT _____

SHEET NO. 44 OF _____

CHKD. BY _____ DATE _____

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R10 to R12

The analysis of this section includes a ring (R11) located on the cone section. This ring has subsequently been removed and ring R12 renumbered as R11. This analysis was not redone with this ring removed since the effects of it were negligible.

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BY _____ DATE _____

SUBJECT _____

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R13A to S8

R13A to S8

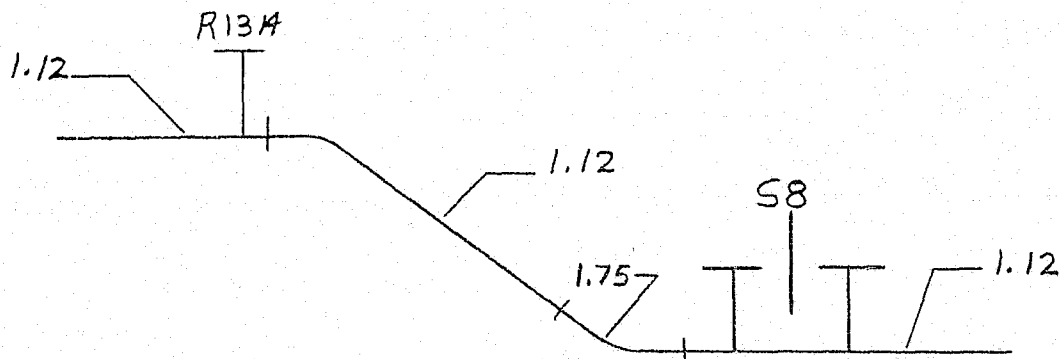


Fig 21 Computer plot of geometry

Fig 22 Average net-section hoop stress
 $P = 119$ psi

Fig 23 Inside surface stress
longitudinal & hoop
 $P = 119$ psi

Fig 24 Outside surface stress
longitudinal & hoop
 $P = 119$ psi

Fig 25 Radial displacement
 $P = 119$ psi

Knuckle region at the small dia cylinder

Membrane stress (intensity)

Primary local membrane stress
intensity see Fig 22

$$\sigma_1 = 24.0 \text{ KSI}$$

$$\sigma_2 = \frac{-9.0 + 17.5}{2} = 4.25 \text{ KSI}$$

$$\sigma_3 = \frac{-1.19}{2} = -.06 \text{ KSI}$$

$$S_{12} = 24.0 - 4.25 = 19.75 \text{ KSI}$$

$$S_{23} = 4.25 - (-.06) = 4.31 \text{ KSI}$$

$$S_{31} = -.06 - 24.0 = -24.06 \text{ KSI}$$

$$S = |-24.06| = 24.06 \text{ KSI}$$

$$P_L = 1.5 S_m$$

$$24.06 < 1.5(20) = 30 \text{ KSI}$$

Since the stress intensity (24.06 ksi) is equal to (within reading accuracy of the stress plots) the stress (24.0 ksi), the meridional distance vs. stress intensity is taken from Fig

The meridional distance at a stress intensity of 11.5m (11 x 20 = 22 ksi) is 13.5".

$$13.5" < \sqrt{RT} = \sqrt{132(1.75)} = 15.20" \quad \text{O.K.}$$

\therefore The stress intensity in the region is a local membrane stress intensity.

General Membrane Stress Intensity

$$\sigma_1 = 12.3 \text{ KSI}$$

$$\sigma_2 = \frac{1.2 + 12.8}{2} = 7.0 \text{ KSI}$$

$$\sigma_3 = \frac{-1.19}{2} = -0.6 \text{ KSI}$$

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R13A to S8

$$S_{12} = 12.3 - 7.0 = 5.3 \text{ KSI}$$

$$S_{23} = 7.0 - (-0.06) = 7.06 \text{ KSI}$$

$$S_{31} = -0.06 - 12.3 = -12.36$$

$$S = |-12.36| = 12.36 \text{ KSI}$$

$$P_m \leq S_m$$

$$12.36 < 20.0 \text{ KSI}$$

General principle membrane stress

$$\sigma = 12.3$$

$$\sigma \leq S$$

$$12.3 < 23.7 \text{ KSI}$$

The membrane stress (intensity) meets the stress evaluation criteria.

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R13A to S8

Primary Plus Secondary Stress Intensity

Inside surface (Fig

$$\sigma_1 = 20.0 \text{ KSI}$$

$$\sigma_2 = -9.0 \text{ KSI}$$

$$\sigma_3 = -.119 \text{ KSI}$$

$$s_{12} = 20.0 - (-9.0) = 29.0 \text{ KSI}$$

$$s_{23} = -9.0 - (-.119) = -8.881 \text{ KSI}$$

$$s_{31} = -.119 - 20.0 = -20.119 \text{ KSI}$$

$$s = |29.0| = 29.0 \text{ KSI}$$

$$P_L + P_b + Q \leq 3 S_m$$

$$29.0 < 3(20) = 60 \text{ KSI}$$

O.K.

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RIBA to SG

Outside surface

$$\sigma_1 = 27.5 \text{ KSI}$$

$$\sigma_2 = 17.5 \text{ KSI}$$

$$\sigma_3 = 0.$$

$$S_{12} = 27.5 - 17.5 = 10.0 \text{ KSI}$$

$$S_{23} = 17.5 - 0 = 17.5 \text{ KSI}$$

$$S_{31} = 0 - 27.5 = -27.5 \text{ KSI}$$

$$S = |-27.5| = 27.5 \text{ KSI}$$

$$P_L + P_b + Q \leq 3 S_m$$

$$27.5 < 3(20) = 60 \text{ KSI}$$

\therefore The primary plus secondary stress intensity meets the stress evaluation criteria.

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R13A to S8

Knuckle region at the large dia cylinder

Membrane stress (intensity)

Primary local membrane stress
intensity. see Fig 22

$$\sigma_1 = -8.3 \text{ KSI}$$

$$\sigma_2 = \frac{32.6 + (-16.2)}{2} = 8.2 \text{ KSI}$$

$$\sigma_3 = -\frac{119}{2} = -106 \text{ KSI}$$

$$S_{12} = -8.3 - 8.2 = -16.5 \text{ KSI}$$

$$S_{23} = 8.2 - (-106) = 114.2 \text{ KSI}$$

$$S_{31} = -106 - (-8.3) = -97.7 \text{ KSI}$$

$$S = |-16.5| = 16.5 \text{ KSI}$$

$$P_L \leq 1.5 S_m$$

$$16.5 < 1.5(20) = 30 \text{ KSI}$$

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RIBA to S8

Since the stress intensity is also $< S_m$ (20 KSI), the stress intensity meets the stress evaluation criteria.

General Membrane Stress

$$S = 17.0 \text{ KSI (on cylinder)}$$

$$S \leq 18.2 \text{ KSI}$$

$$17.0 < 18.2 \text{ KSI} \quad \text{O.K.}$$

\therefore The membrane stress (intensity) meets the stress evaluation criteria.

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R13A to S8

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Primary Plus Secondary Stress Intensity

Inside Surface

$$\sigma_1 = 32.8 \text{ KSI}$$

$$\sigma_2 = -1.0 \text{ KSI}$$

$$\sigma_3 = -.119 \text{ KSI}$$

$$S_{12} = 32.8 - (-1.0) = 33.8 \text{ KSI}$$

$$S_{23} = -1.0 - (-.119) = -.881 \text{ KSI}$$

$$S_{31} = -.119 - 32.8 = -32.919 \text{ KSI}$$

$$S = |33.8| = 33.8 \text{ KSI}$$

$$P_L + P_b + Q \leq 3 S_m$$

$$33.8 < 3(70) = 60 \text{ KSI}$$

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RIBA to SGOutside surface

$$\sigma_1 = -16.5 \text{ KSI}$$

$$\sigma_2 = -15.8 \text{ KSI}$$

$$\sigma_3 = 0.$$

$$s_{12} = -16.5 - (-15.8) = 0.7 \text{ KSI}$$

$$s_{23} = -15.8 - 0 = -15.8 \text{ KSI}$$

$$s_{31} = 0 - (-16.5) = 16.5 \text{ KSI}$$

$$s = |16.5| = 16.5 \text{ KSI}$$

$$P_L + P_b + Q \leq 1.5 S_m$$

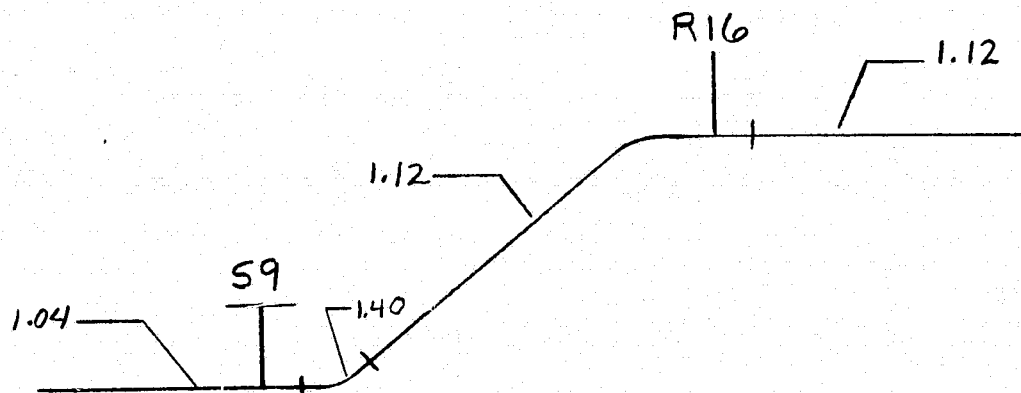
$$16.5 < 1.5(20) = 30 \text{ KSI}$$

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S9 to R16

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S9 to R16



- Fig 26 Computer plot of geometry
- Fig 27 Average net-section hoop stress
 $P = 119 \text{ psi}$
- Fig 28 Inside surface stress
longitudinal + hoop
 $P = 119 \text{ psi}$
- Fig 29 Outside surface stress
longitudinal + hoop
 $P = 119 \text{ psi}$
- Fig 30 Radial displacement
 $P = 119 \text{ psi}$

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Knuckle region at the small dia cylinder

Membrane Stress (intensity)

Primary local membrane stress intensity
see Fig 27

$$\sigma_1 = 24.4 \text{ KSI}$$

$$\sigma_2 = \frac{18.0 + (-8.0)}{2} = 5.0 \text{ KSI}$$

$$\sigma_3 = -\frac{.119}{2} = -.06 \text{ KSI}$$

$$S_{12} = 24.4 - 5.0 = 19.4 \text{ KSI}$$

$$S_{23} = 5.0 - (-.06) = 5.06 \text{ KSI}$$

$$S_{31} = -.06 - 24.4 = -24.46$$

$$S = |-24.46| = 24.46 \text{ KSI}$$

$$P_L \leq 1.5 S_m$$

$$24.46 < 1.5(20) = 30 \text{ KSI}$$

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59 to R16

Since the stress intensity (24.46 KSI) is equal to (within reading accuracy of the stress plots) the stress (24.4 KSI), the meridional distance vs. stress intensity is taken from Fig 27.

The meridional distance at a stress intensity of 1.15m ($1.1 \times 20 = 22$ KSI) is 12".

$$12" < \sqrt{RT} = \sqrt{(110)(1.40)} = 12.5"$$

\therefore the stress intensity in the region is a local membrane stress intensity.

General Membrane Stress Intensity

$$\sigma_1 = 15.0 \text{ KSI}$$

$$\sigma_2 = \frac{6.0 + 8.6}{2} = 7.3 \text{ KSI}$$

$$\sigma_3 = -\frac{.119}{2} = -.06 \text{ KSI}$$

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S9 to R16

$$S_{12} = 15.0 - 7.3 = 7.7 \text{ KSI}$$

$$S_{23} = 7.3 - (-0.06) = 7.36 \text{ KSI}$$

$$S_{31} = -0.06 - 15.0 = -15.06 \text{ KSI}$$

$$S = |-15.06| = 15.06 \text{ KSI}$$

$$P_m \leq S_m$$

$$15.06 < 20.0 \text{ KSI}$$

General principal membrane stress

$$\sigma = 15.0 \text{ KSI}$$

$$\sigma \leq S$$

$$15.0 < 23.7 \text{ KSI}$$

The membrane stress (intensity)
meets the stress evaluation criteria.

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59 to R16

Primary plus Secondary Stress Intensity

Inside Surface (Fig 28)

$$\sigma_1 = 20.5 \text{ KSI}$$

$$\sigma_2 = -8.0 \text{ KSI}$$

$$\sigma_3 = -.119$$

$$S_{12} = 20.5 - (-8.0) = 28.5 \text{ KSI}$$

$$S_{23} = -8.0 - (-.119) = -7.881 \text{ KSI}$$

$$S_{31} = -.119 - 20.5 = -20.619 \text{ KSI}$$

$$S = |28.5| = 28.5 \text{ KSI}$$

$$P_L + P_b + Q \leq 3 S_m$$

$$28.5 < 3(20) = 60 \text{ KSI} \quad \text{O.K.}$$

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S9 to R16

Outside Surface

$$\sigma_1 = 28.5 \text{ KSI}$$

$$\sigma_2 = 18.0 \text{ KSI}$$

$$\sigma_3 = 0.$$

$$S_{12} = 28.5 - 18.0 = 10.5 \text{ KSI}$$

$$S_{23} = 18.0 - 0 = 18.0 \text{ KSI}$$

$$S_{31} = 0 - 28.5 = -28.5 \text{ KSI}$$

$$S = |-28.5| = 28.5 \text{ KSI}$$

$$P_L + P_b + Q \leq 3 S_m$$

$$28.5 < 3(20) = 60 \text{ KSI}$$

\therefore The primary plus secondary stress intensity meets the stress evaluation criteria.

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S9 to R16

Knuckle region at large dia cylinder

Membrane Stress (intensity)

Primary local membrane stress intensity. see Fig 27.

$$\sigma_1 = -9.1 \text{ KSI}$$

$$\sigma_2 = \frac{-16.0 + 32.2}{2} = 8.1 \text{ KSI}$$

$$\sigma_3 = \frac{-119}{2} = -0.06 \text{ KSI}$$

$$S_{12} = -9.1 - 8.1 = -17.2 \text{ KSI}$$

$$S_{23} = 8.1 - (-0.06) = 8.16 \text{ KSI}$$

$$S_{31} = -0.06 - (-9.1) = 9.04 \text{ KSI}$$

$$S = |-17.2| = 17.2 \text{ KSI}$$

$$P_L \leq 1.5 S_m$$

$$17.2 < 1.5(20) = 30 \text{ KSI}$$

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SA to R16

Since the stress intensity is also $< S_m (20 \text{ KSI})$, the stress intensity meets the stress evaluation criteria.

General Membrane Stress

$$S = -9.1 \text{ KSI} \quad (\text{largest negative stress})$$

or

$$S = 17.0 \text{ KSI} \quad (\text{positive stress on cone})$$

$$S \leq 18.2 \text{ KSI}$$

$$17.0 < 18.2 \text{ KSI} \quad \text{o.k.}$$

\therefore The membrane stress (intensity) meets the stress evaluation criteria.

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59 TO R16

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Primary Plus Secondary Stress Intensity

Inside Surface

$$\sigma_1 = 32.5 \text{ KSI}$$

$$\sigma_2 = -2.0 \text{ KSI}$$

$$\sigma_3 = -.119 \text{ KSI}$$

$$S_{12} = 32.5 - (-2.0) = 34.5 \text{ KSI}$$

$$S_{23} = -2.0 - (-.119) = -1.881 \text{ KSI}$$

$$S_{31} = -.119 - 32.5 = -32.619 \text{ KSI}$$

$$S = |34.5| = 34.5 \text{ KSI}$$

$$P_L + P_b + Q \leq 3 S_m$$

$$34.5 < 3(70) = 60 \text{ KSI}$$

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S9 to R16

Outside Surface

$$\sigma_1 = -16.0 \text{ KSI}$$

$$\sigma_2 = -15.8 \text{ KSI}$$

$$\sigma_3 = 0$$

$$S_{12} = -16.0 - (-15.8) = -0.2 \text{ KSI}$$

$$S_{23} = -15.8 - 0 = -15.8 \text{ KSI}$$

$$S_{31} = 0 - (-16.0) = 16.0 \text{ KSI}$$

$$S = |16.0| = 16.0 \text{ KSI}$$

$$P_L + P_b + Q \leq 1.5 S_m$$

$$16.0 < 1.5(20) = 30 \text{ KSI}$$

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∴ The primary plus secondary stress intensity meets the stress evaluation criteria.

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S9 to R21

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S9 to R21

This region of the tunnel is a long shallow cone. The cone angle for the cone is shallower than the cone angle for the region between R6 + R9. Due to the fact shallow cone angles do not produce high stresses at the knuckles (Ref. Fig 11 -14 and the evaluation of R6 to R9 cone p. 20 thru 32) the area was not analyzed by finite difference methods.

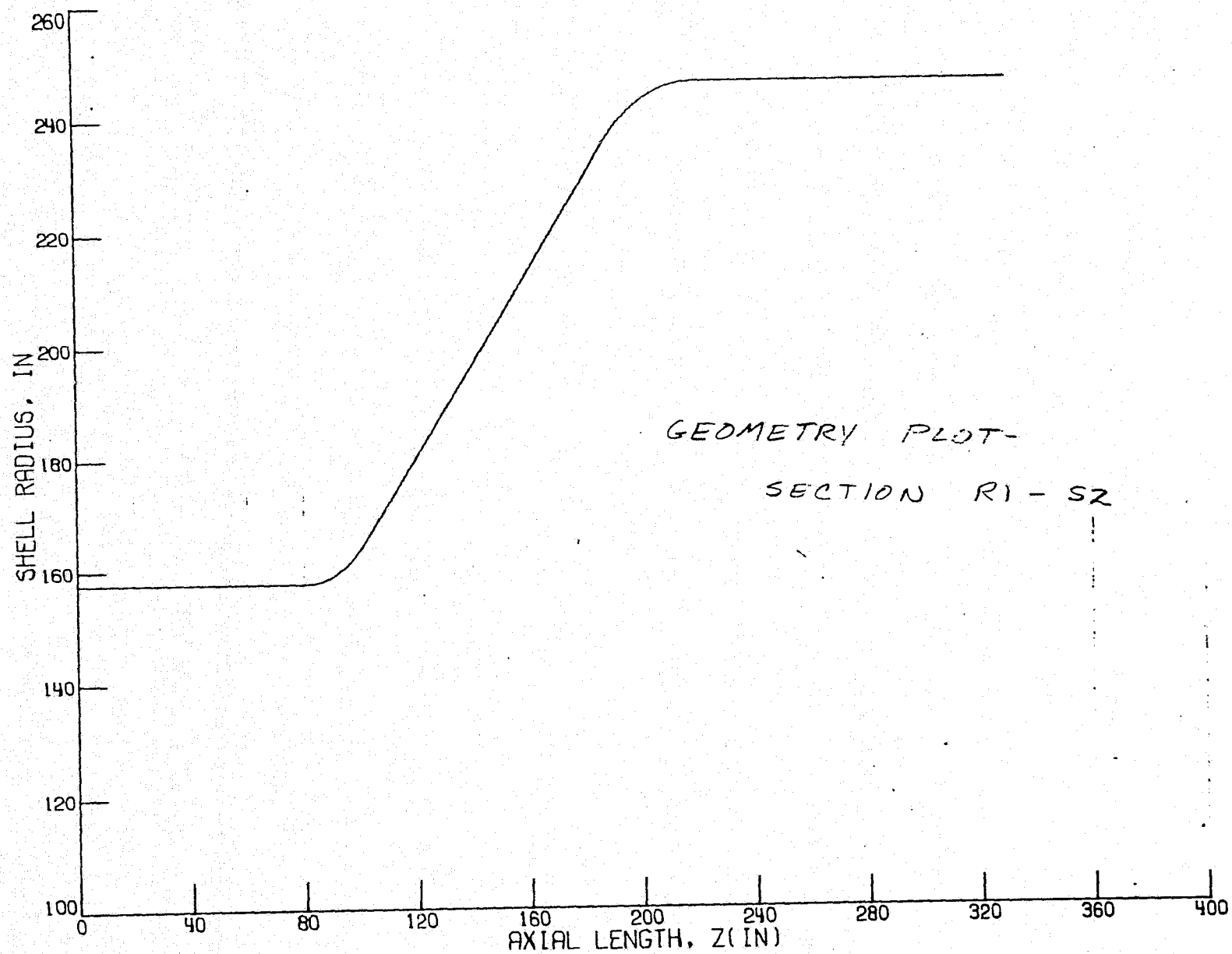


FIGURE 1

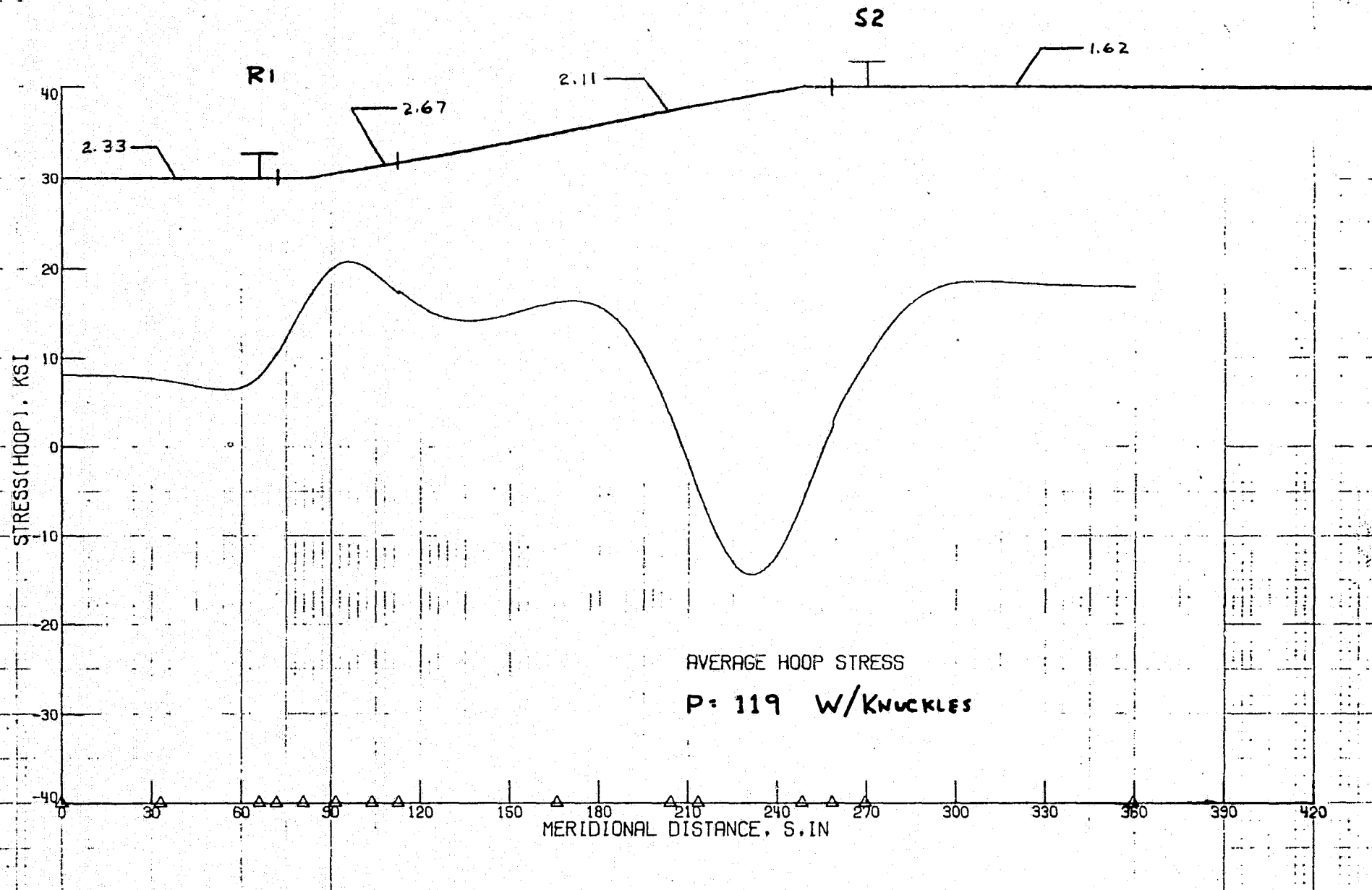


FIGURE 2

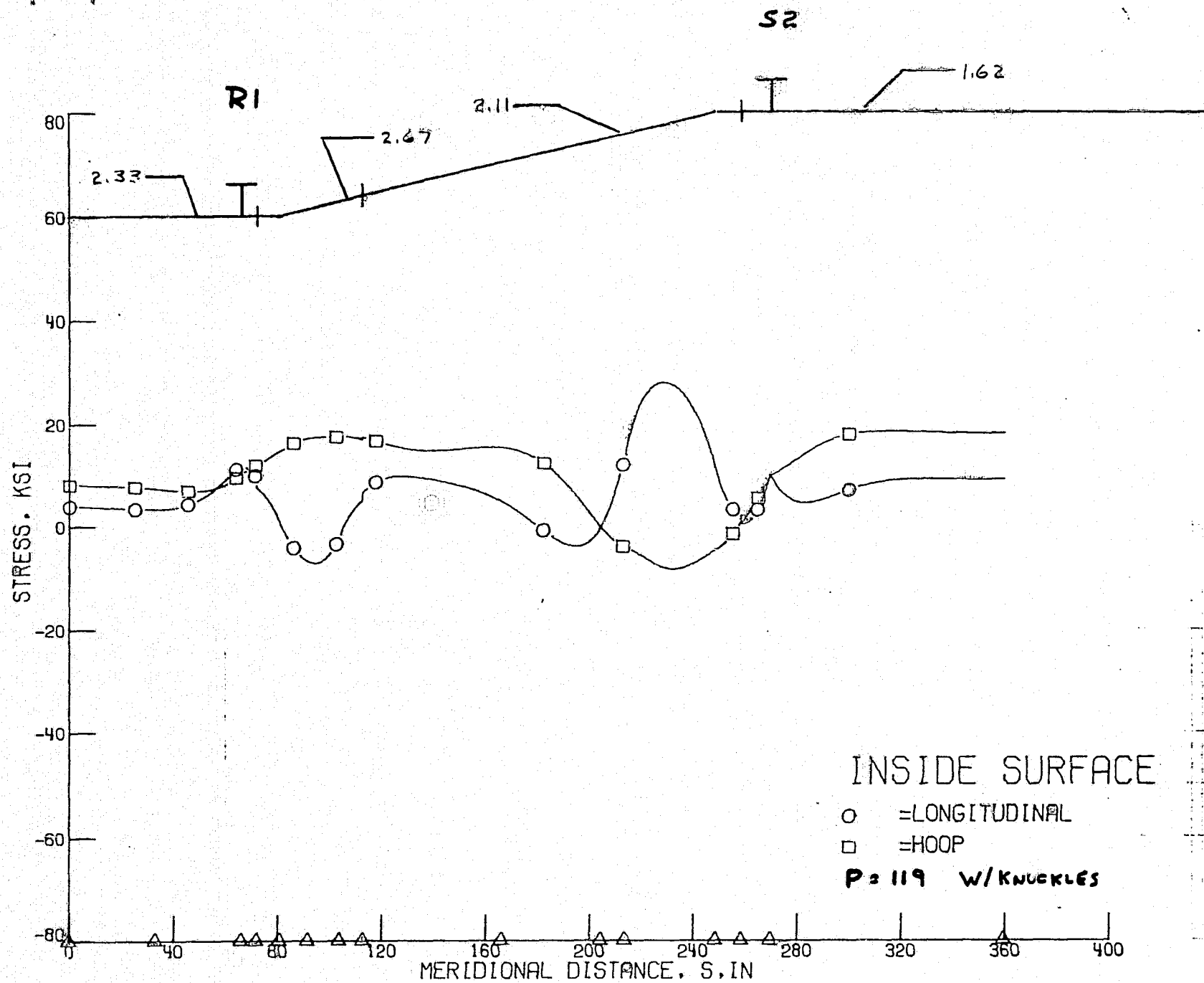


FIGURE 3

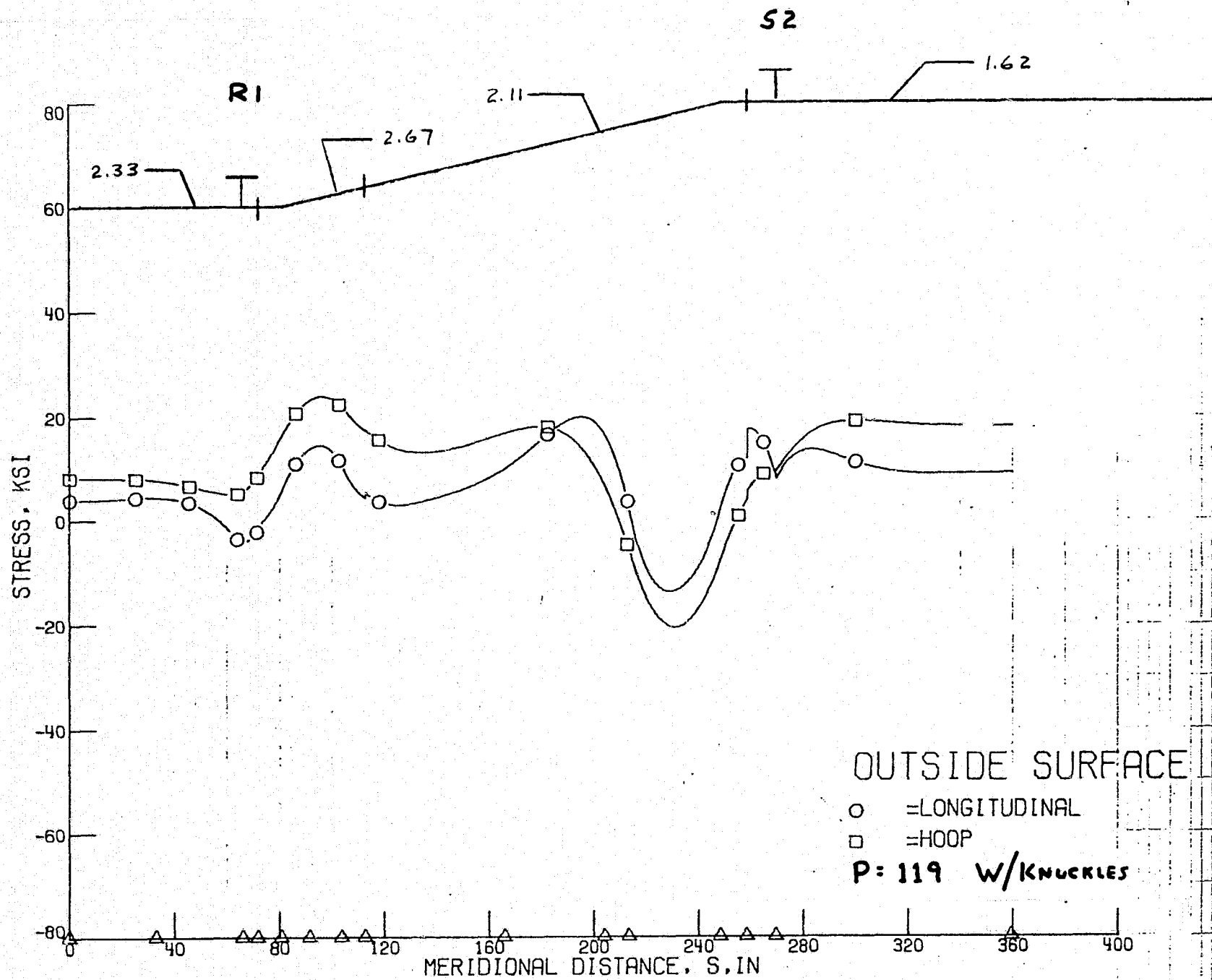


FIGURE 4

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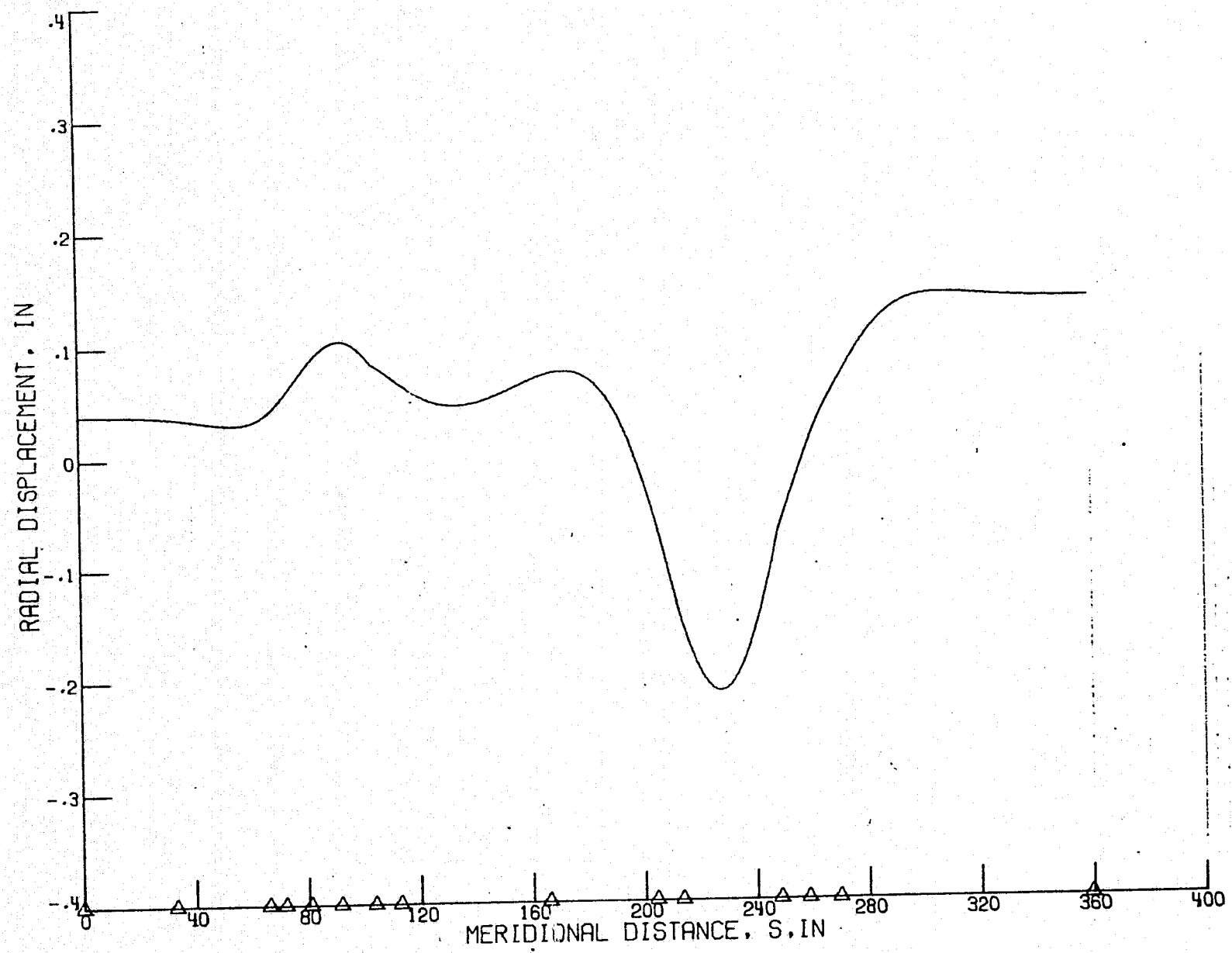


FIGURE 5

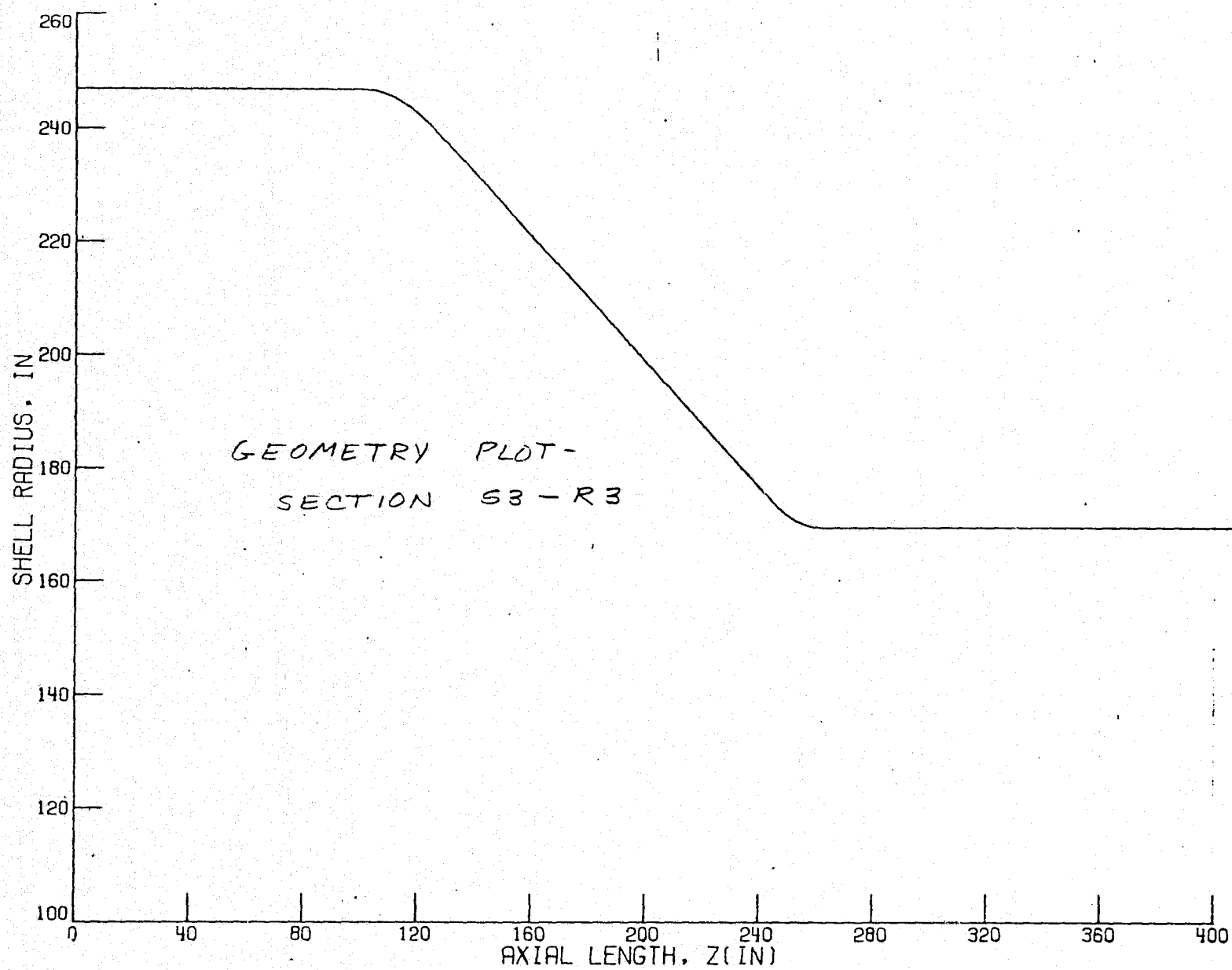


FIGURE 6

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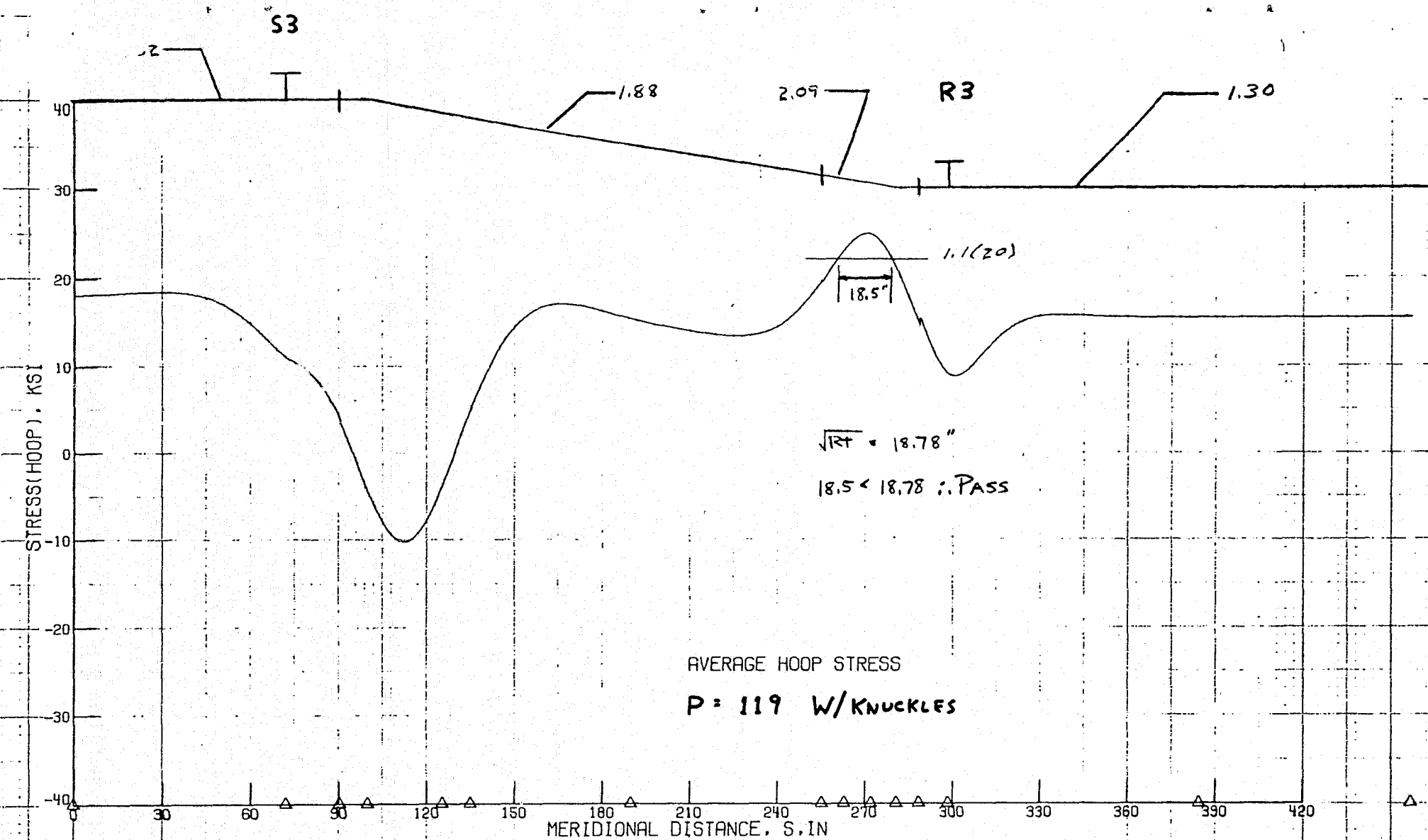


FIGURE 7

S3

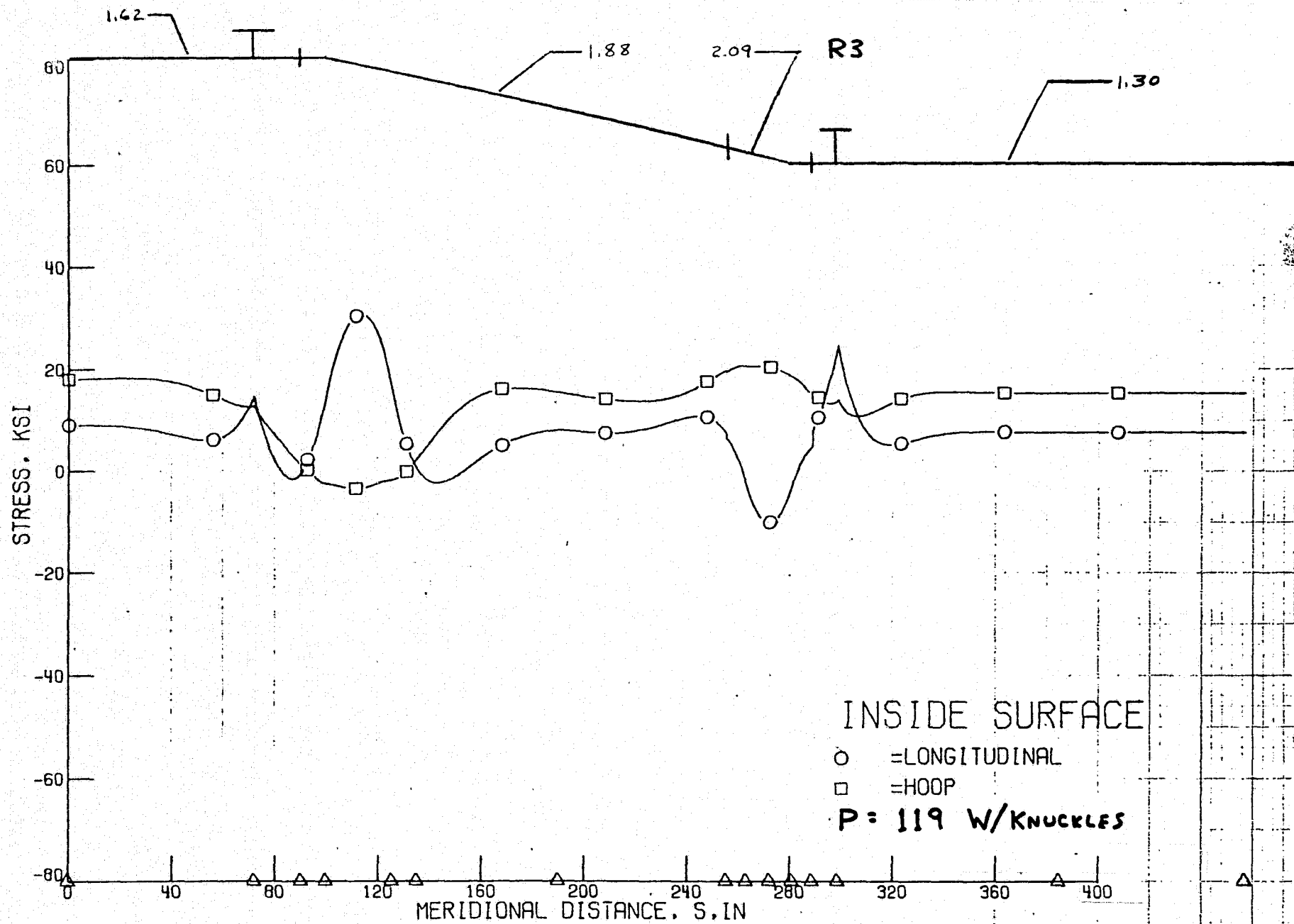


FIGURE 8

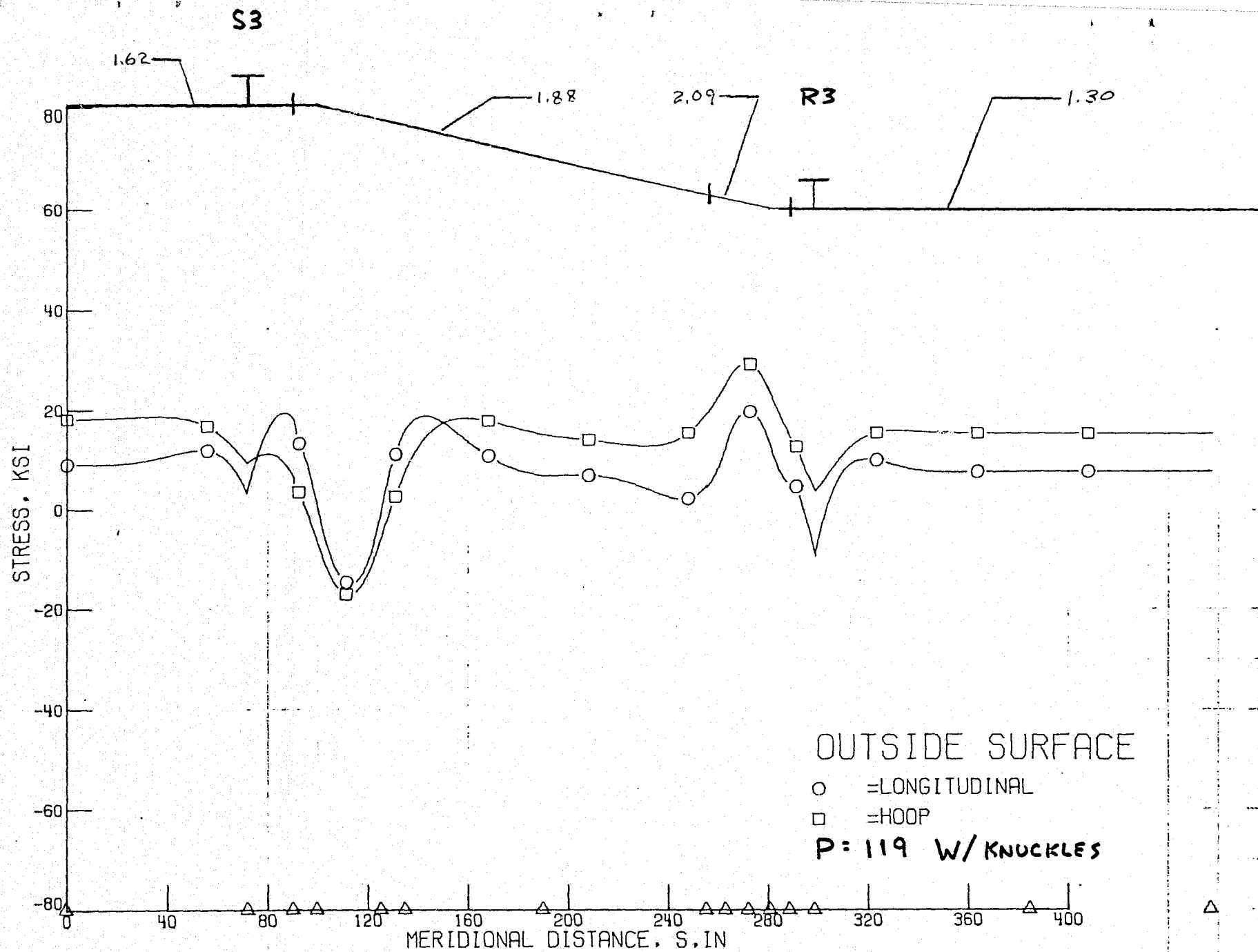


FIGURE 7

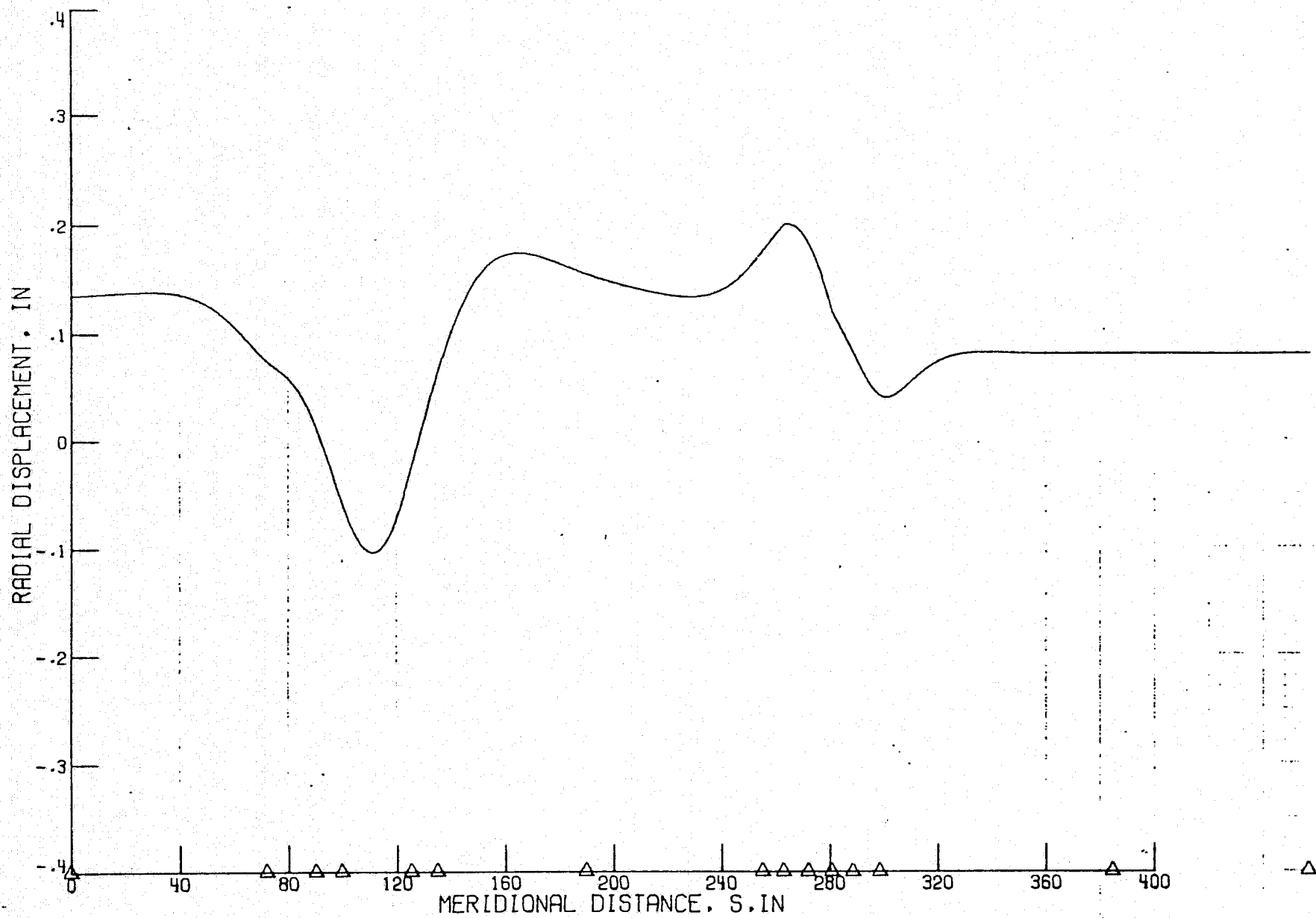


FIGURE 10

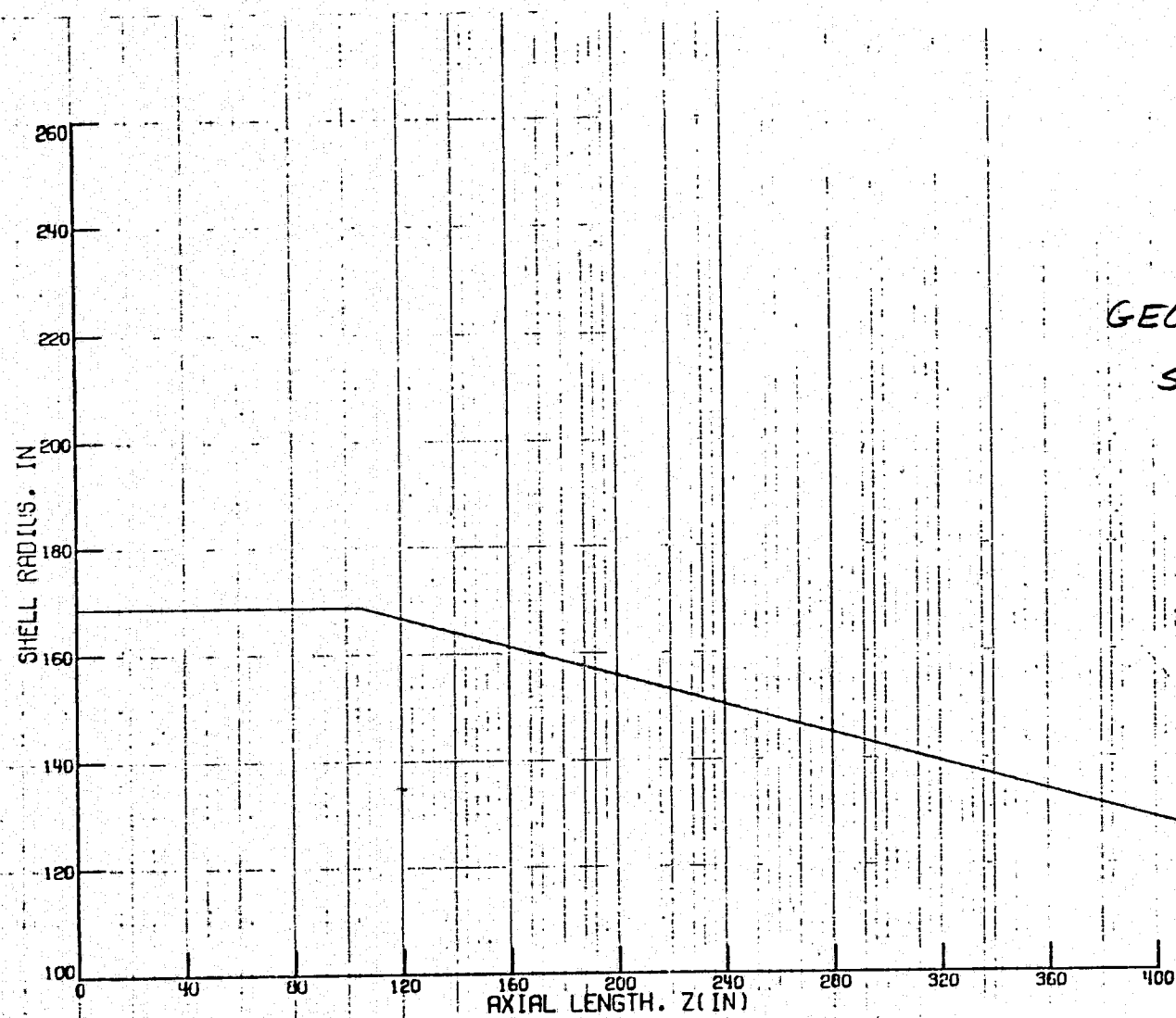
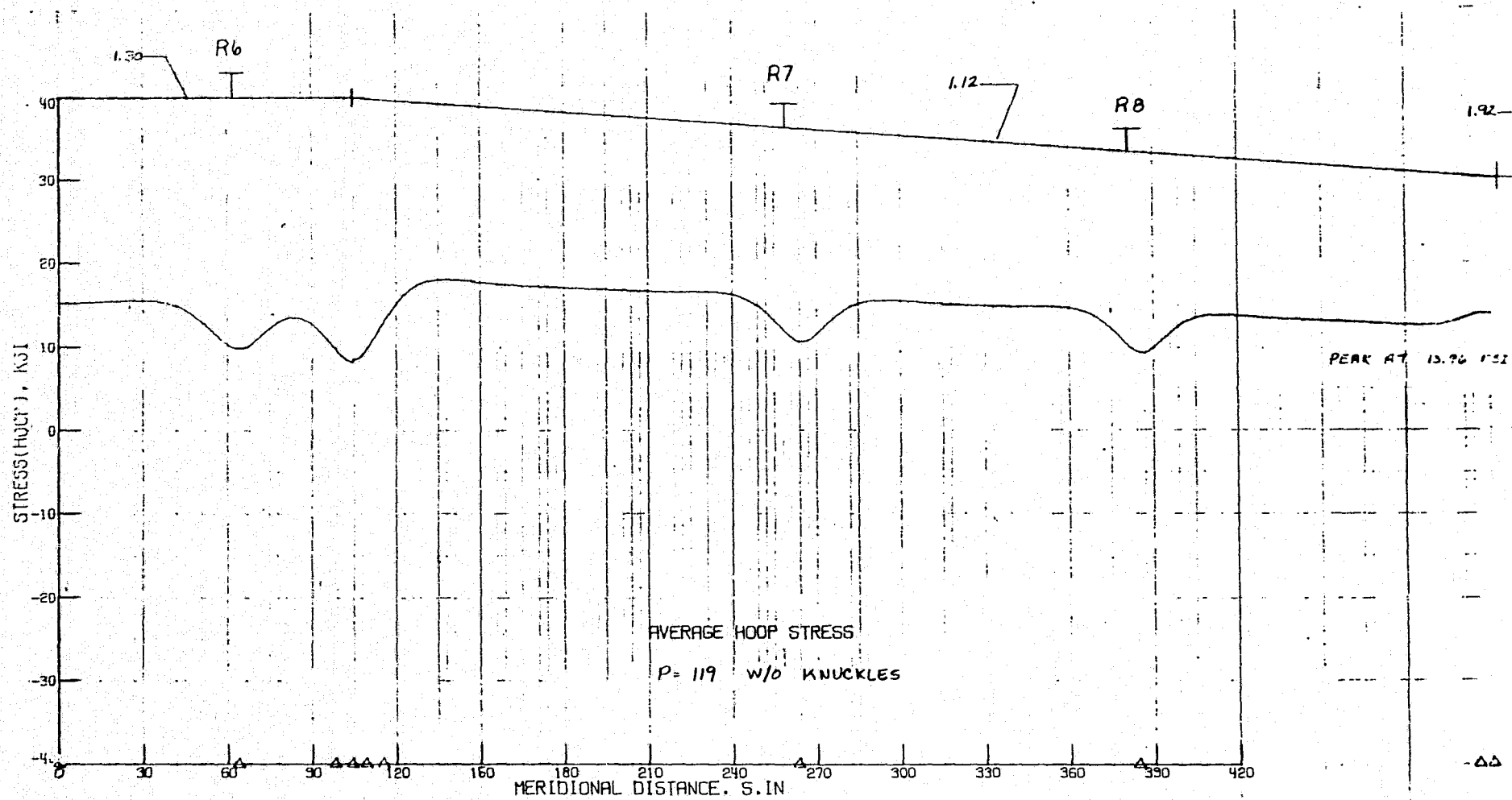


FIGURE 11



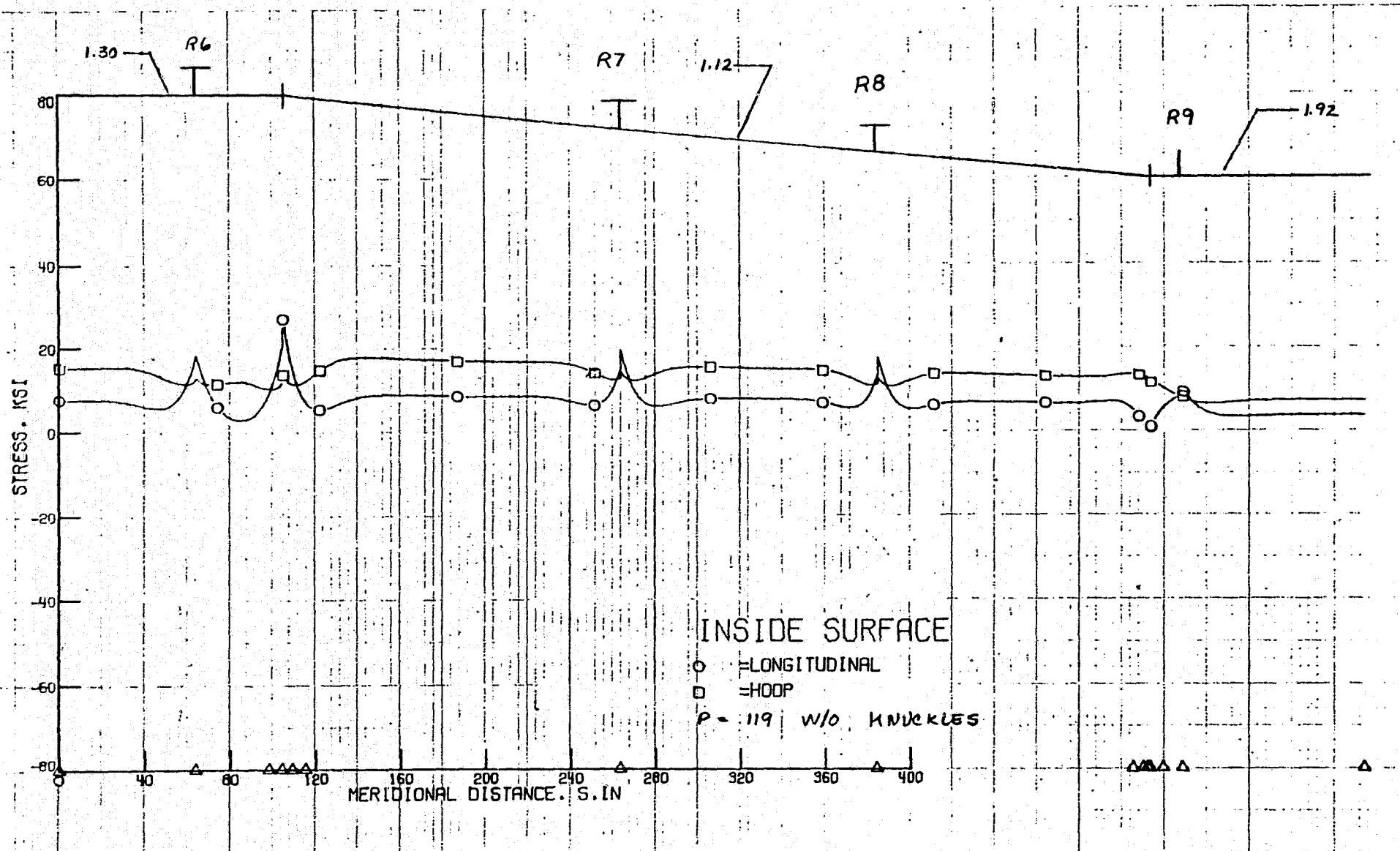


FIGURE 13

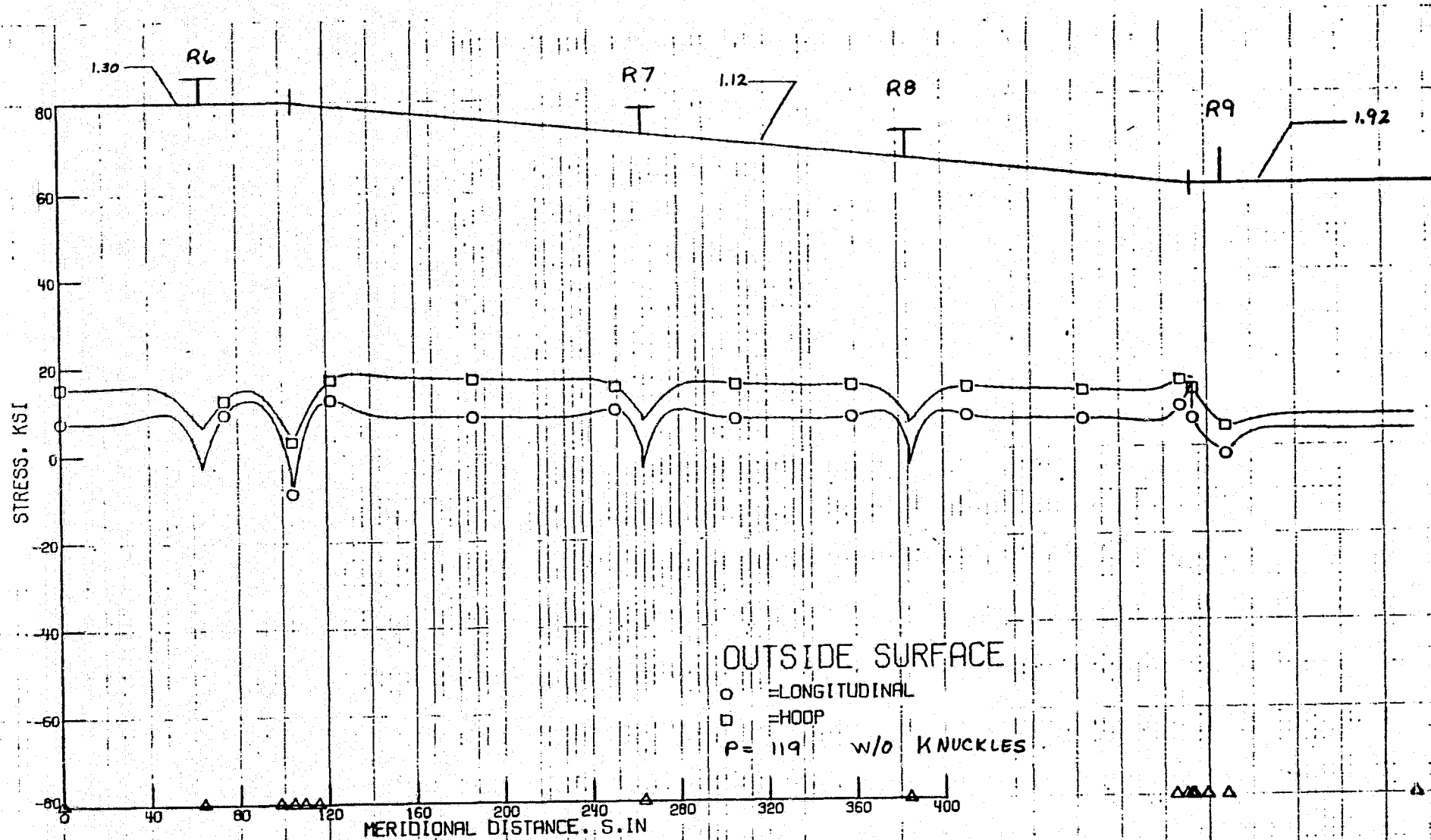


FIGURE 14

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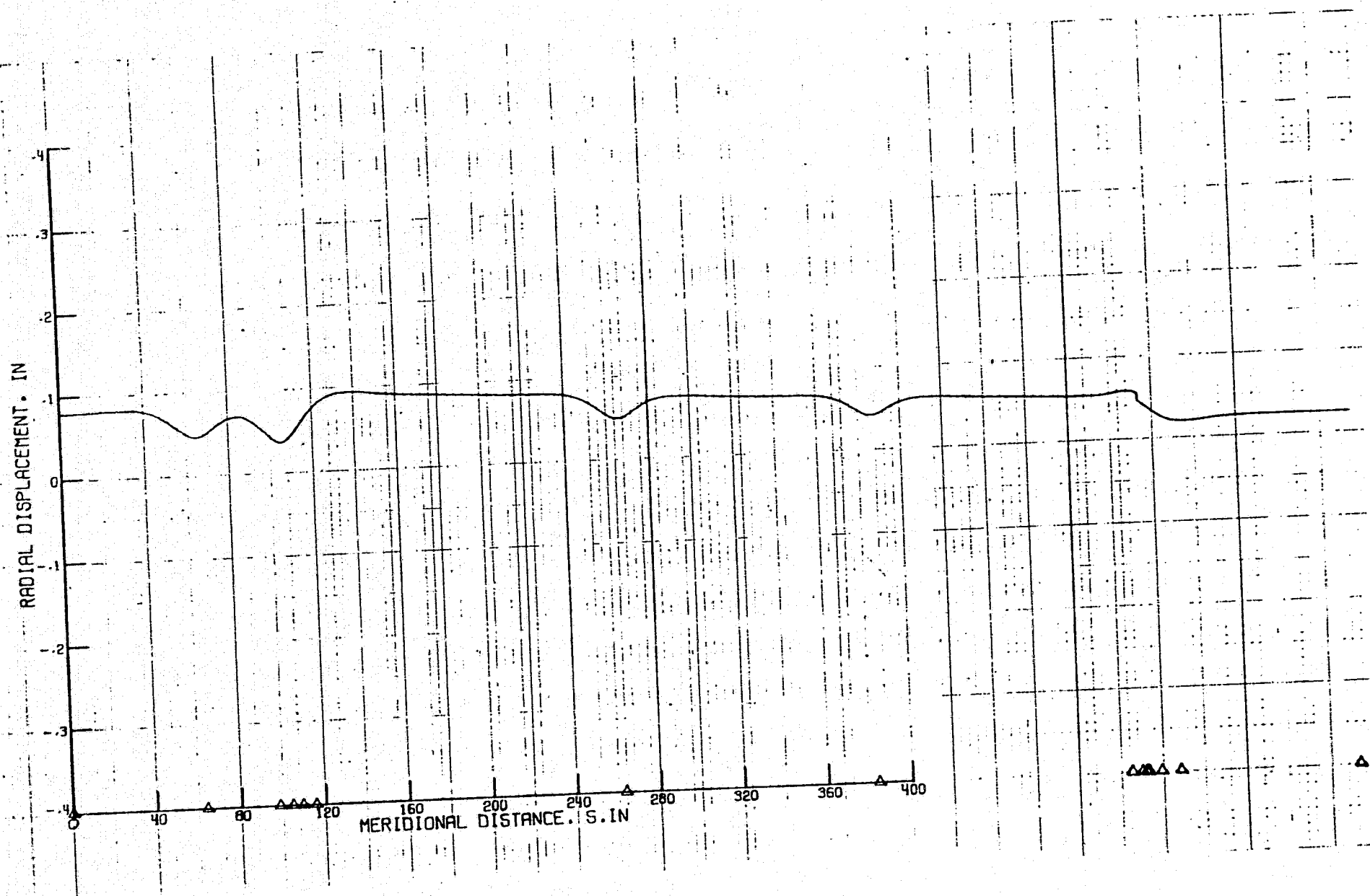


FIGURE 15

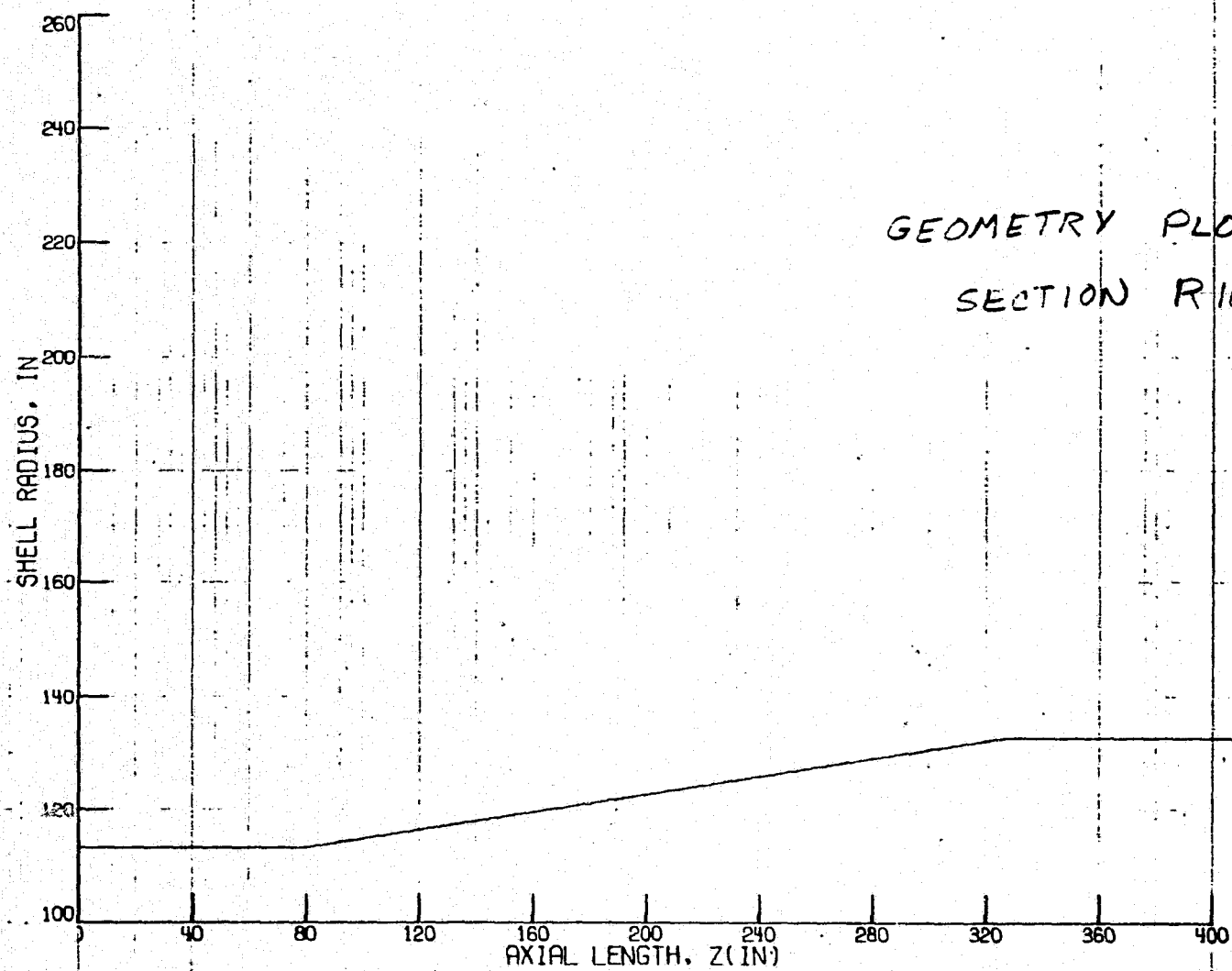


FIGURE 16

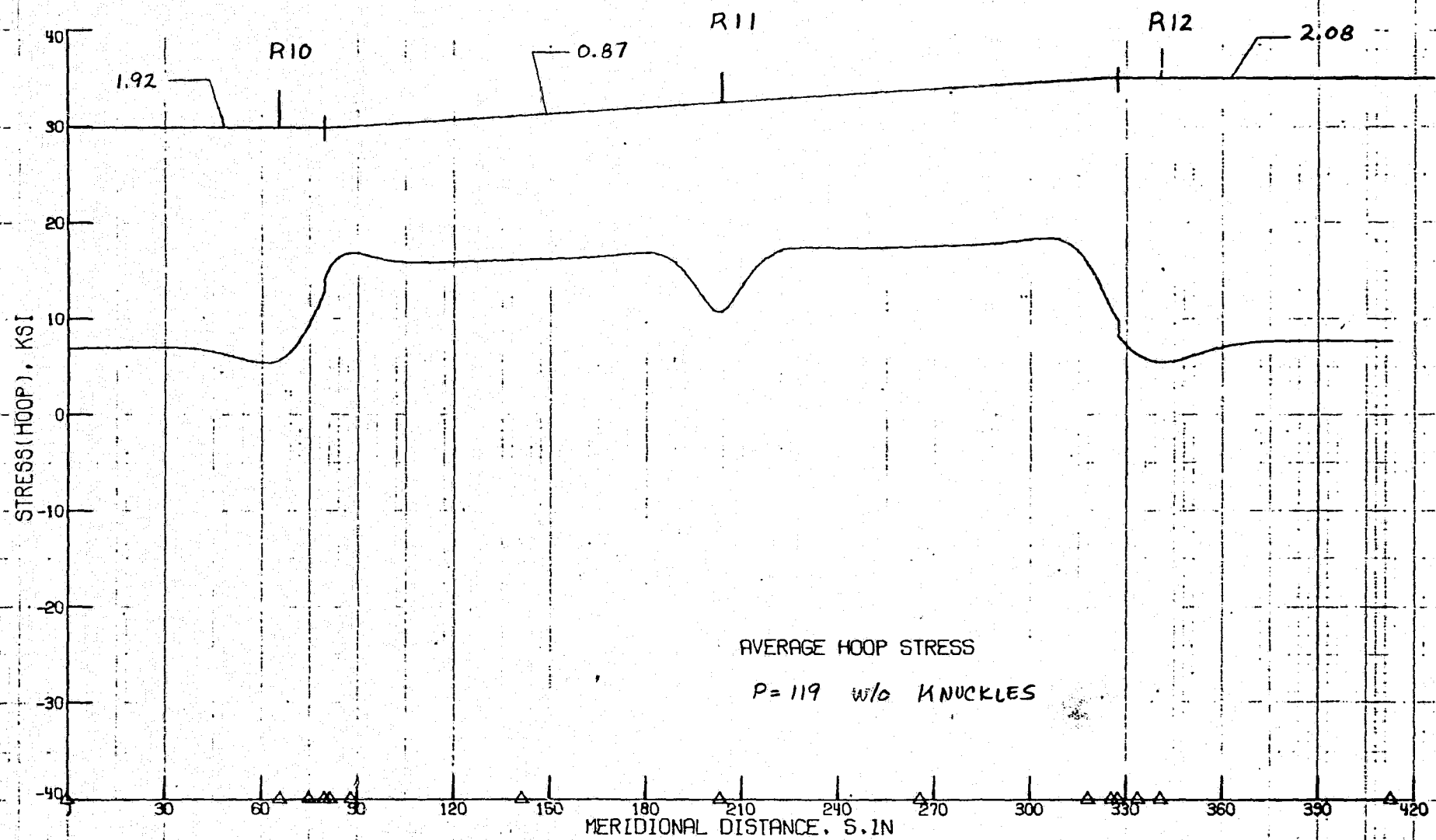


FIGURE 17

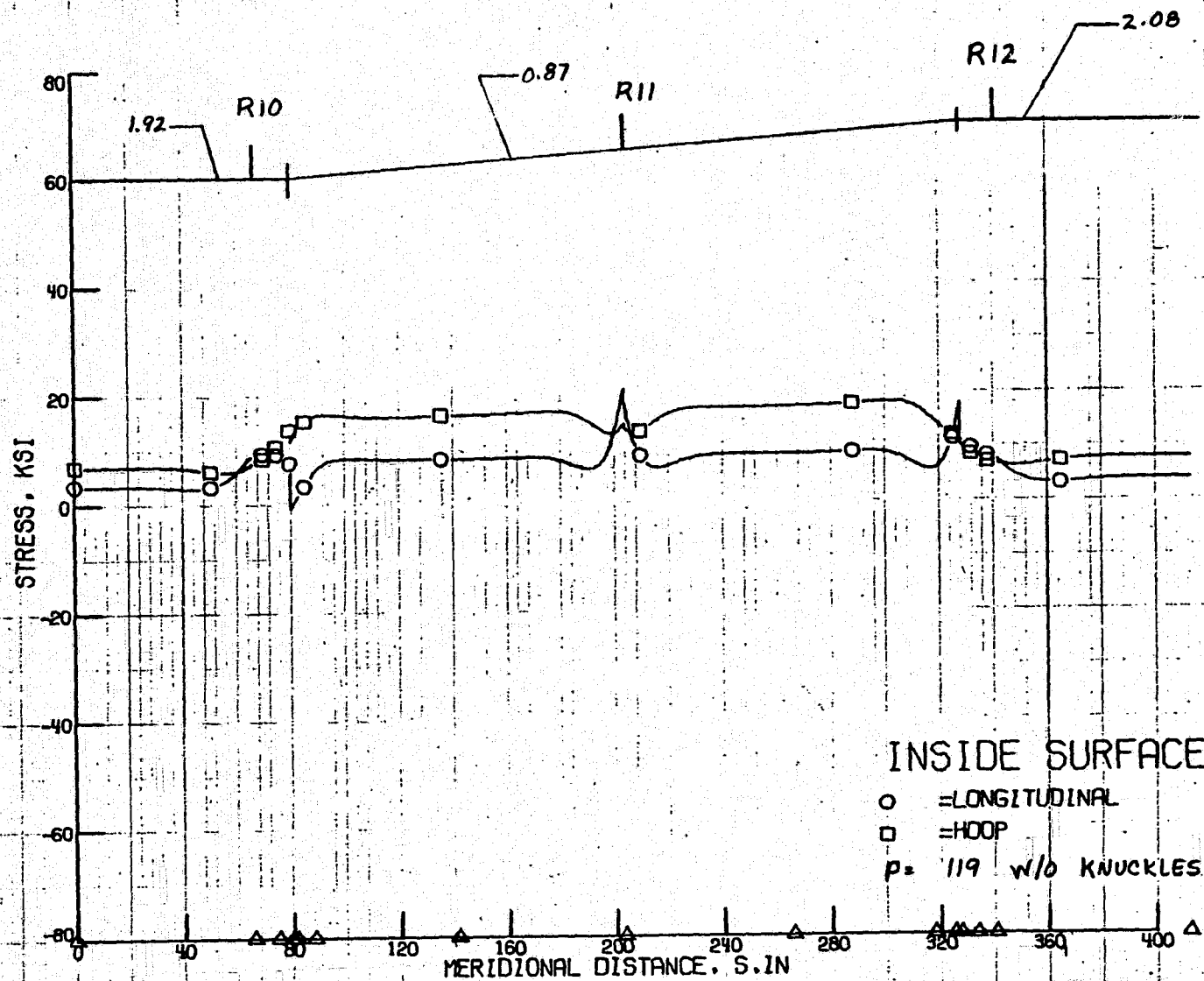


FIGURE 18

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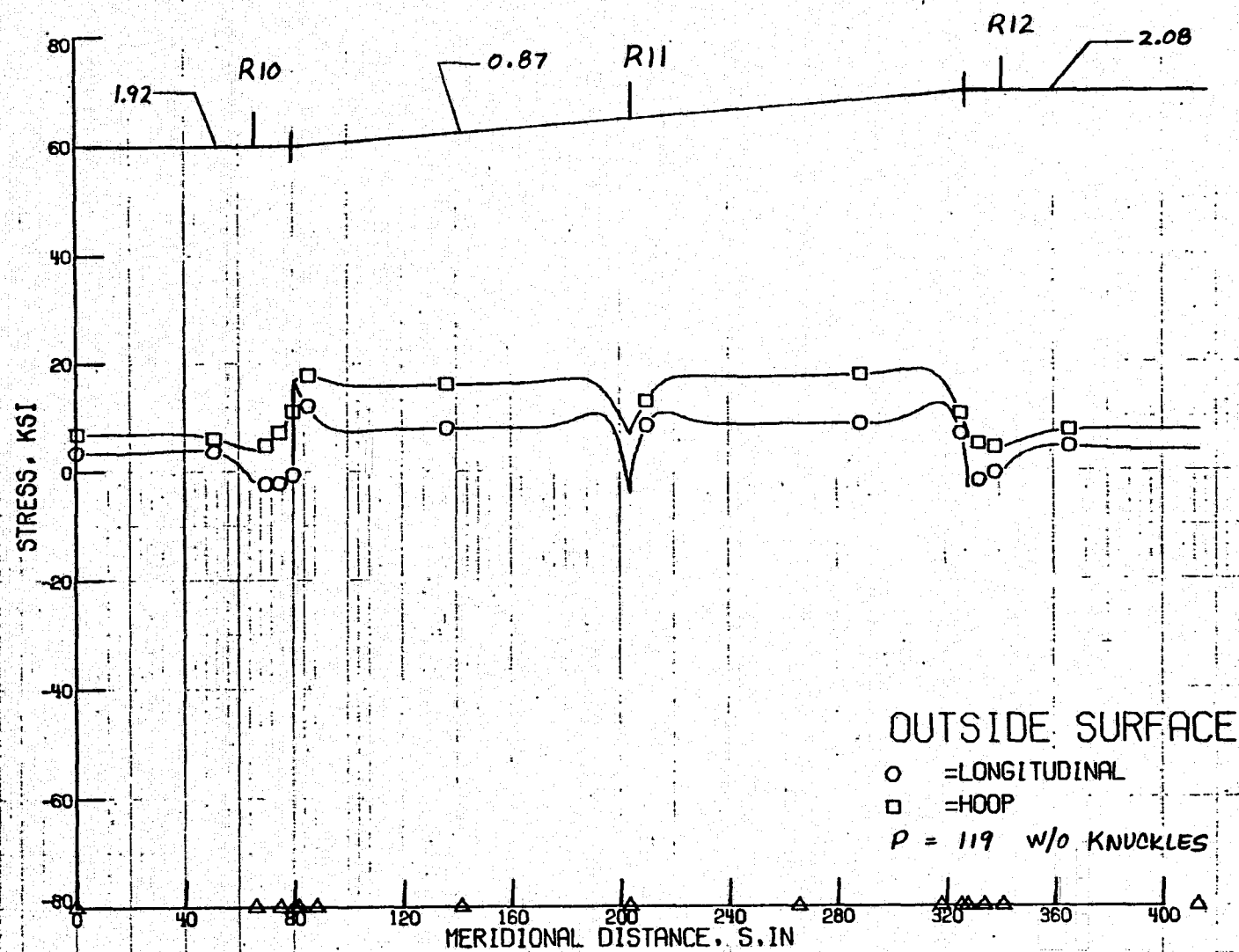


FIGURE 19

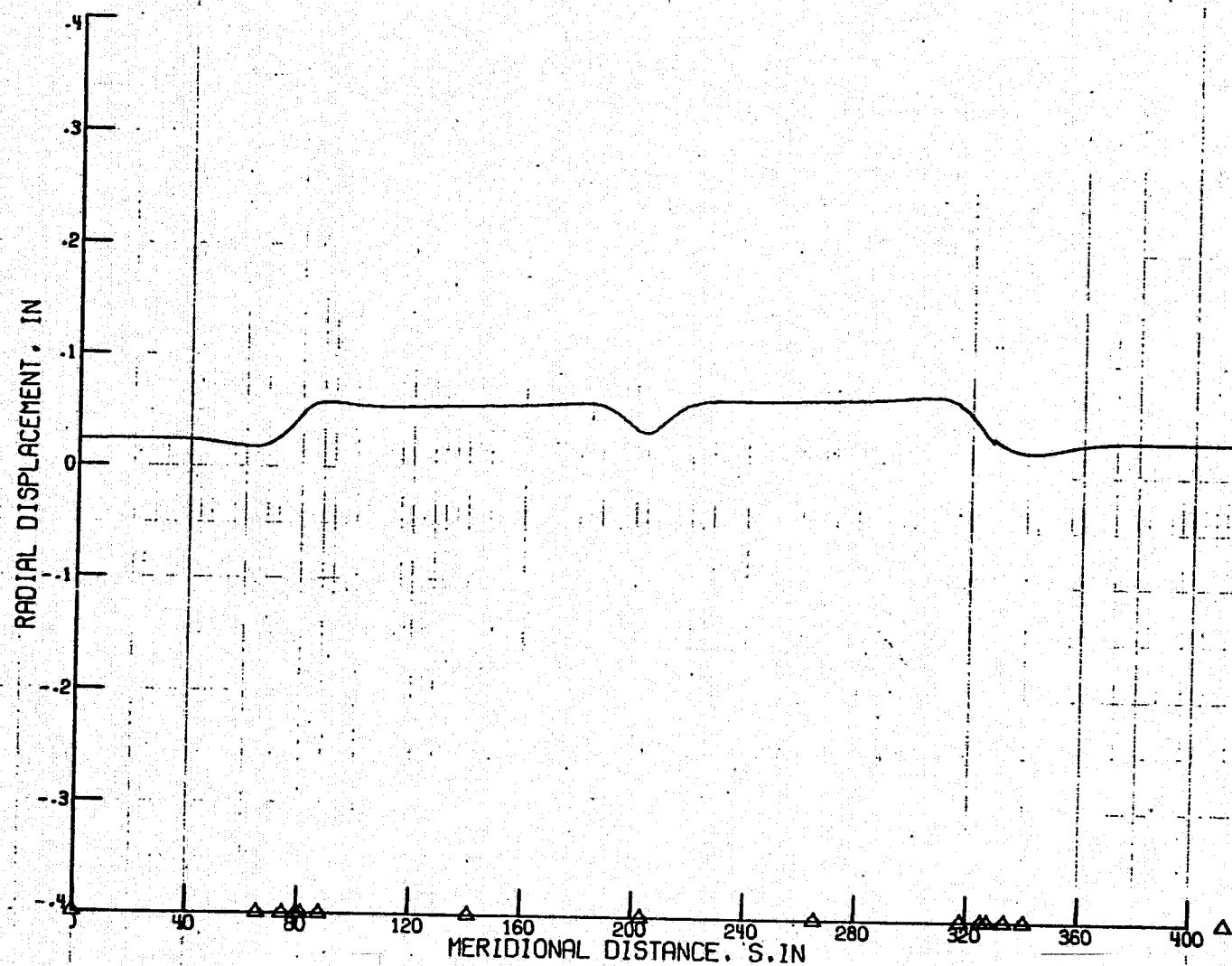


FIGURE 20

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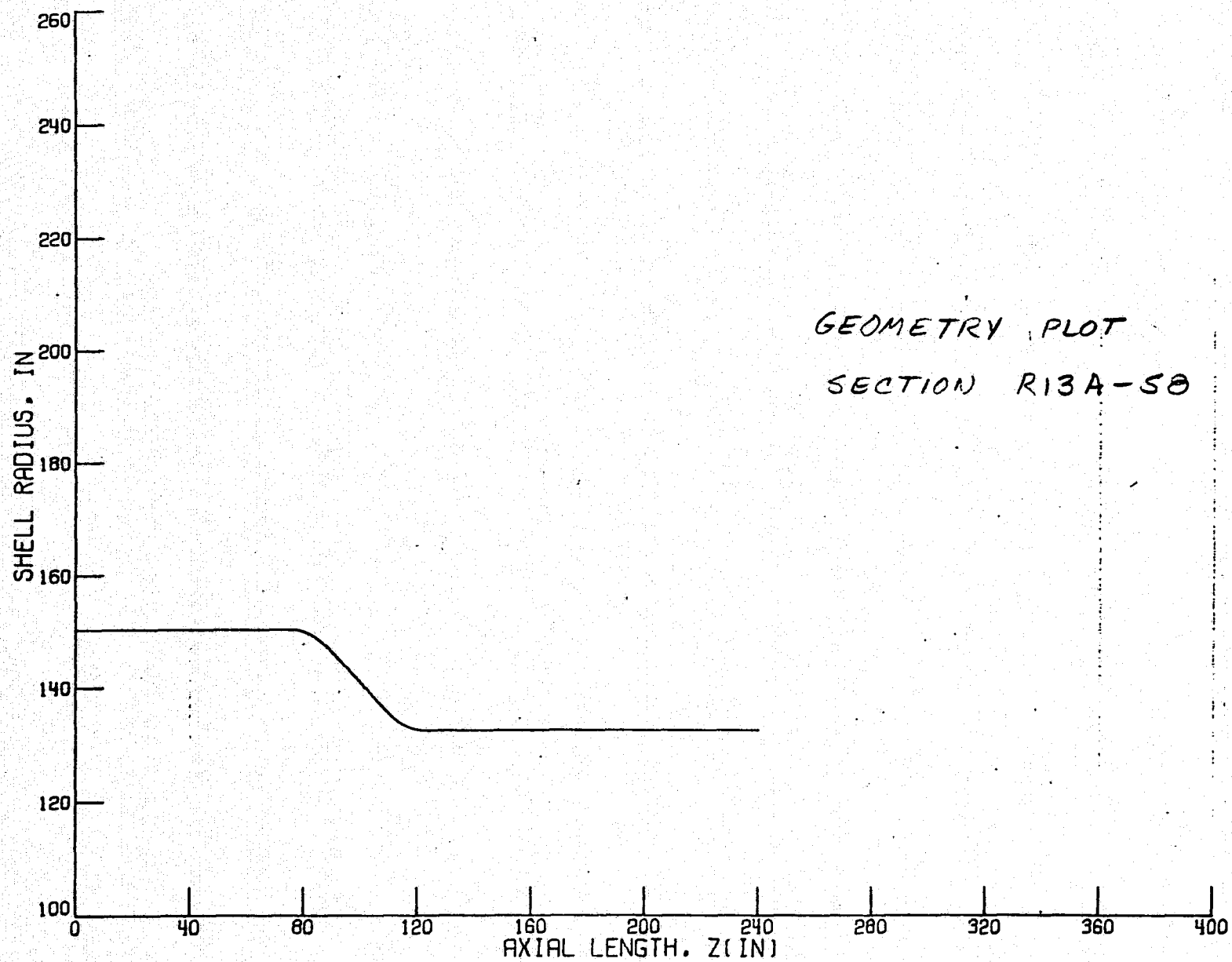


FIGURE 21

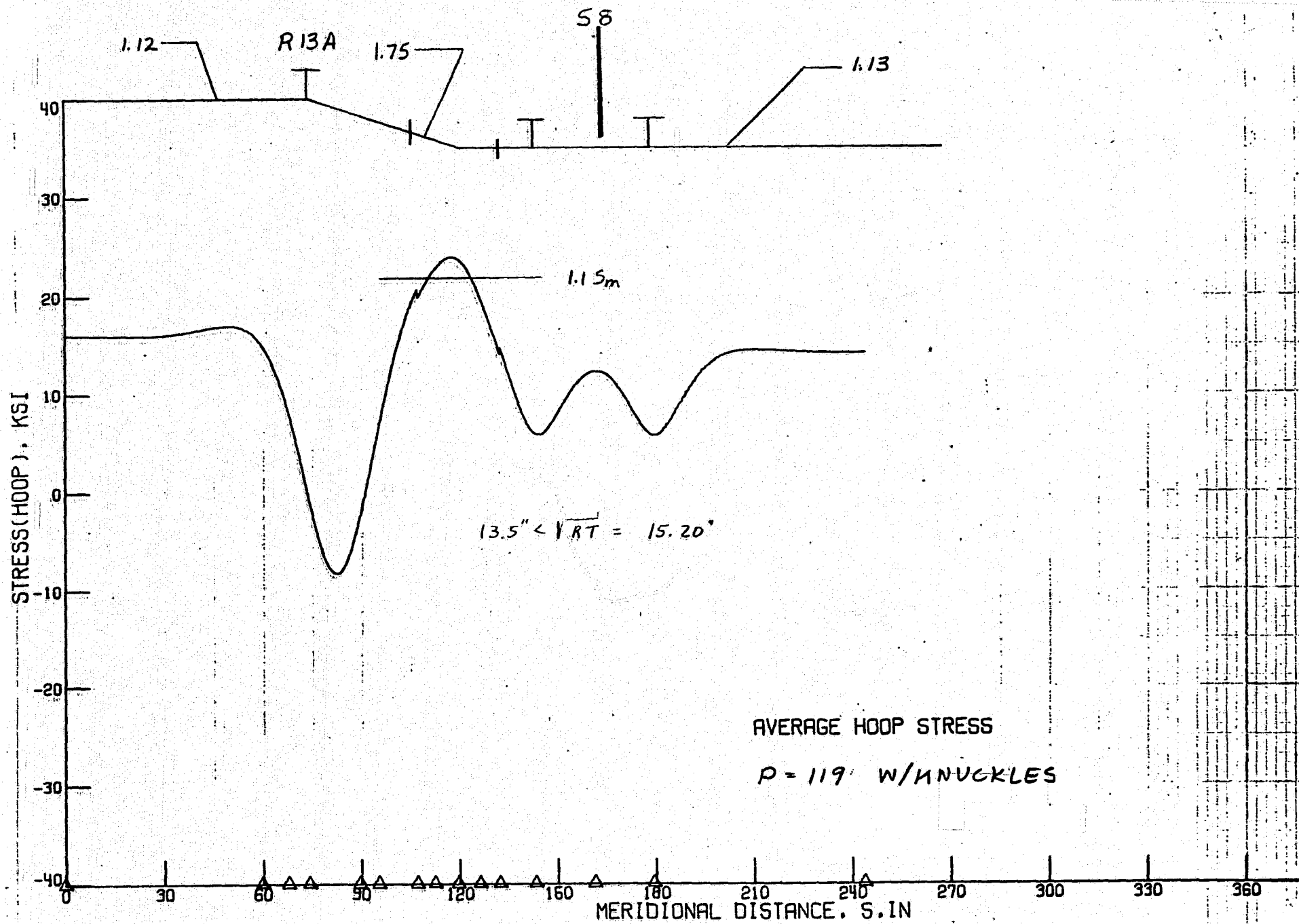


FIGURE 22

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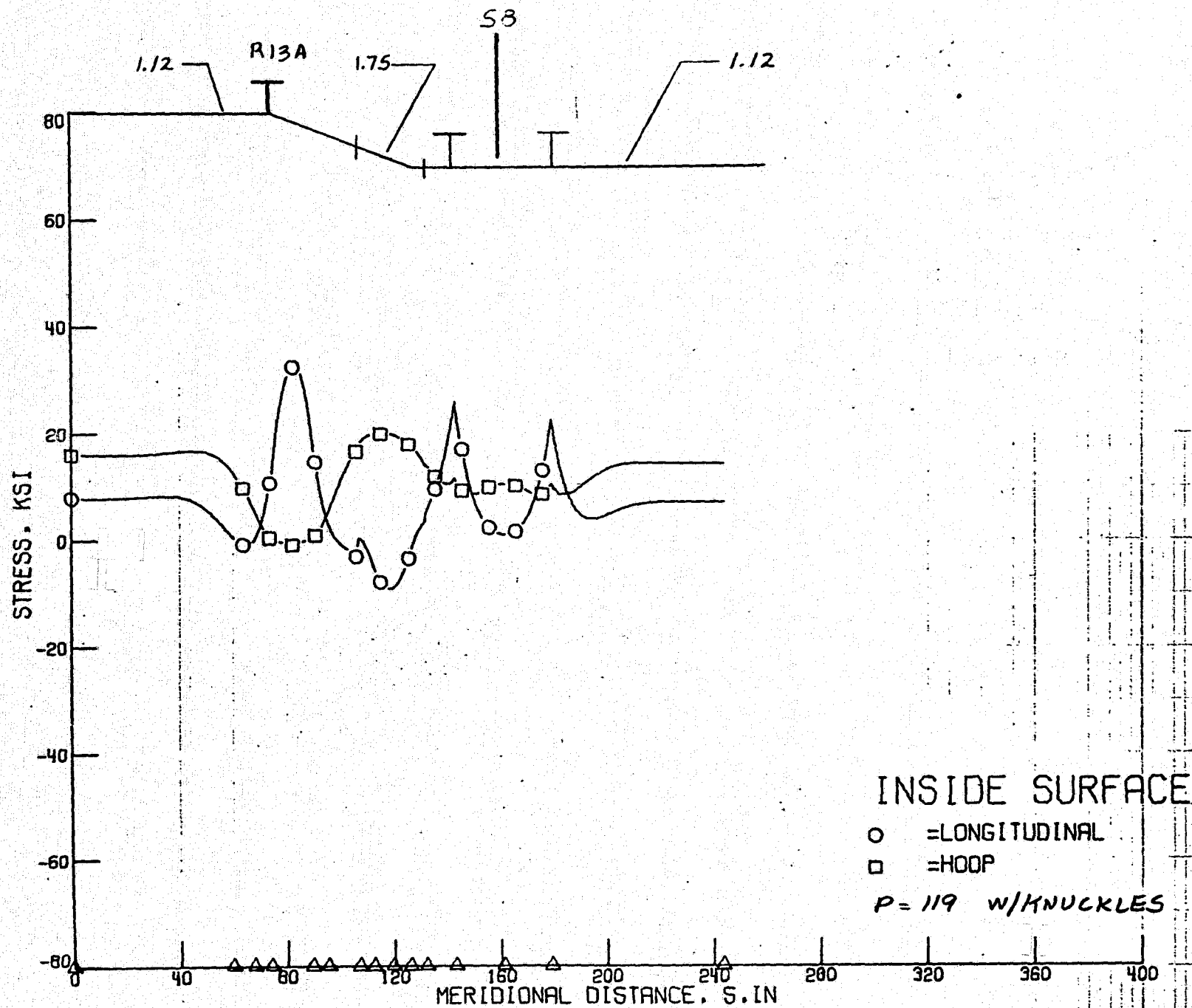


FIGURE 23

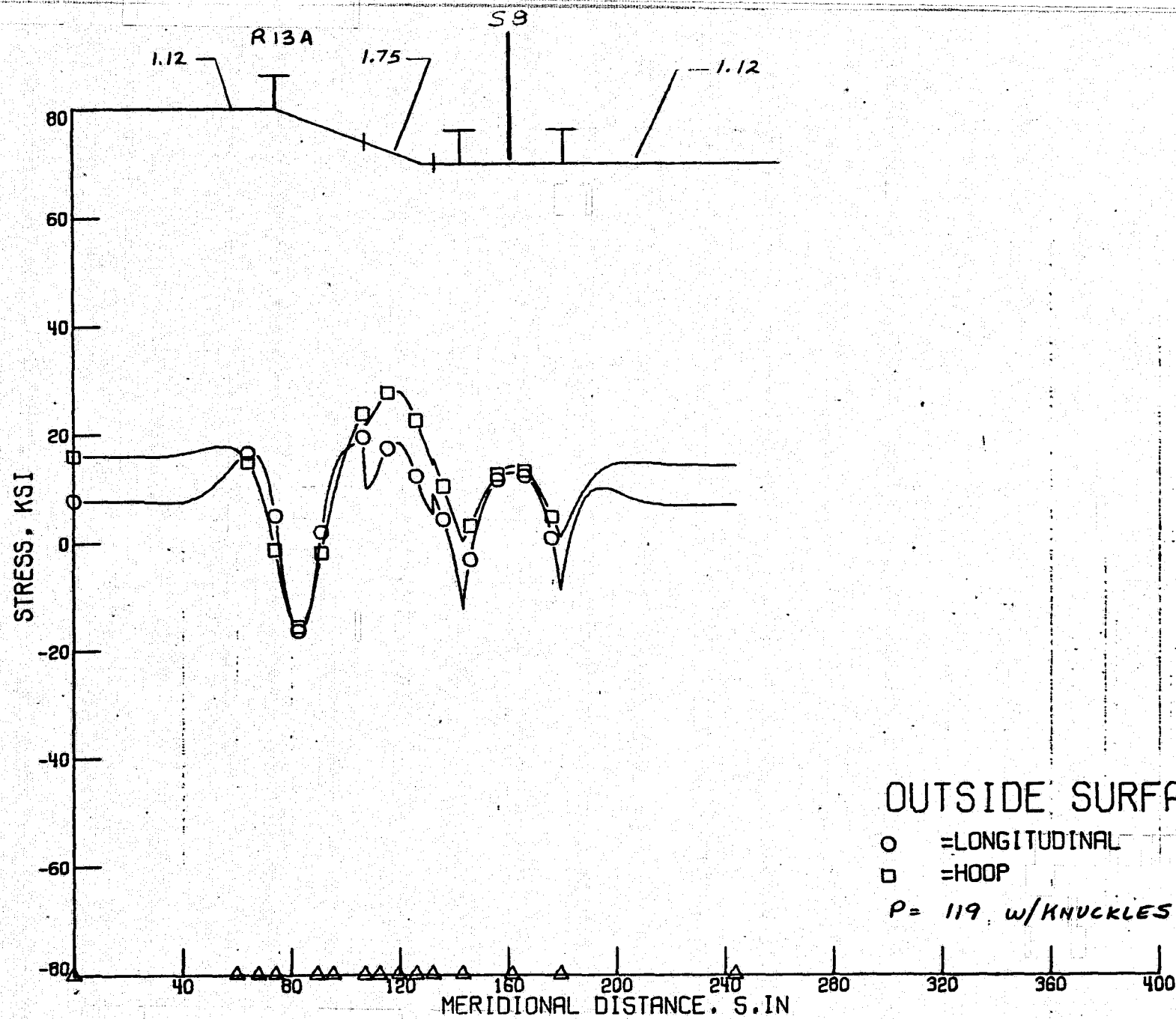


FIGURE 24

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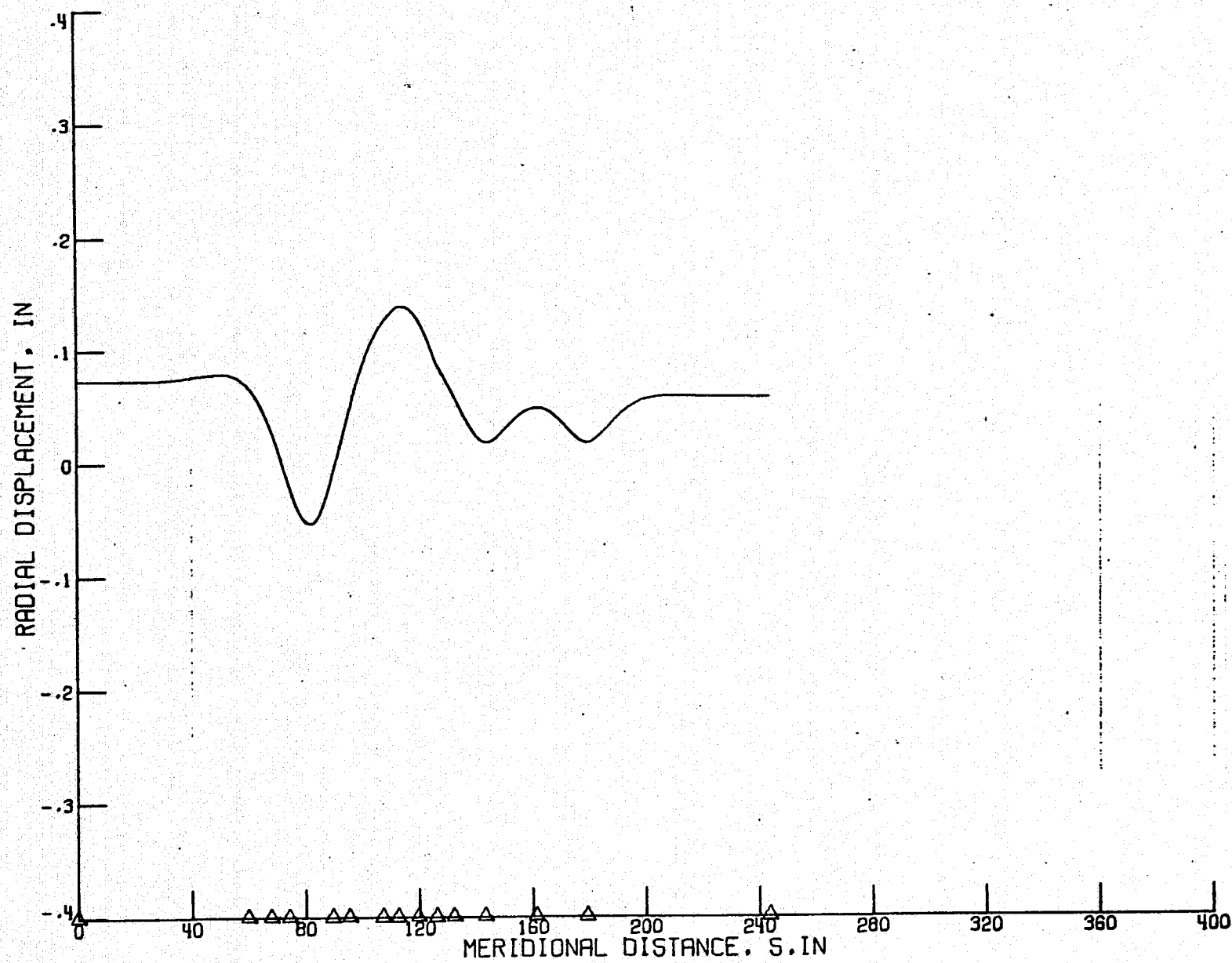


FIGURE 25

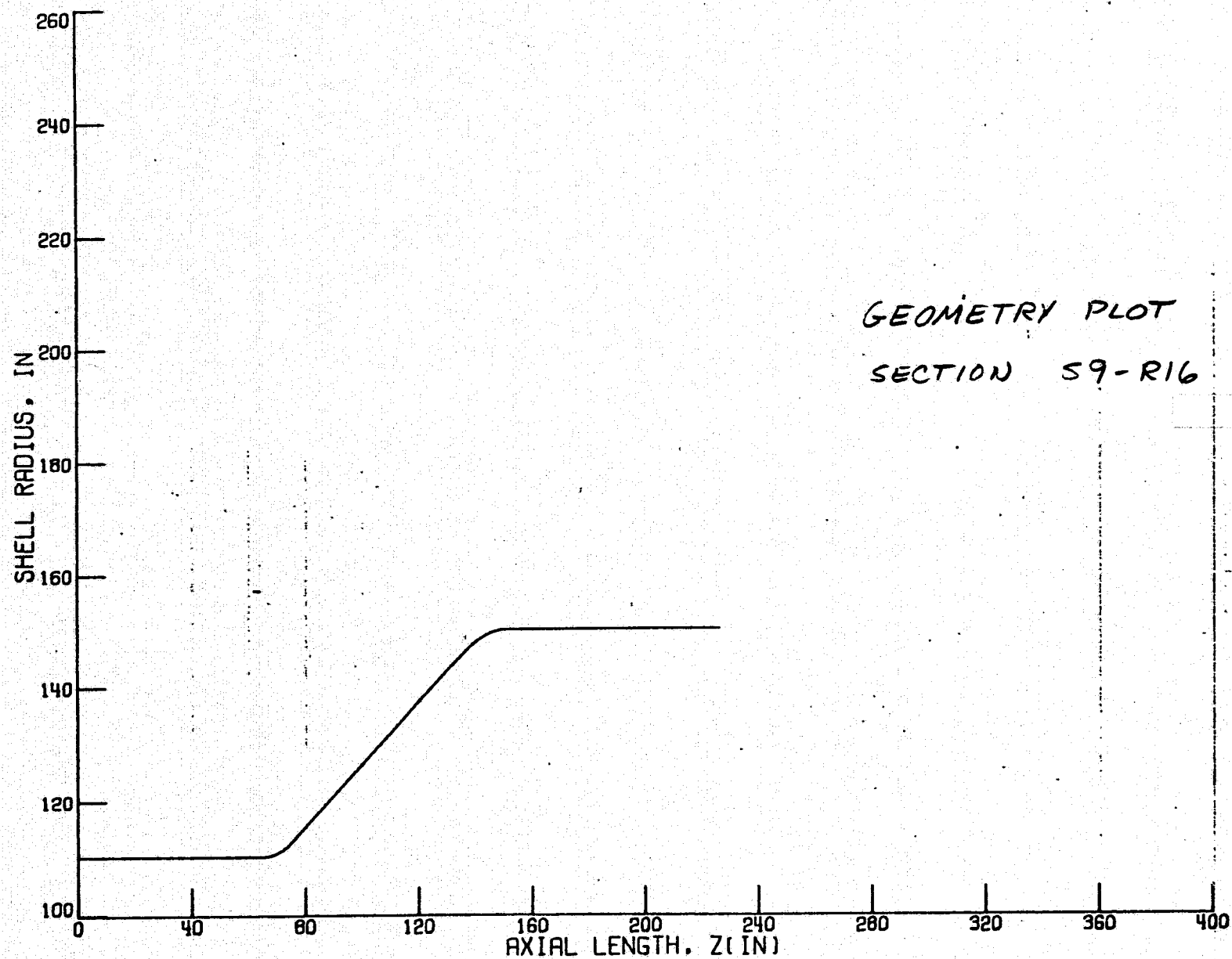


FIGURE 26

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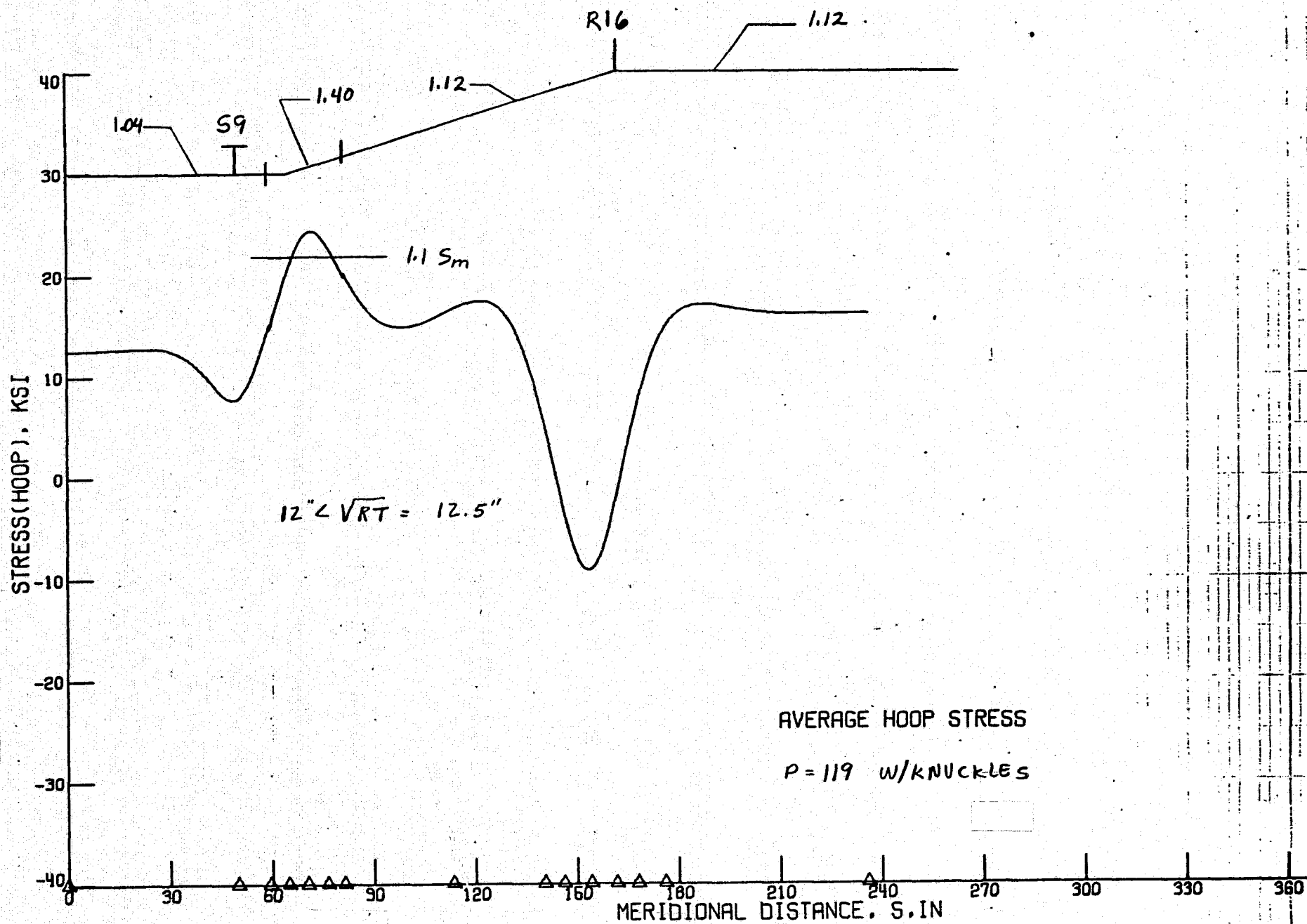


FIGURE 27

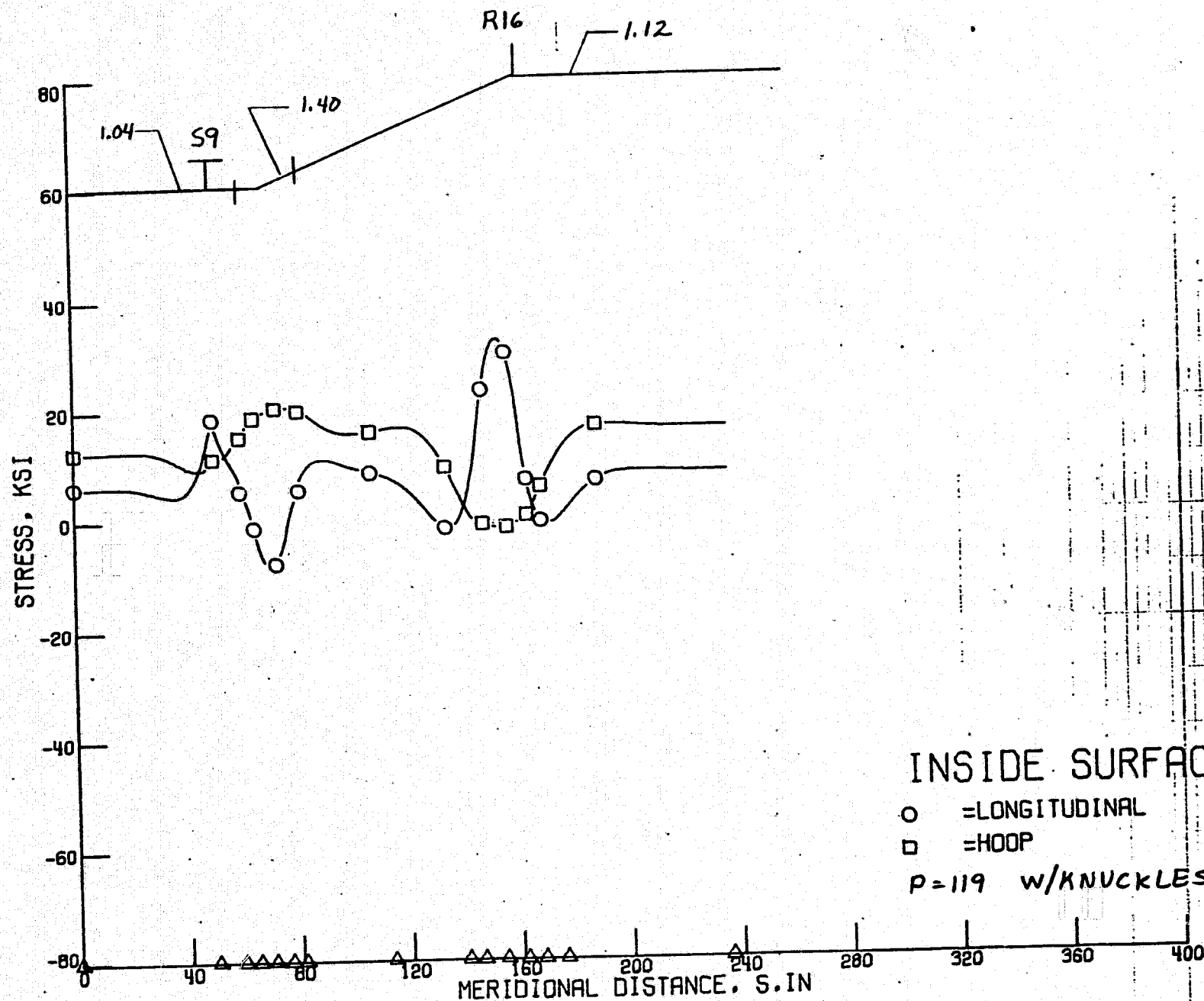


FIGURE 28

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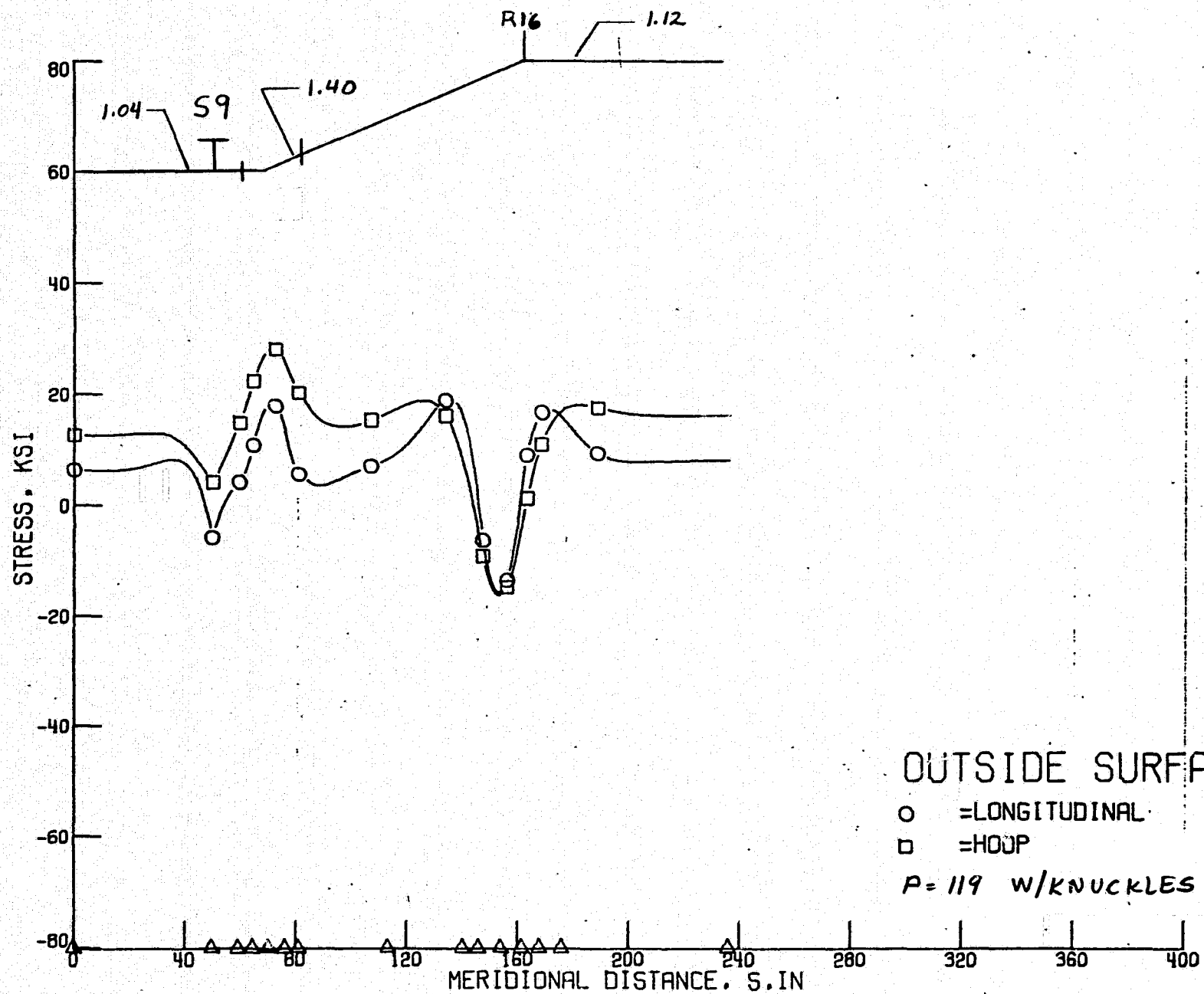


FIGURE 29

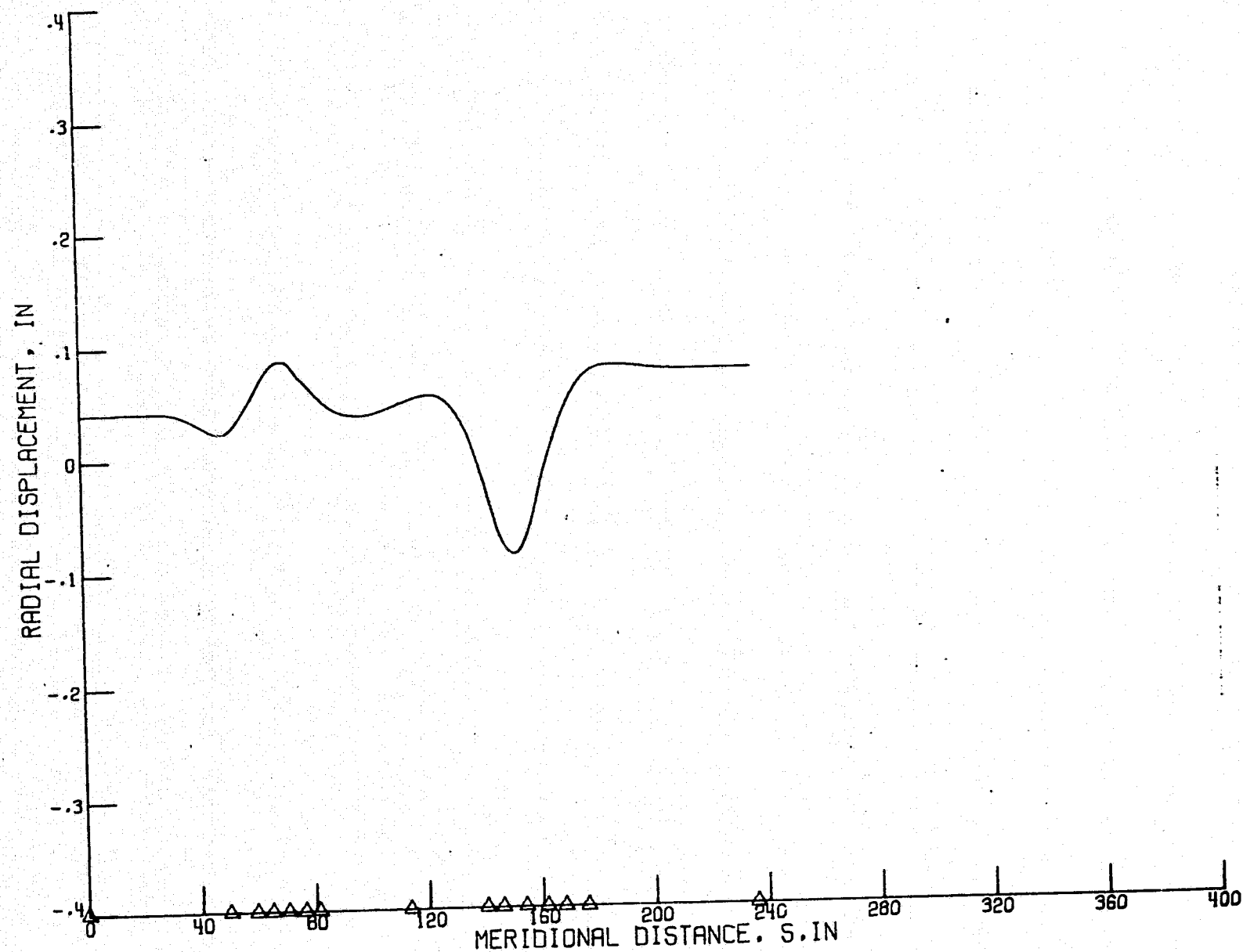


FIGURE 30

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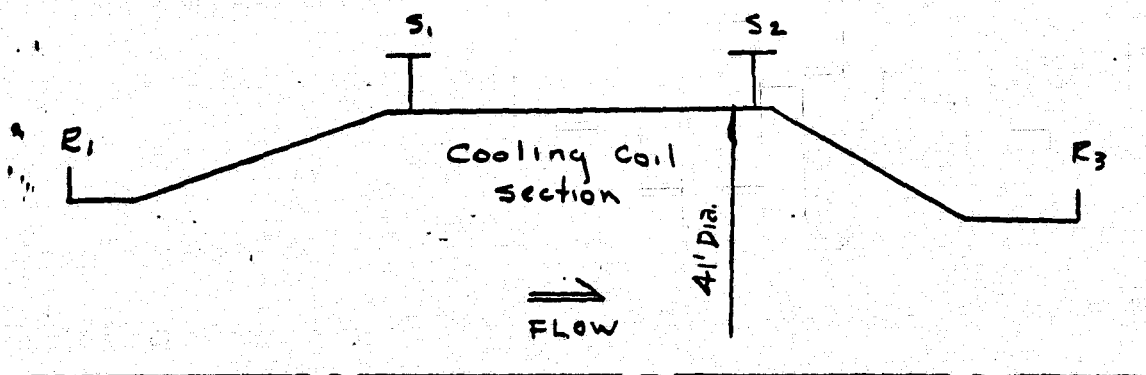
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SUBJECT NTE Pressure Shell SHEET NO. 1 OF _____
Stress Variation in shell
due to Hydro Conditions JOB NO. _____

304 S.S

Part 2

The following section of the tunnel was modeled using Nastran.



Note:

This is not a detailed analysis of this section of the Tunnel.

The following computer results and hand calculations are used only to verify stress scaling, due to Hydro Test conditions

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Nastran (Nasa Structural analysis) is
a general purpose digital computer program
for the analysis of large complex structures.

Nasa SP-222(01)

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Nastran model description

This section of the tunnel was modeled using homogenous quadrilateral membrane and bending elements. Except for R_1 & R_3 which were modeled using beam elements.

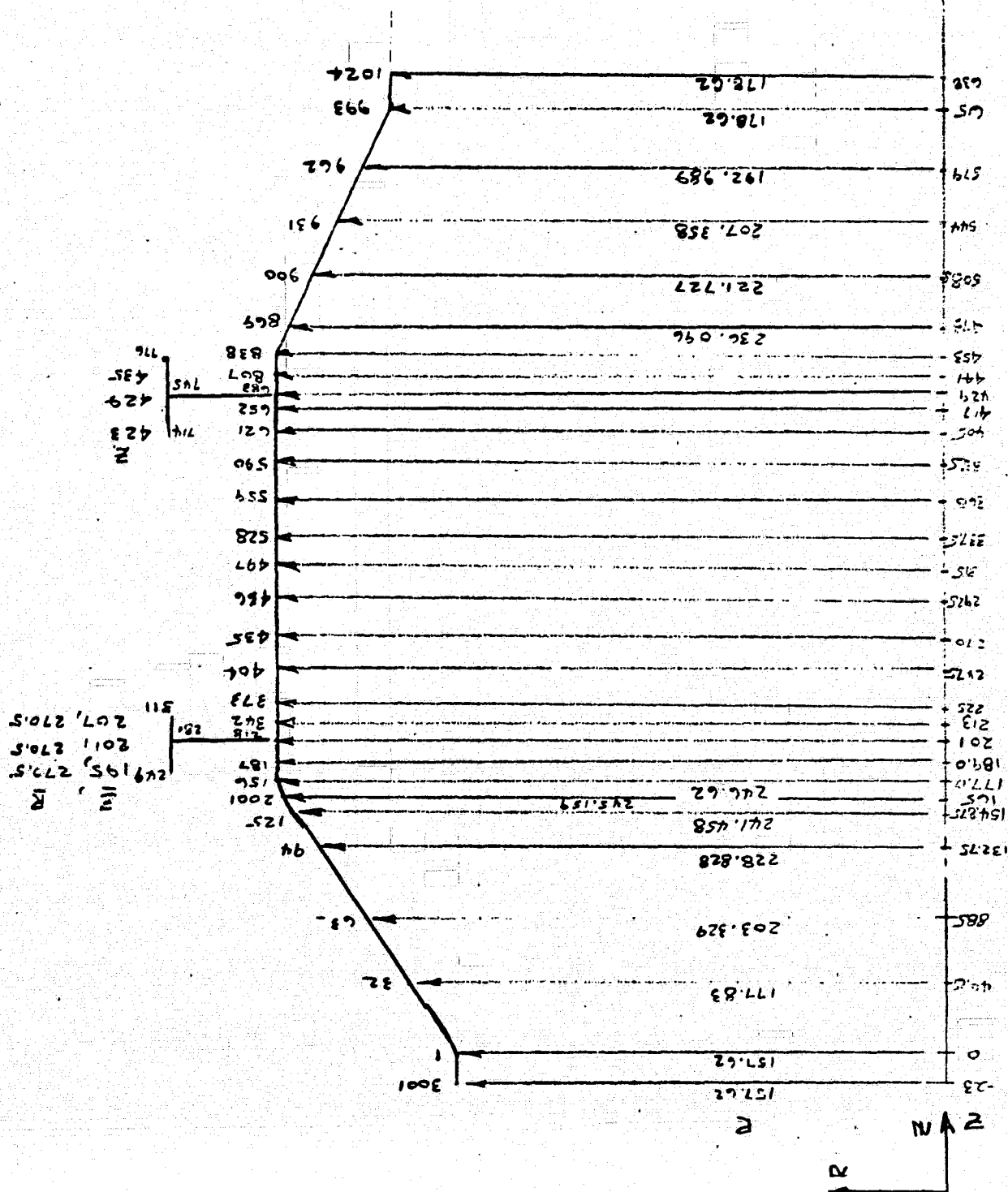
Due to the need of modeling a variable pressure, a half model using 31 elements around the circumference was generated.

See figure 1 for a joint location sketch of this area

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NASTRAN COARSE MODEL

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Constraints.

The R, Z plane was modeled as a plane of symmetry.

on the R_1 end of the model the Z displacement and rotations were removed.

on the R_3 end of the model all rotations were removed.

The rotation normal to the shell elements was removed

nodes on the flange of the support tee S2 and S3 located at $\theta = 90$ were fixed in the vertical direction. (fig 2)

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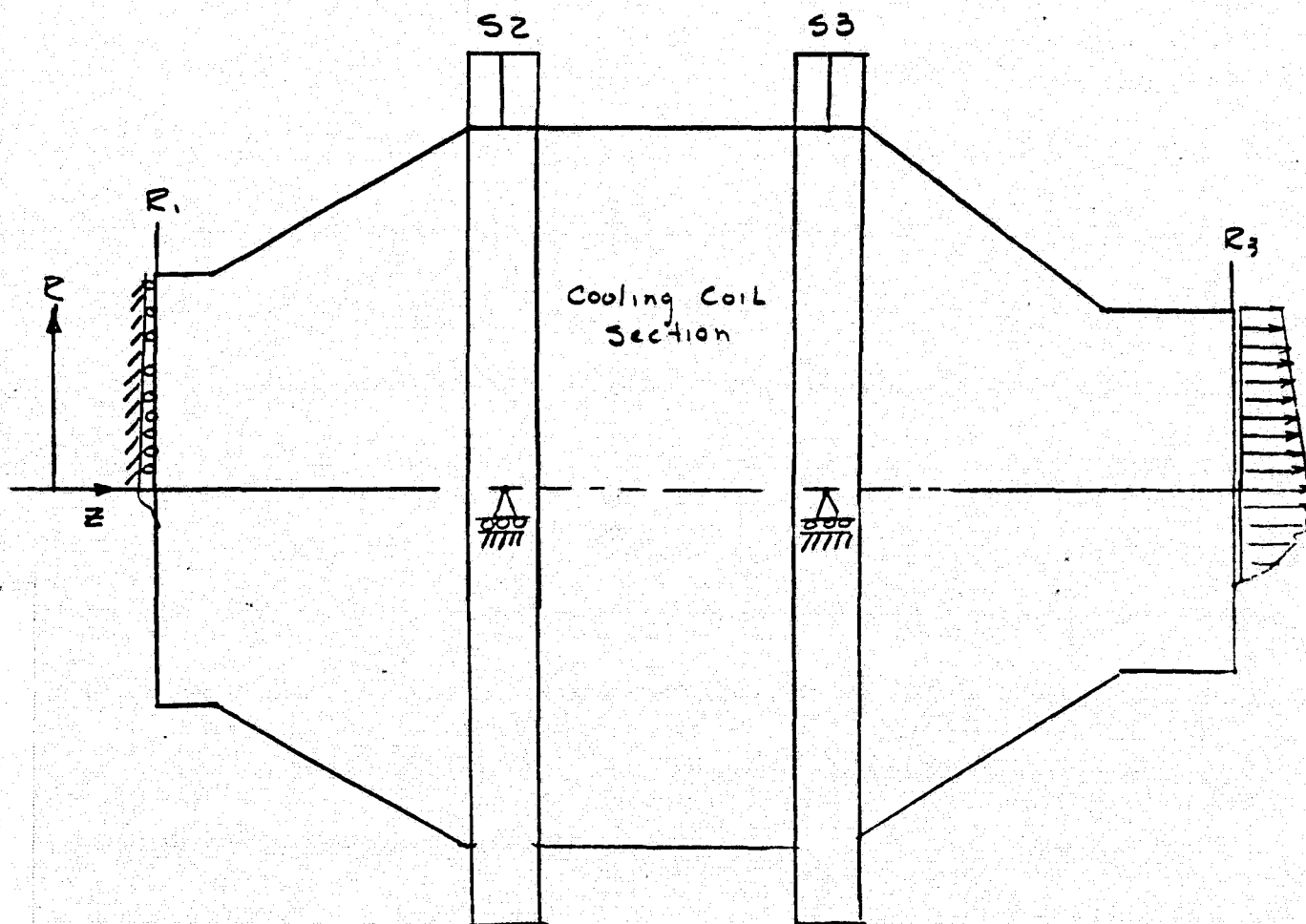
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JOB NO. _____

Boundary Conditions



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JOB NO. _____

Model geometry

as previously mentioned a half model with 31 elements around the circumference and 1117 nodes, was generated.

The model ran from ring location R1 to ring location R3.

all shell thickness were 1.24" except for the down stream shell past the second cone cylinder junction. this thickness was 1.00".

Material Constants

$$E = 30 \times 10^6 \text{ PSI}$$

$$\nu = 0.3$$

$$\rho = .283 \text{ lbs/in}^3$$

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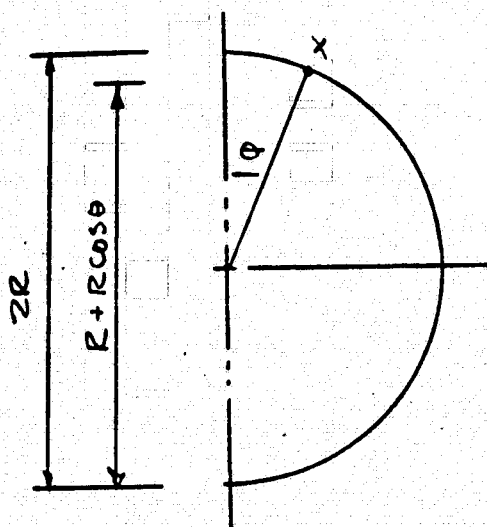
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Loading

a uniform element pressure of 119 psig was applied to all shell elements. In addition to this uniform pressure, a variable pressure due to the water head was added on.



$$\gamma = 62.4 \text{ lbs/ft}^3 = .0361 \frac{\text{lb}}{\text{in}^3}$$

$$P_{1x} = \gamma [2R - (R + R \cos \theta)]$$

$$P_{1x} = \gamma (R - R \cos \theta)$$

$$P_{\text{Test}} = 1.5 \times P_{\text{Opp}} = 1.5 \times 119 = 178.5$$

The variable pressure at any point x around the circumference was defined by $P_{1x} = \gamma (R - R \cos \theta) + P_{\text{Test}}$

Note: all pressures are in psi.

With 31 element around the circumference
 The enclosure angle between elements
 $180^\circ / 30 \text{ spaces} = 6^\circ$

Theta to the first element is 3° .

Pressure at $\theta = 3^\circ$

$$P_a = 119 \times 1.5 + .0361(20.5 \times 12 - 20.5 \times 12 \times \cos(3^\circ))$$

$$= 178.55 \text{ PSI}$$

Pressure at $\theta = 180^\circ$

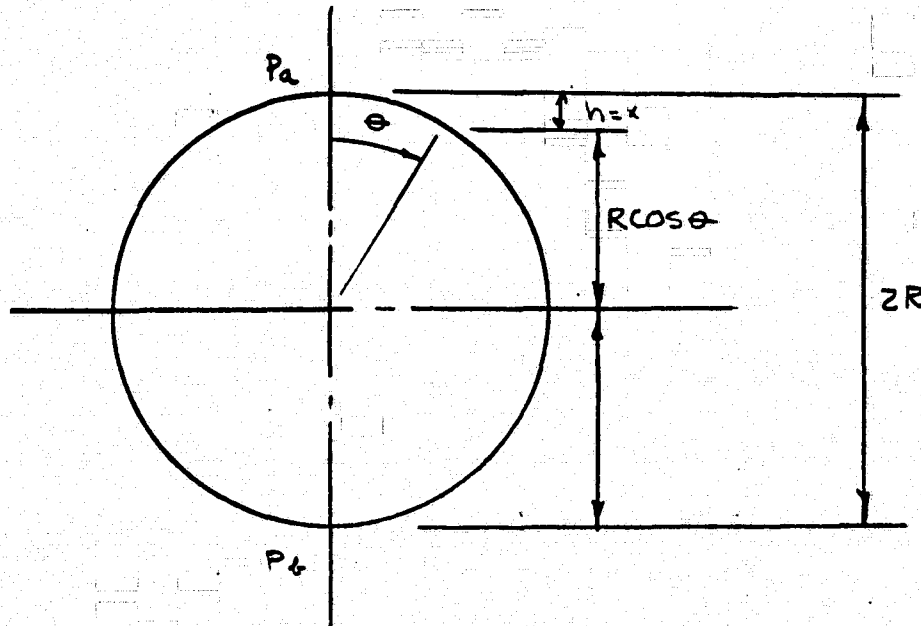
$$P_b = 119 \times 1.5 + .0361(20.5 \times 12 - 20.5 \times 12 \times \cos(180^\circ))$$

$$P_b = 196.23 \text{ PSI}$$

Where $(119 \times 1.5) = 178.5 \text{ PSI}$ is the Hydro test pressure.

$WR(1 - \cos \theta)$ is the additional pressure at any point due to the water head.

a linear variable end force was applied to the R_3 end of the model given by



$$\begin{aligned} h &= 2R - [R + R \cos \theta] \\ 2R - R - R \cos \theta \\ R[2 - 1 - \cos \theta] \\ R[1 - \cos \theta] \end{aligned}$$

$$P = \left(\frac{P_b - P_a}{2R} \right) x + P_a$$

$$P = \frac{\Delta P}{2R} [R(1 - \cos \theta)] + P_a$$

$$\text{Pressure} \Big|_x = \frac{\Delta P}{2} [1 - \cos \theta] + P_a$$

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Results

The primary purpose for this model was to verify that scaling of operating stresses to hydro stresses wouldn't generate any incorrect stresses results.

also, note that this section of the tunnel has the highest and lowest elevation. Therefore, The highest pressure due to the water head.

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JOB NO.

Water weights - From computer run GT 78007

Node	Constraint Reaction lbs
264	-5.59934 $\times 10^4$
295	-4.44728 $\times 10^5$
362	-5.115345 $\times 10^4$
729	-5.441023 $\times 10^4$
760	-4.59041 $\times 10^5$
791	-5.556915 $\times 10^4$
3016	-4.098632 $\times 10^5$
1039	-3.277161 $\times 10^5$
TOTAL	1858475. lbs

Calculated

Weight of water in this section of
the tunnel. 1851719.0 lbs.

Calculated	1851719.0	lbs
Model generated	1858475.0	lbs
Δ Water weight	-6756.0	lbs.
	.36 %	diff.

operating pressure $P = 119.00$

(top) Hydro pressure at element 2001 = $1.0483 + 178.5 = 179.55 \text{ psi}$
(bottom) Hydro pressure at element 2030 = $17.73 + 178.5 = 196.23 \text{ psi}$

(bottom) Hydro pressure at element 2030 = $17.73 + 178.5 = 196.23 \text{ psi}$

Scale factor for top of tunnel:

$$\frac{178.55}{119} = 1.5004$$

Scale factor for bottom of tunnel:

$$\frac{196.23}{119} = 1.649$$

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From Nastran run GT 78007 w/ P=119
(Peak stresses)

element	$\sigma_{\text{Hoop KSI}}$ inside	$\sigma_{\text{Hoop KSI}}$ outside	$\sigma_{\text{axial KSI}}$ inside	$\sigma_{\text{axial KSI}}$ outside	Location
2001	-4.43	-13.9	+27.59	-3.71	$\theta = 0^\circ$
2030	-4.44	-13.9	+27.59	-3.71	$\theta = 180^\circ$

Scaled Stresses from above

element	σ_{hoop} inside	σ_{hoop} outside	σ_{axial} inside	σ_{axial} outside	scale factor
2001	-6.647	-20.86	+41.396	-5.57	1.5
2030	-7.32	-22.92	+45.50	-6.12	1.649

Stresses from Hydro run $P = 178.5 + H_2O$ head

element	σ_{Hoop} KSI inside	σ_{Hoop} KSI outside	σ_{axial} KSI inside	σ_{axial} KSI outside	location
2001	-7.14	-21.6	42.0	-5.71	$\theta = 0^\circ$
2030	-7.15	-22.6	45.23	-6.03	$\theta = 180^\circ$

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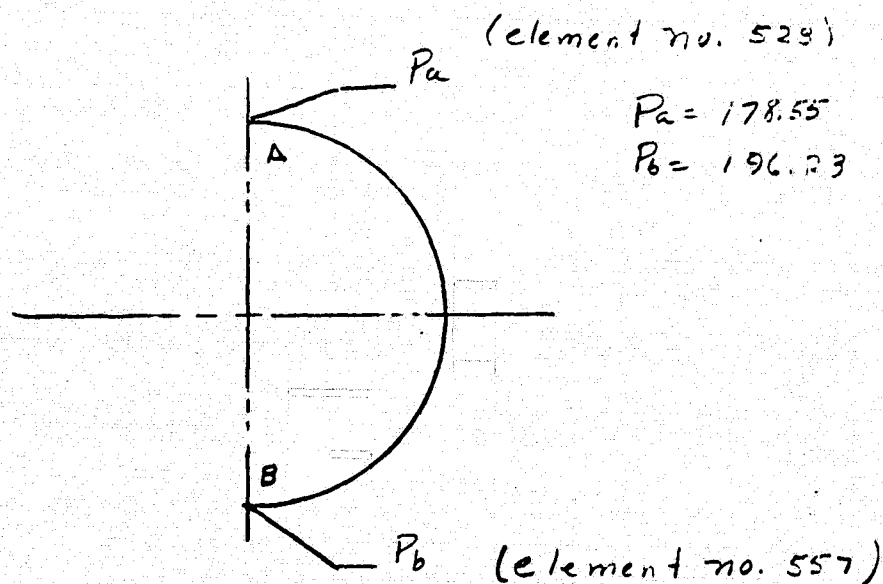
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As can be seen from the last two tables, scaling of operating stresses does not generate any erroneous stresses.

To add additional verification to the above procedure some hand calculations to predict stresses in the region between support ring were made.



For Hydro conditions.

At point A. element no. 528

$$R = 246.62 \quad \theta = 0^\circ \quad P_r = 178.55 \text{ PSI}$$

$$\sigma_{\text{Hoop}} = (178.55 \times 246.62) / 1.24 = 35511 \text{ PSI}$$

at point B element no. 557

$$R = 246.62 \quad \theta = 180^\circ \quad P_r = 196.23$$

$$\sigma_{\text{Hoop}} = (196.23 \times 246.62) / 1.24 = 39027 \text{ PSI}$$

From Nastran run no. 6T 78007 :

at element no. 528

$$\sigma_{\text{Hoop}} (\text{inside}) = 35509.9$$

$$\sigma_{\text{Hoop}} (\text{outside}) = 35405.8$$

$\sigma_{\text{ave}} = 35457 \text{ PSI}$
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at element no. 557

$$\sigma_{\text{Hoop}} (\text{inside}) = 38929$$

$$\sigma_{\text{Hoop}} (\text{outside}) = 39025$$

$$\sigma_{\text{ave}} = 38977 \text{ PSI}$$

element	Hand Calculation	Computer results	θ	
528	35.5 KSI	35.5 KSI	0°	
557	39.0 KSI	39.0 KSI	180°	

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In conclusion the following method was used in predicting hydro test stresses from operating conditions.

Upper center line

$$P_{\text{hydro}} = 119 \times 1.5 + \gamma (41 \times 12/2 + D_L \times 12/2)$$

$$\gamma = 0.0361 \text{ lbs/in}^3$$

D_L = Local Diameter.

Lower center line

$$P_{\text{hydro}} = 119 \times 1.5 + \gamma (41 \times 12/2 + D_L \times 12/2 + 9135)$$

Therefore,

$$\sigma'_{\text{(HYDRO)}} = \left\{ \sigma'_{\text{(operating)}} \right\} \left\{ \frac{P_{\text{hydro}}}{119.0} \right\}$$

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SUBJECT NTF PRESSURE SHELL
Effects of Insulation Rings
on shell stresses

SHEET NO. 1 OF.....
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Part 3
304. S.S.

Assume a cylinder with no structural
support ring and design in accordance
with Div I of Code

$$t = \frac{PR}{SE - 0.6P}$$

$$P = 119 \text{ psi}$$

$$R = 120"$$

$$S = 25.0 \text{ ksi}$$

$$E = 1$$

$$t = \frac{(119 \times 120)}{(25000 \times 1) - 0.6(119)}$$

$$t = .573 \text{ in.}$$

$$\sigma = \frac{Pr}{t} = \frac{(119 \times 120 \times .286)}{.573} = 24,980 \text{ psi}$$

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SHEET NO. 2 OF

JOB NO......

A hand-drawn diagram of a horizontal bar. The total length of the bar is indicated by a dimension line at the top, labeled $196''$. The bar is supported from below by a vertical line on the left, which is labeled $R = 120.7865$. A downward arrow points to the center of the bar, and a vertical dimension line from the center of the bar to the bottom line is labeled $.573$.

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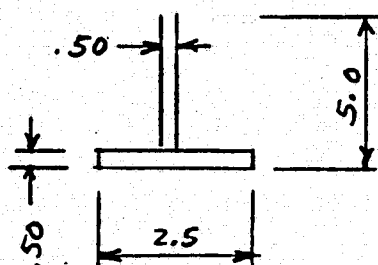
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Cylinder with insulation rings Model 3

Same as model 2 except ring as follows



Boundary Forces

$$F = \frac{(119 \times 120.2865)}{2}$$

$$F = 7157.0516 \text{ N}$$

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SUBJECT NTF PRESSURE SHELL
INS. RINGS

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T-SECTION
PROPERTIES.

=====

B= 2.500*
D= 5.000*
S= 0.250*
T= 0.250*
H= 4.875
AREA= 1.844
V= 3.267
I X-X= 4.797
I Y-Y= 0.332

T-SECTION
PROPERTIES.

=====

B= 2.500*
D= 5.000*
S= 0.500*
T= 0.500*
H= 4.750
AREA= 3.625
V= 3.267
I X-X= 3.360
I Y-Y= 0.698

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SUBJECT NTF PRESSURE SHELL
Effects of Insulation Rings
on Shell Stresses

SHEET NO. 5 OF.....
JOB NO.
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Results: (From SALORS)

Cylinder with Insulation Rings # STUDY #

CASE 1 NO RINGS

(76/03/01. 14.00.05)

Hoop Stress = 25,003 psi

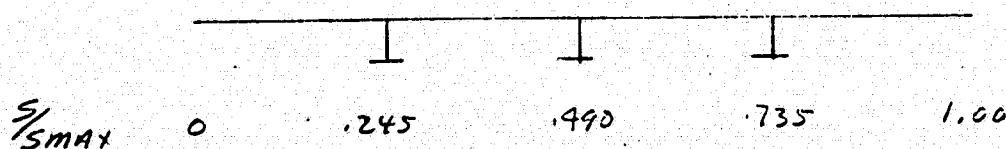
Long. Stress = 12,490 psi

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rad d.s.p. = .0882 in.

Case 2 With Insulation Rings $t = .25$

(76/03/01. 14.21.26)



S/s_{max}		Long. Stress (psi)	Hoop Stress (psi)	Rad. Defl. (in)	Net Section Hoop
.490	inside	20,329.8	23,605.8	.0701	20647
	outside	4,601.9	18,287.5		
.611	I	12,360.6	25,261.1	.0894	25295
	O	12,588.1	25,329.3		

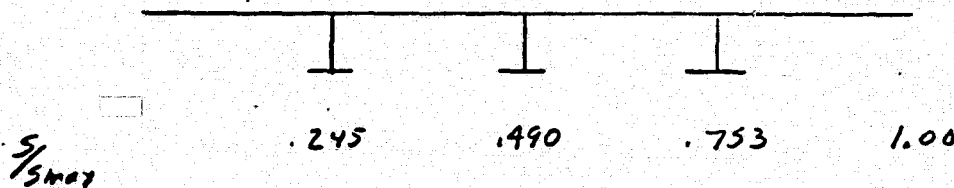
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SUBJECT NTF PRESSURE SHELL
Effects of Insulation Rings
on Shell Stresses

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Case 3 With Insulation Rings $t = 0.50$

76/03/01 14.13.51



s/s_{max}		Long Stress (psi)	Hoop Stress (psi)	Rad. defl. (in)	Net Section Hoop
.366	INSIDE	12 277	25 426		25 426
	OUTSIDE	12 650	25 538		
.490	INSIDE	25,353	21 725		17 314
	OUTSIDE	- 4535	13 983		
.611	INSIDE	12 277	25 426		25 426
		12 650	25 538		

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SUBJECT NTF PRESSURE SHELL
Effects of Insulation Rings
on Shell Stresses

SHEET NO. 7 OF.....
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For 0.25 in thick Rings

$$\frac{25295 - 25003}{25295} = .011$$

∴ Insulation rings results in 1.1% net section stress
in Hoop direction

For 0.50 in thick Rings

$$\frac{25482 - 25003}{25003} = 0.019$$

∴ Insulation results in 1.9% net section
stress in Hoop direction

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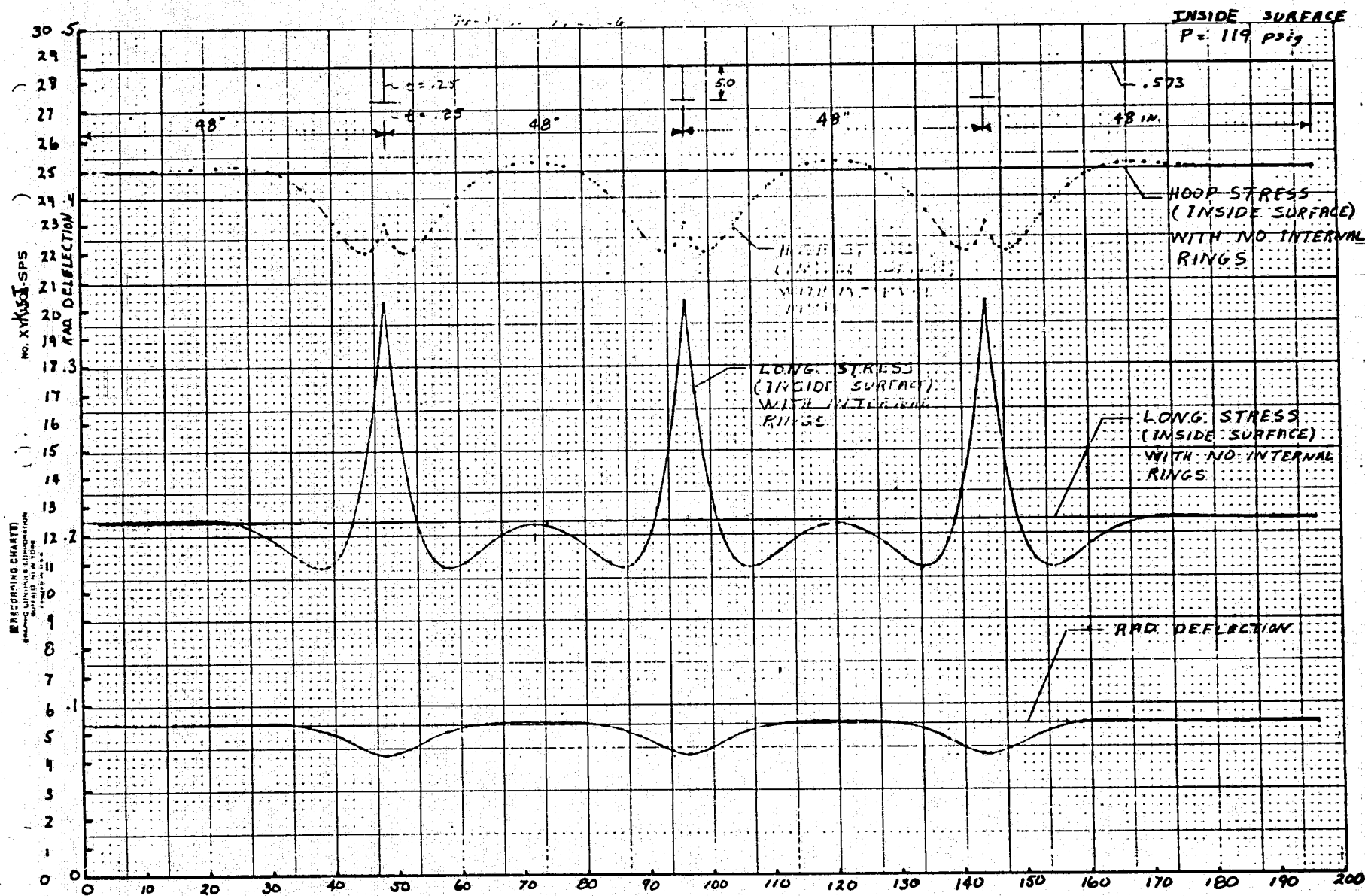
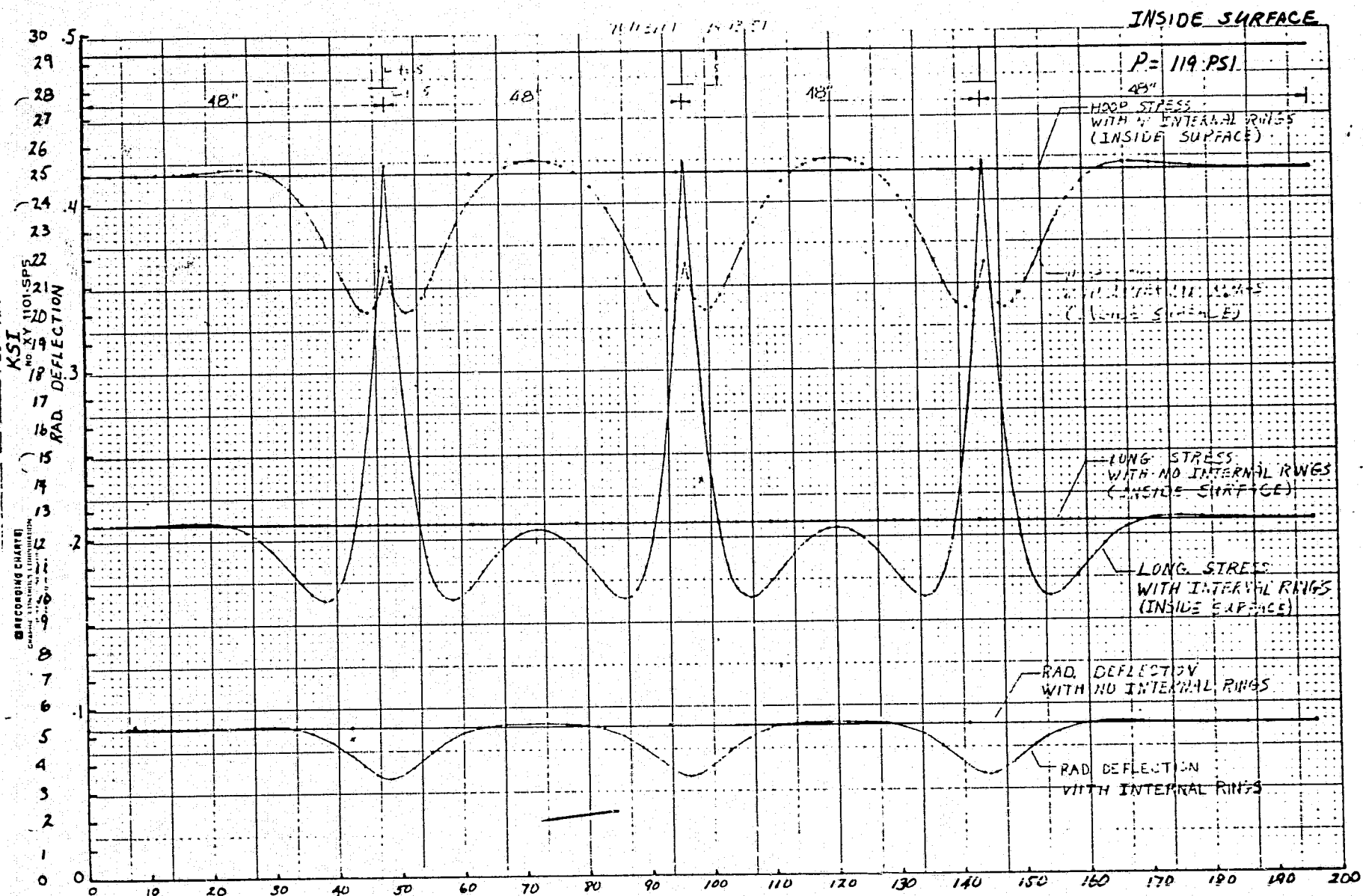


Fig 2

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